

Investigation of Rotary Compressor Heat Dissipation Model

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OUTLINE



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COMPRESSOR MODEL

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UNCERTAINTY ANALYSIS

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HEAT DISSIPATION MODEL

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COMPARISON RESULT

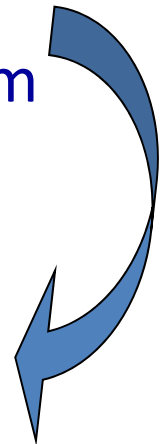
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CONCLUSION

Traditional method

Adiabatic model \longrightarrow Discharge temperature T_d
Test \longrightarrow measured temperature T_{dm}

$$T_d > T_{dm}$$



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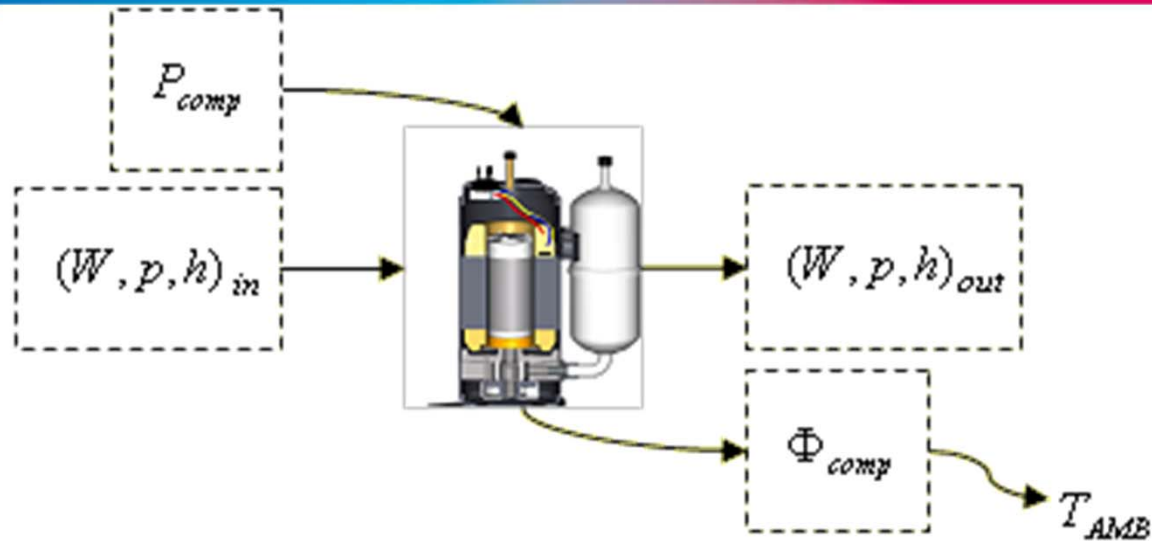
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COMPRESSOR MODEL



academic equation

Conservation of Mass:

$$m_{out} = m_{in} (= m_{comp})$$

Conservation of Energy:

$$m_{in} h_{in} + P_{comp} - m_{out} h_{out} - \Phi_{comp} = 0$$

$$h = f(p, T)$$

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UNCERTAINTY ANALYSIS



1HP constant speed rotary compressor

Condition: $T_e=7.2^{\circ}\text{C}$, $T_c=54.4^{\circ}\text{C}$, $T_s=18.3^{\circ}\text{C}$, $T_d=97.2^{\circ}\text{C}$

Mass Flow: $m=63.4\text{kg/h}$, Input Power: $P=936\text{w}$

	Variable \pm Uncertainty	% of uncertainty
heat dissipation/W	$\Phi=163.5 \pm 4.56$	
input power/W	$P=936 \pm 0.936$	4.21%
discharge pressure/KPa	$P_d=2147 \pm 5.368$	5.56%
suction pressure/KPa	$P_s=625 \pm 1.563$	1.29%
discharge temp./ $^{\circ}\text{C}$	$T_d=97.2 \pm 0.2$	48.49%
suction temp./ $^{\circ}\text{C}$	$T_s=18.3 \pm 0.2$	33.99%
mass flow kg/h	$m=63.4 \pm 0.095$	6.46%

There is 4.56W uncertainty of heat dissipation, about 2.8% of the whole dissipation. The discharge temperature makes the biggest contribution.

