

# Development of a New Dual-Cylinder Rotary Compressor for VI System

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From GMCC

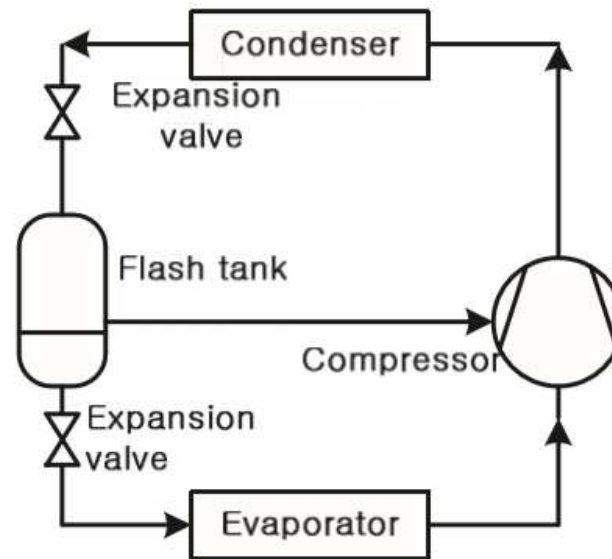
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# INTRODUCTION



- Vapor injection compression cycle's superiority over non-injection cycle has been well known
- VI system produces the high heating/cooling capacity, and its power consumption is less than the non-injection system.

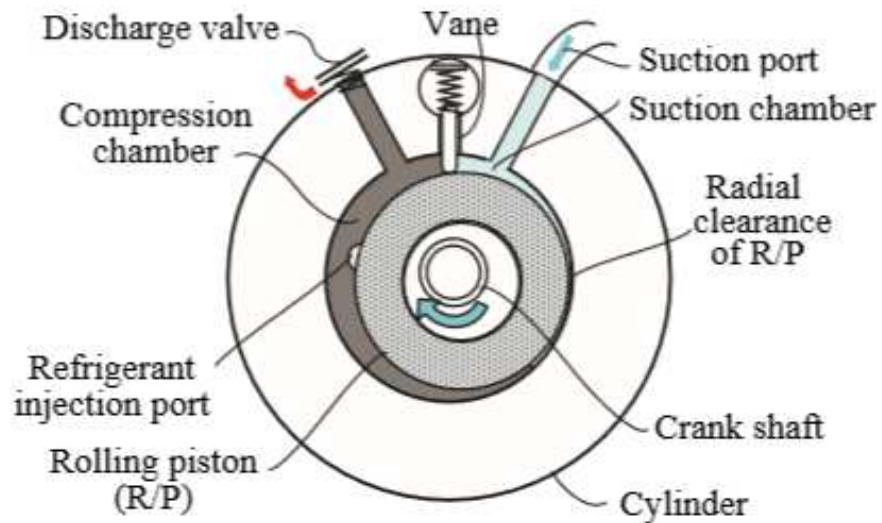




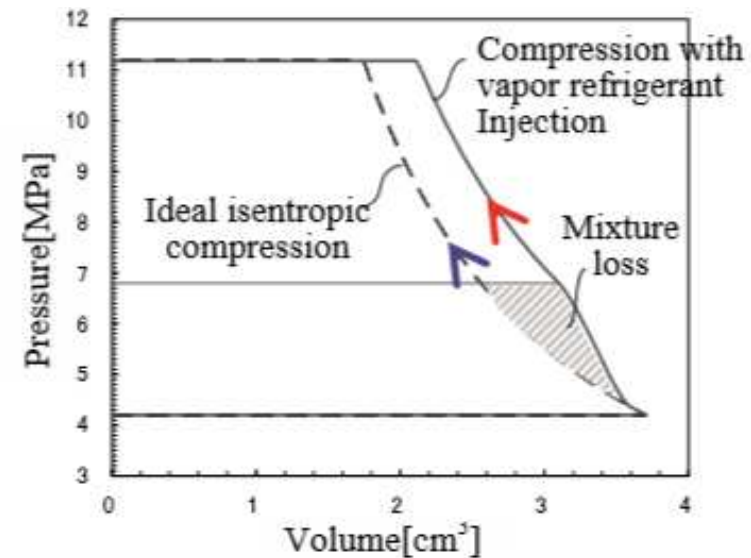
# INTRODUCTION



- Unfortunately, other problems arise, such as mixture loss on refrigerant injection into the compression chamber (Sekiya *et al.*, 2005, Tetsuhide *et al.*, 2008)



Structure of rotary compression mechanism in single type cylinder with refrigerant injection



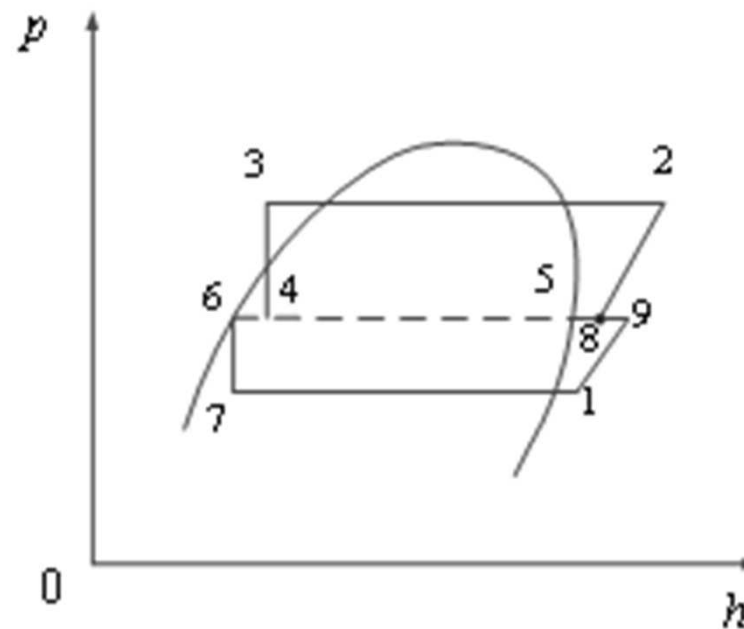
P-V diagram in cylinder of single rotary compressor with vapor refrigerant injection



# INTRODUCTION



- Another solution for the two-stage compression cycle is the use of a two-stage compressor, because most of the gas is compressed two times, the indicated power increases because of its two times exhaust process.

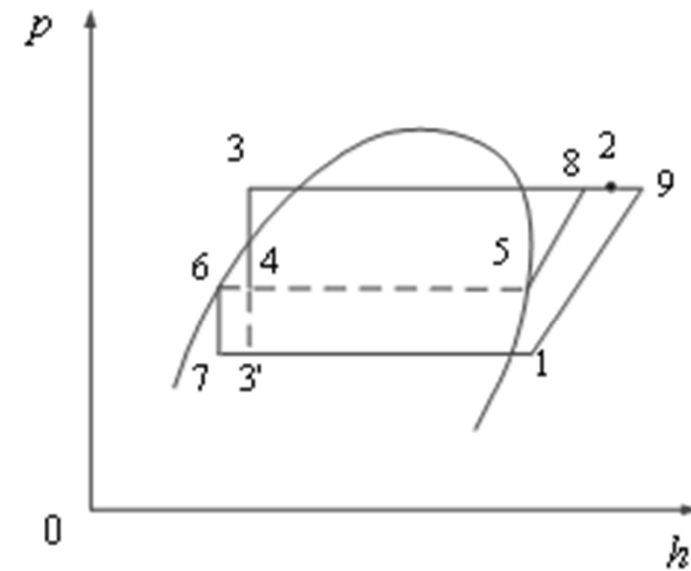
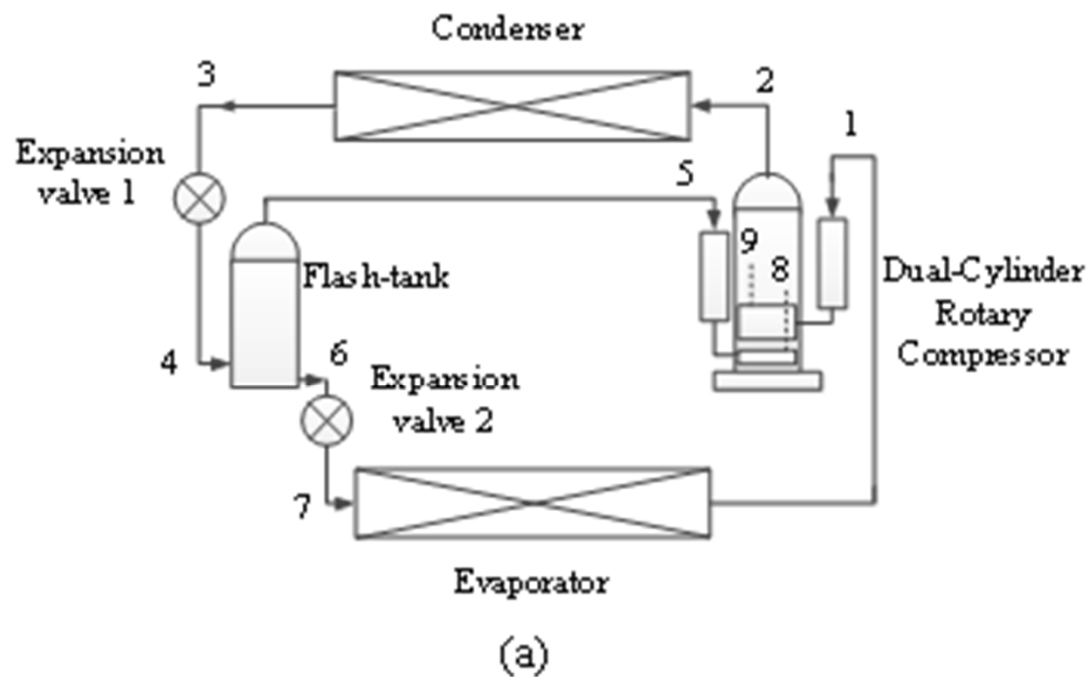