UNCERTAINTY ANALYSIS



1.5HP variable speed rotary compressor

Condition: $T_e=17.8^\circ\text{C}$, $T_c=40.6^\circ\text{C}$, $T_s=19.7^\circ\text{C}$, $T_d=53.5^\circ\text{C}$

Mass Flow: $m=56\text{kg/h}$, Input Power: $P=318.5\text{W}$

	Variable \pm Uncertainty	% of uncertainty
heat dissipation/W	$\Phi=28.52 \pm 6.66$	
input power/W	$P=318.55 \pm 0.3185$	0.23%
discharge pressure/KPa	$P_d=2445 \pm 6.113$	10.37%
suction pressure/KPa	$P_s=1352 \pm 3.38$	5.53%
discharge temp./ $^\circ\text{C}$	$T_d=53.5 \pm 0.2$	46.35%
suction temp./ $^\circ\text{C}$	$T_s=19.7 \pm 0.2$	37.10%
mass flow kg/h	$m=56 \pm 0.084$	0.43%

There is 6.66W uncertainty of heat dissipation, about 23% of the whole dissipation. The discharge temperature makes the biggest contribution.

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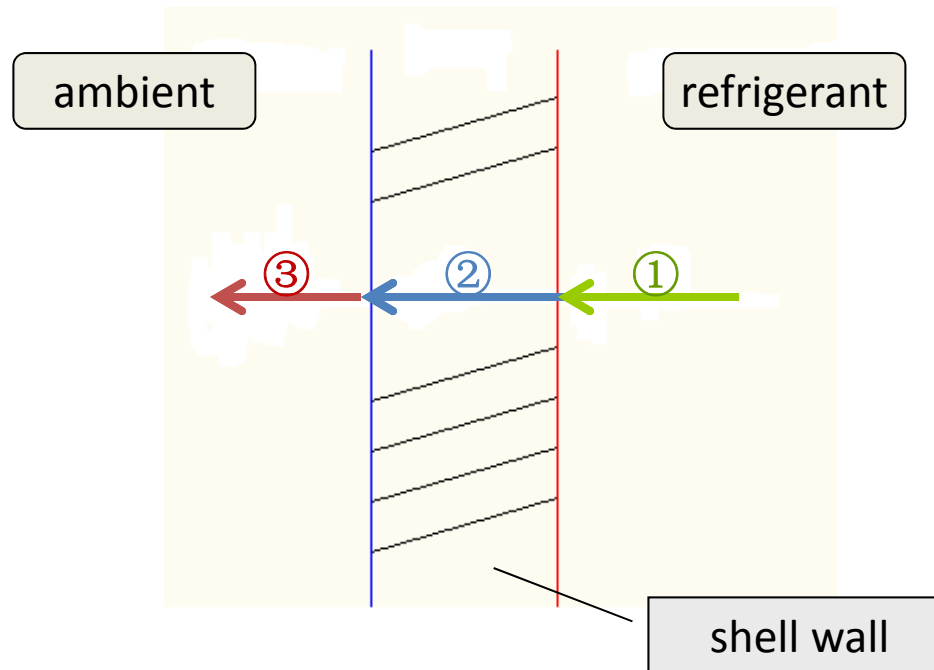
CONCLUSION

HEAT DISSIPATION MODEL



- ① heat transfer from the refrigerant to shell wall inside;
- ② heat transfer from shell wall inside to shell wall outside ;
- ③ heat transfer from shell wall outside to ambient air.

Heat flow Φ of each stage should be the same due to the steady state process.