# MODEL OF THE NEW VI SYSTEM



- The gas from the flash-tank will be compressed independently



(b)



# THERMODYNAMIC ANALYSIS



- The refrigerating capacity in evaporator is

$$q_o = h_1 - h_7$$

- The cooling COP of the new VI system is

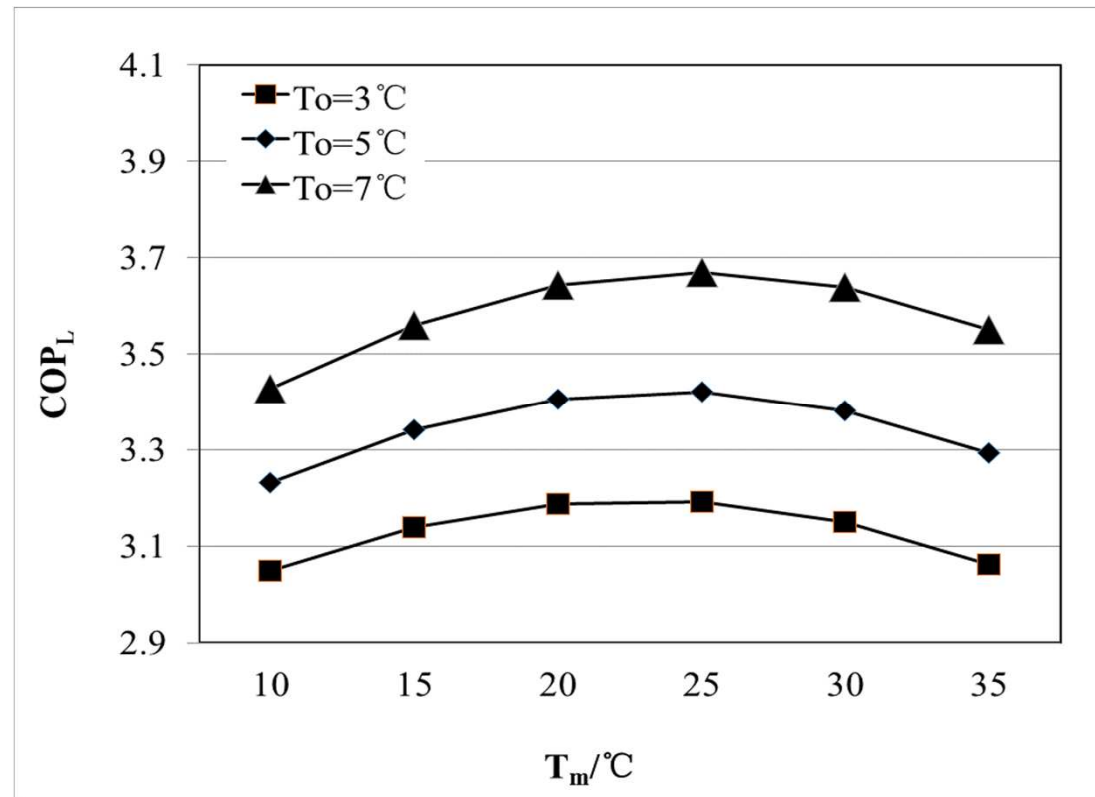
$$\text{COP}_L = \frac{q_o}{w_{1-9} + w_{5-8}}$$