HEAT DISSIPATION MODEL



The heat transfer coefficient k can be expressed:

$$\frac{1}{k} = \frac{1}{h_{ref}} + \frac{\delta}{\lambda} + \frac{1}{h_{amb}}$$

In the heat dispersion model, the air-side heat transfer includes convective heat transfer and heat radiation:

$$\Phi = \Phi_c + \Phi_r$$

Convective heat transfer



Convective heat transfer can be described as following :

$$\Phi_c = h \cdot A \cdot (T_d - T_{amb})$$

Heat transfer coefficient can be calculated as following :

$$h = \text{Nu} \times \frac{\lambda}{d}$$

Forced convective heat transfer



The forced convection is described :

$$Nu = C Re^n Pr^{1/3}$$

Pr is the Prandtl number, Renault number is listed :

$$Re = \frac{ud}{\nu}$$

Re is about 10,000 \longrightarrow C=0.193,
n=0.618

Natural convective heat transfer



The empirical correlation of natural convection is shown as following:

$$Nu_m = C(GrPr)_m^n$$

Here, Gr is the Grashof number .C=0.59, n=0.25.

Heat radiation



Environment is assumed as the black body, and the compressor is a gray body
emissivity ε and absorptivity α ratio of paint is 0.95.
The actual heat radiation is shown:

$$\Phi_r = A(E_1 - E_2)$$

E_1 is heat radiation from compressor to environment :

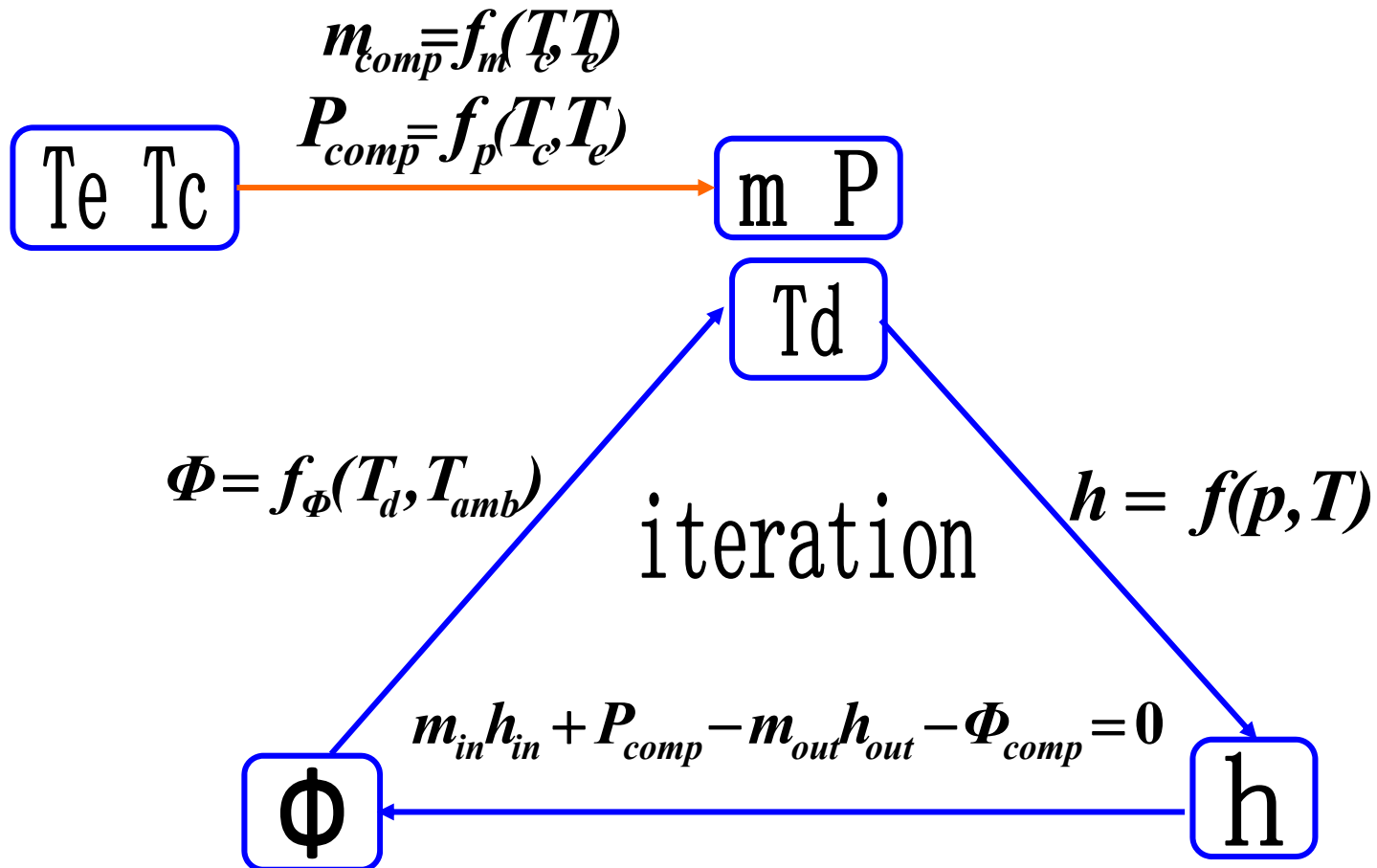
$$E_1 = \varepsilon \cdot \sigma T_d^4$$