# THERMODYNAMIC ANALYSIS



- The variation of  $COP_L$  with  $T_m$  (temperature of flash tank)



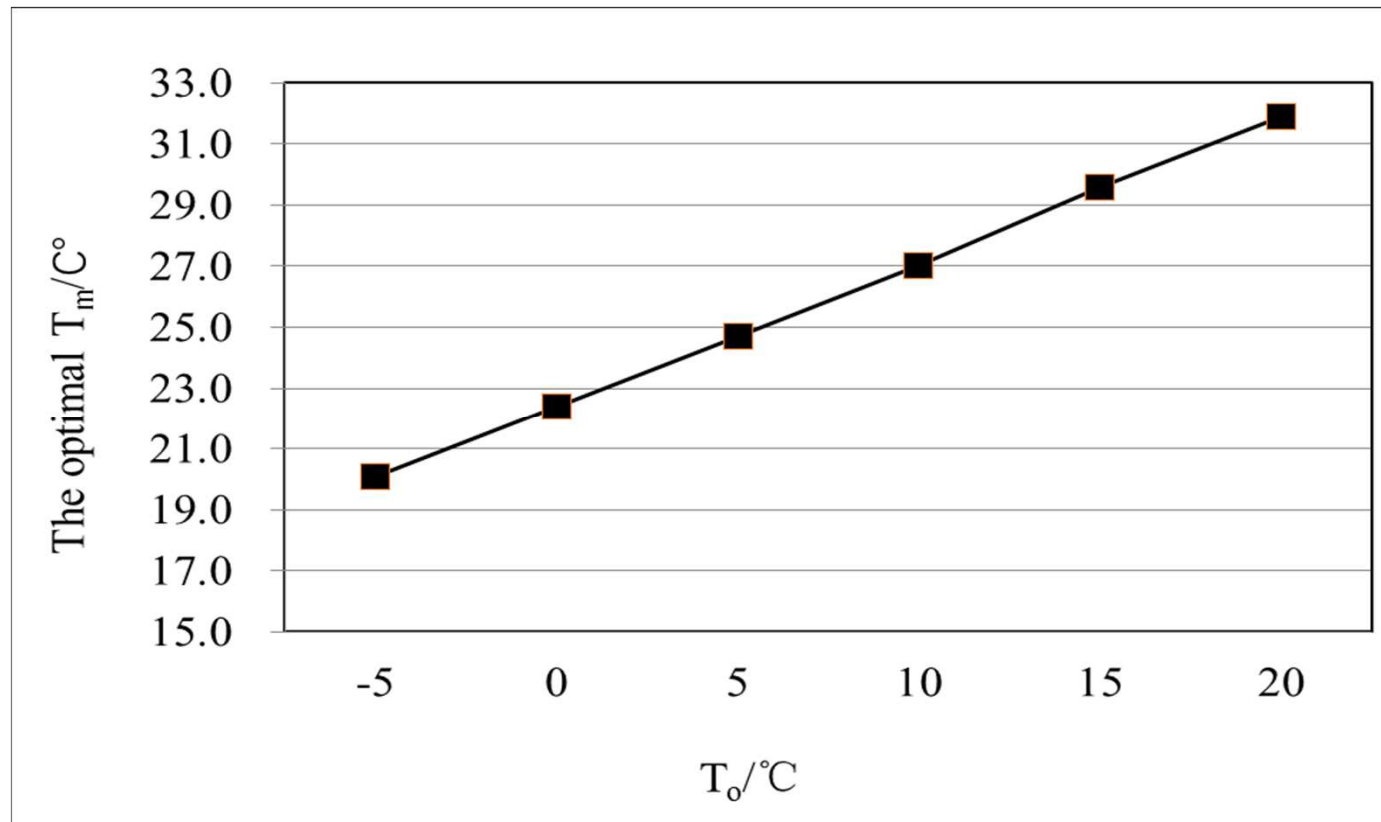
(condensing temperature,  $T_k$ , of 45°C, evaporating temperature,  $T_o$ , of 3°C, 5°C, 7°C under cooling condition)



# THERMODYNAMIC ANALYSIS



- The variation of the optimal  $T_m$  with  $T_o$



(condensing temperature,  $T_k$ , of  $45^\circ\text{C}$ , the superheat =  $5^\circ\text{C}$  and the sub cooling =  $5^\circ\text{C}$ )

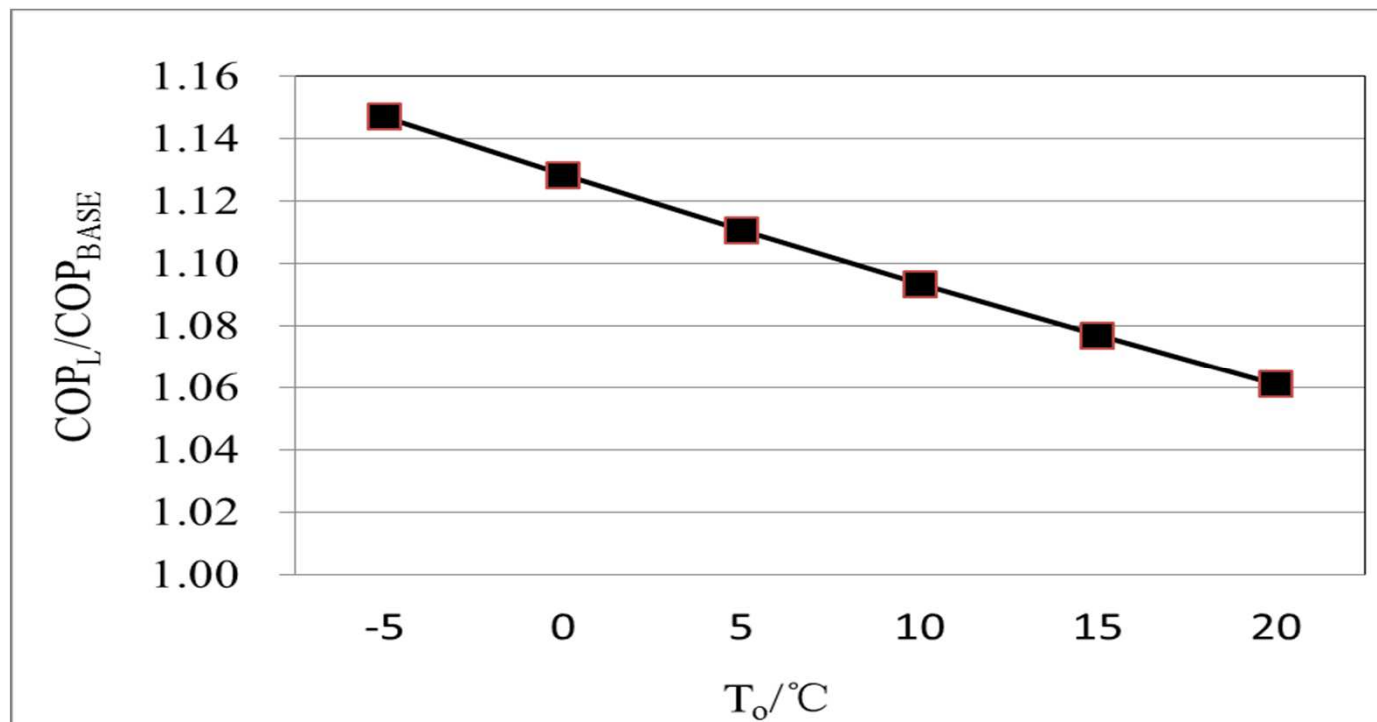




# THERMODYNAMIC ANALYSIS



- The variation of  $\text{COP}_L/\text{COP}_{\text{BASE}}$  with  $T_0$  (under the optimum intermediate temperature  $T_m$ )



(condensing temperature,  $T_k$ , of  $45^\circ\text{C}$ , the superheat =  $5^\circ\text{C}$  and the sub cooling =  $5^\circ\text{C}$ )



# EXPERIMENTAL RESULT



- Based on the above analysis, a new dual-cylinder compressor is designed and evaluated
- The compressor is designed based on 5 conditions