E_2 is heat radiation from environment to compressor:

$$E_2 = \alpha \cdot \sigma T_{amb}^4$$

$$\sigma = 5.67 \times 10^{-8}$$

APPLICATION



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1.5HP constant speed rotary compressor data in forced-convection

Te(°C)	Tc(°C)	Measured discharge temp (°C)	Model discharge temp (°C)	Adiabatic discharge temp (°C)	Radiation (W)	Forced-convection heat transfer(W)	Model heat dissipation(W)	Measured heat dissipation(W)
10.4	65	113						201
7.2	54.4	97.						163
-0.4	40	83.						110
-4	30	73.						84
1.5	40	81.						106
7.2	45	83.						112
10	45	81.						106
1.5	30	68.						70
10	40	74.						87
10	30	60.8	60.8	64.2	20	38	58	58

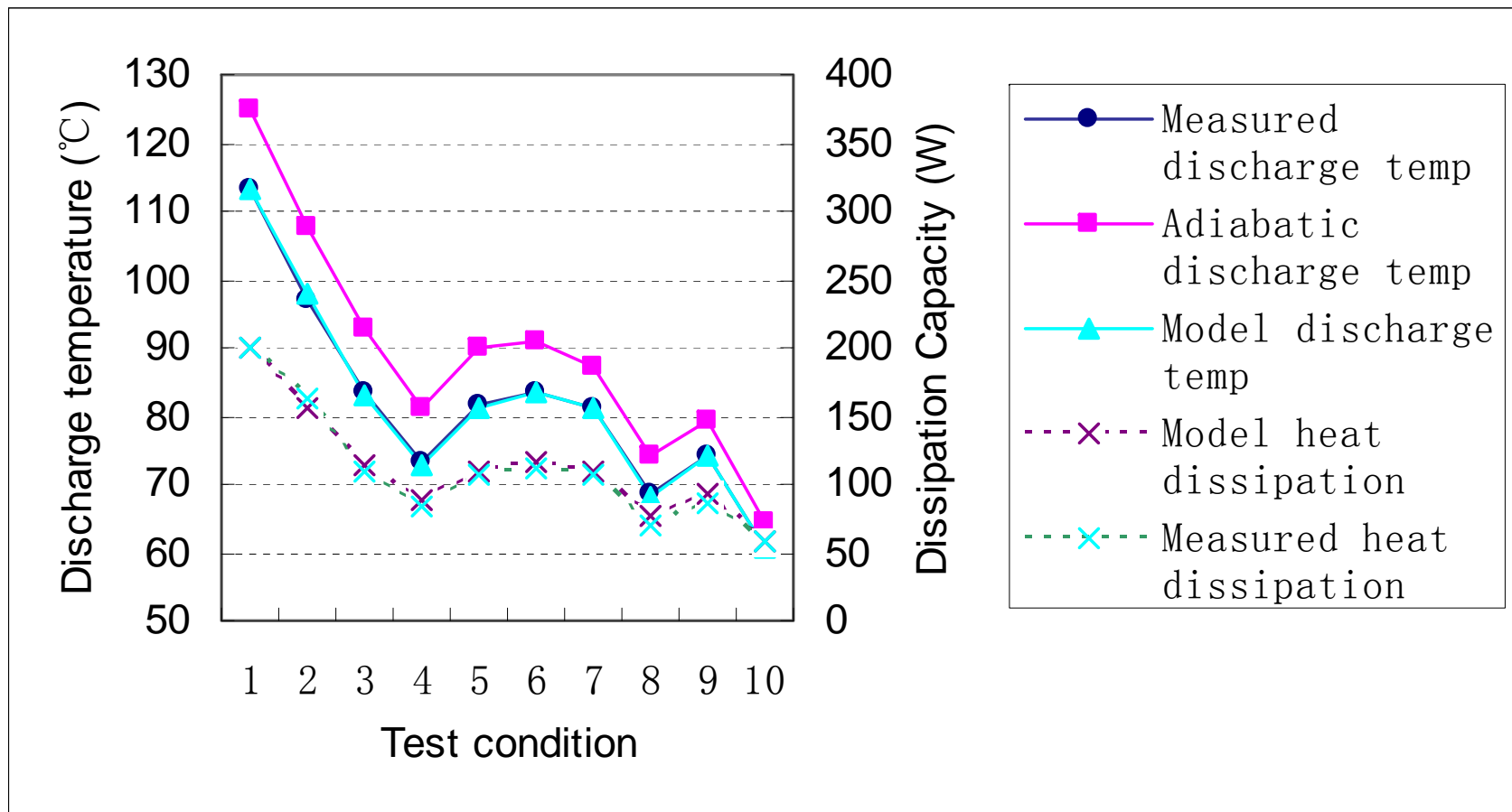
63% 37%

■ Radiation
■ Convecting

COMPARISON RESULT



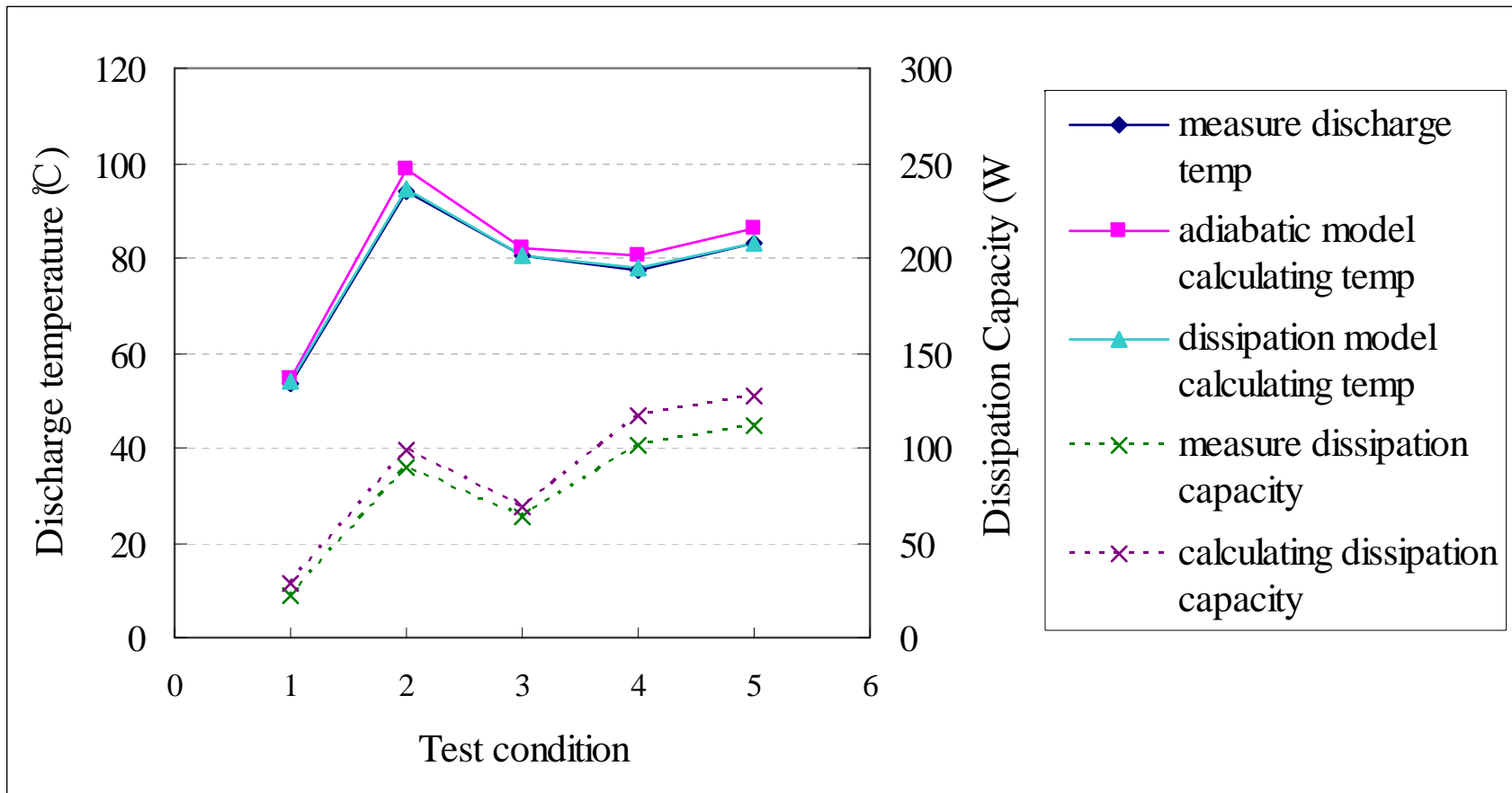
- Comparison in discharge temperature and heat dissipation of 1.5HP constant speed compressor in forced-convection.



COMPARISON RESULT



- Comparison in discharge temperature and heat dissipation of 1.5HP variable speed compressor in forced-convection

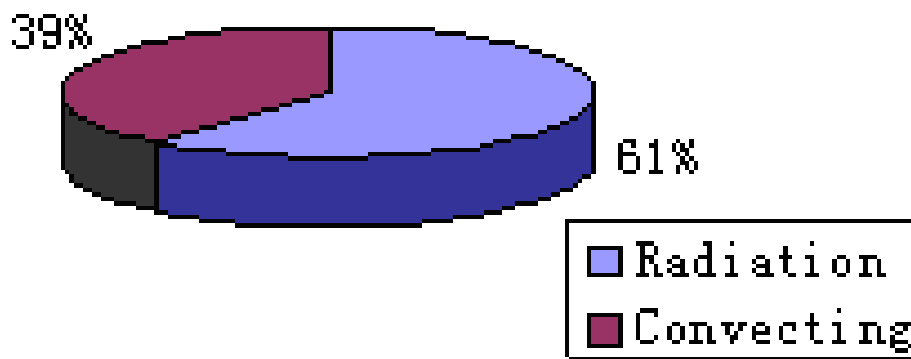


COMPARISON RESULT



1.5HP constant speed rotary compressor data in natural-convection

Te(°C)	Tc(°C)	Measured discharge temp (°C)	Model discharge temp (°C)	Adiabatic discharge temp (°C)	Radiation(W)	Natural-convection heat transfer(W)	Model heat dissipation(W)
10.4	65	113.1	116.7	124.9	83	56	139
7.2	54.4					42	105
-0.4	40					30	76
-4	30					23	57
1.5	40					29	72
7.2	45					30	75
10	45					28	70
1.5	30	68.4	70.2	74	29	19	48
10	40	74.4	76.1	79.5	35	23	58
10	30	60.8	62.2	64.2	21	13	35



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CONCLUSION



- Heat dissipation model
 - forced-convection
 - natural-convection
 - heat radiation
- The discharge temperature model deflection is less than 4 °C, the average heat dissipating error is below 15%,
- Calculate an accurate value for discharge temperature



THE END
THANKS !