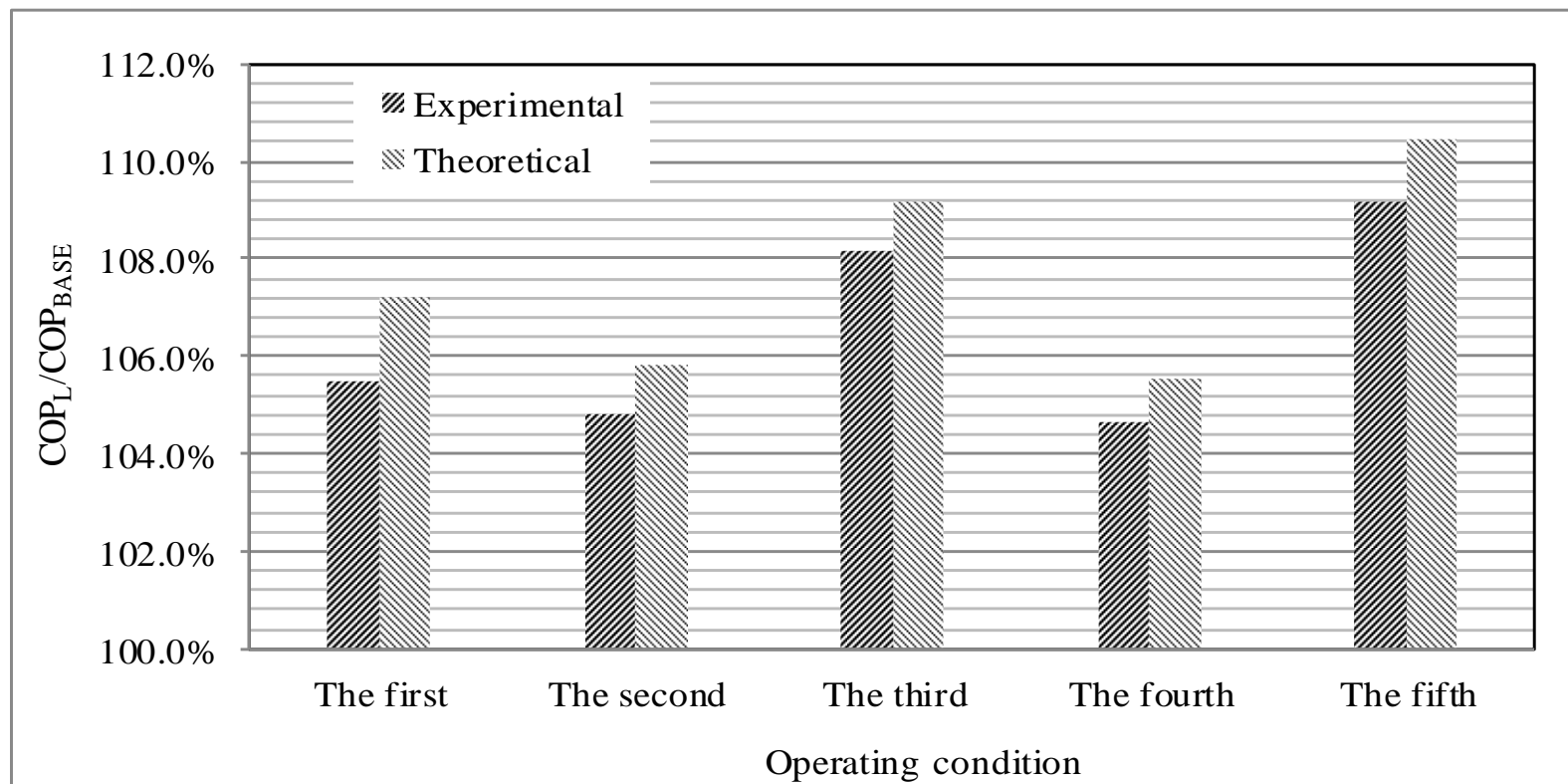
Operating condition	The first	The second	The third	The fourth	The fifth
Condensing temperature (°C)	45.6	39.6	43.6	31.5	46.5
Evaporating temperature (°C)	10.7	17.8	0.8	2.9	-4.7
Superheat (°C)	9.3	7.2	5.3	5.1	6.7
Sub cooling (°C)	6.1	1.0	7.0	4.0	9.9
Ambient temperature (°C)	35	35	35	35	35
Contribution ratio	25%	33%	13%	10%	19%



# EXPERIMENTAL RESULT



- The Comparison between theory and experiment of  $COP_L/COP_{BASE}$

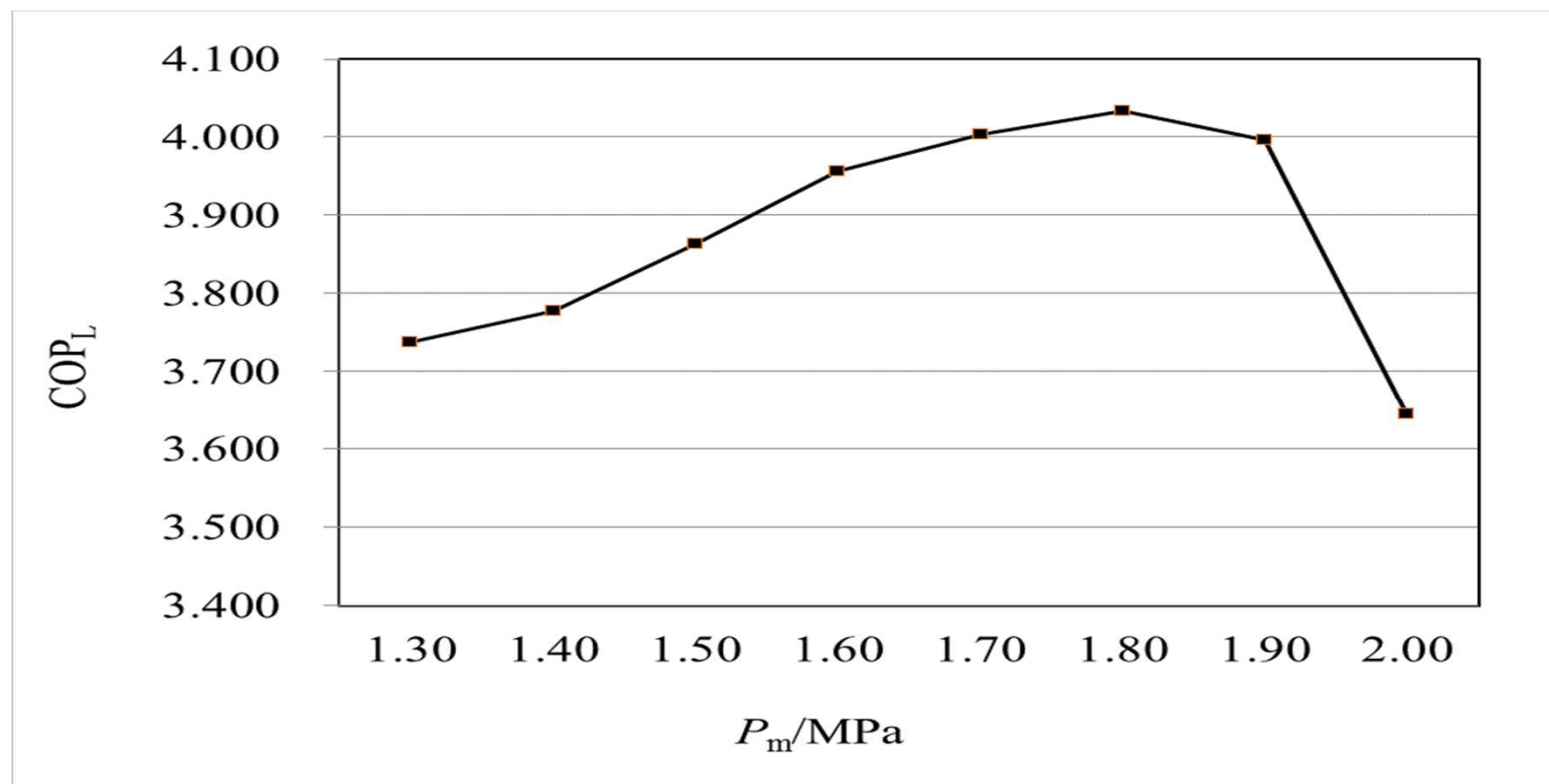




# EXPERIMENTAL RESULT



- The variation of  $COP_L$  with the intermediate pressure  $p_m$  about the first condition





# CONCLUSIONS



- A new dual-cylinder rotary compressor for VI systems is developed and evaluated
- The cooling COP increase 4.6%-9.1% under the test conditions.
- The variations of the  $COP_L$  with the intermediate pressure  $p_m$  (pressure of flash tank) indicates that when the intermediate pressure exceeds the optimum value, the  $COP_L$  is decreased dramatically because of the suction of liquid and this should be avoided in the application.



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**Thank you!**