

Conversion Factors for Comparing the Performance of Variable Refrigerant Flow Systems

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Background

Energy consumption of the building sector exceeds 30%
of the total consumption



CO2 emissions reached 32 billion tons in 2013



The ratio of air conditioning energy consumption in
office buildings approached 40%



Reduction potential is quite considerable



Background



In Japan

Performance evaluation of
VRF (Variable Refrigerant Flow) systems



Rated by JIS (Japan Industrial Standards) Committee



In 2006, the Committee revised
the air-conditioning performance evaluation





Background



Factors

COP = Coefficient of Performance

$$= \frac{\text{Rated capacity (kW)}}{\text{Rated power consumption (kW)}}$$

➔ No consideration of seasonal temperature changes

Far from the actual operation

APF = Annual Performance Factor

$$= \frac{\text{Annual total capacity of cooling / heating periods (kWh)}}{\text{Annual total consumption of cooling / heating periods (kWh)}}$$

➔ Proper evaluations

Close to the actual operation



Background

In the market

COP machines



COP = 2.6
COP = 3.3
:

APF machines



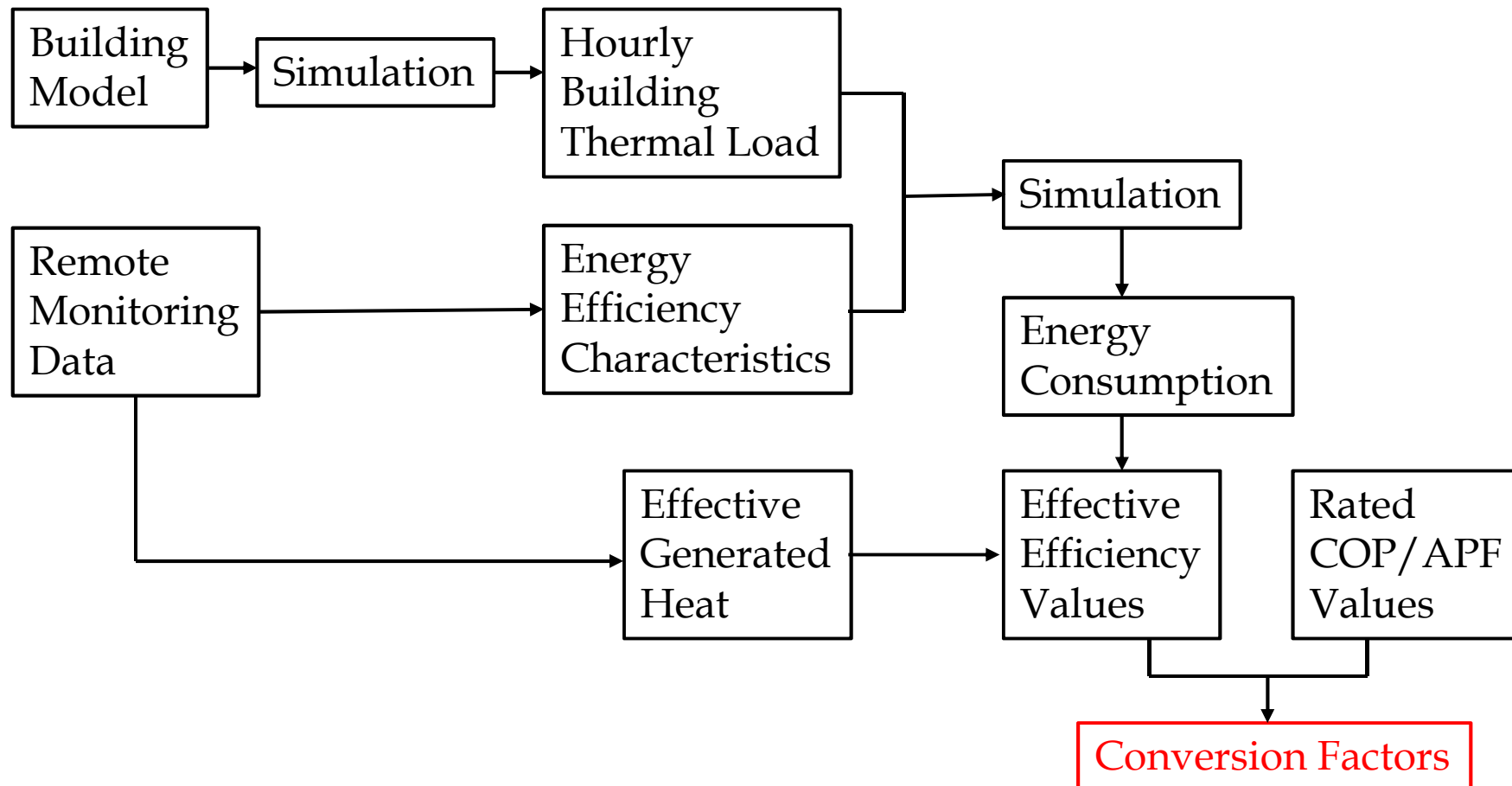
APF = 4.0
APF = 5.5
:

Difficult to compare

Conversion factors



Study flow

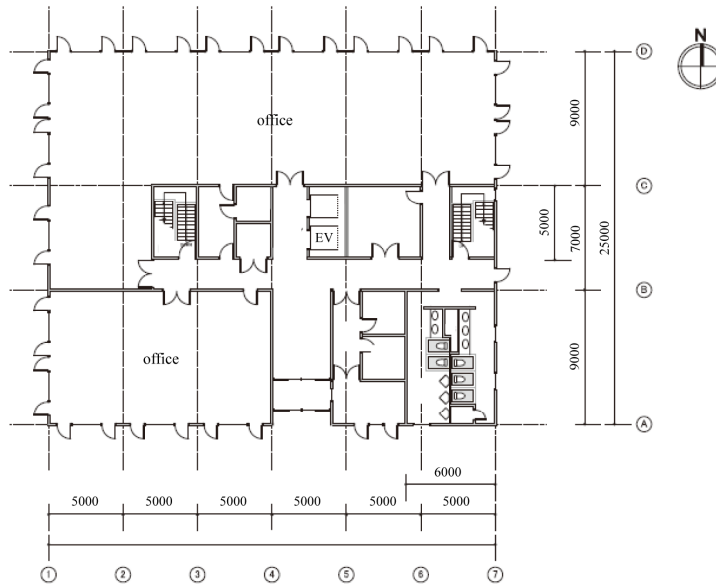




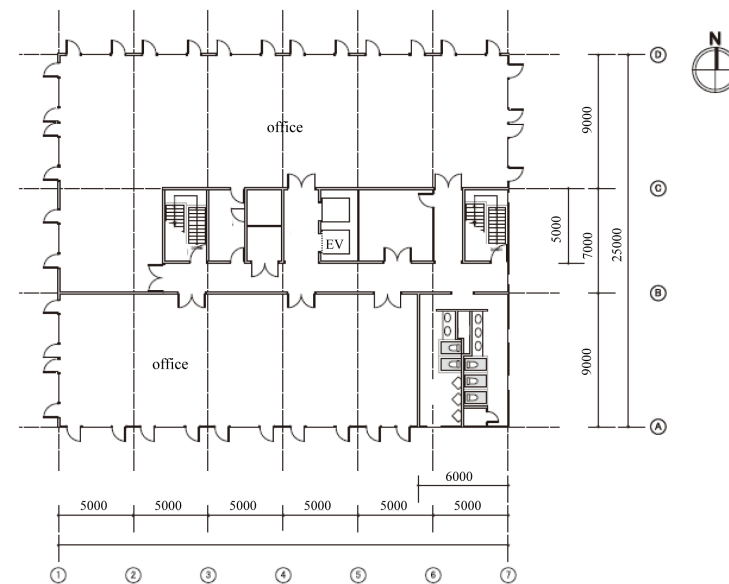
Simulation model



Office building - 5000 m²



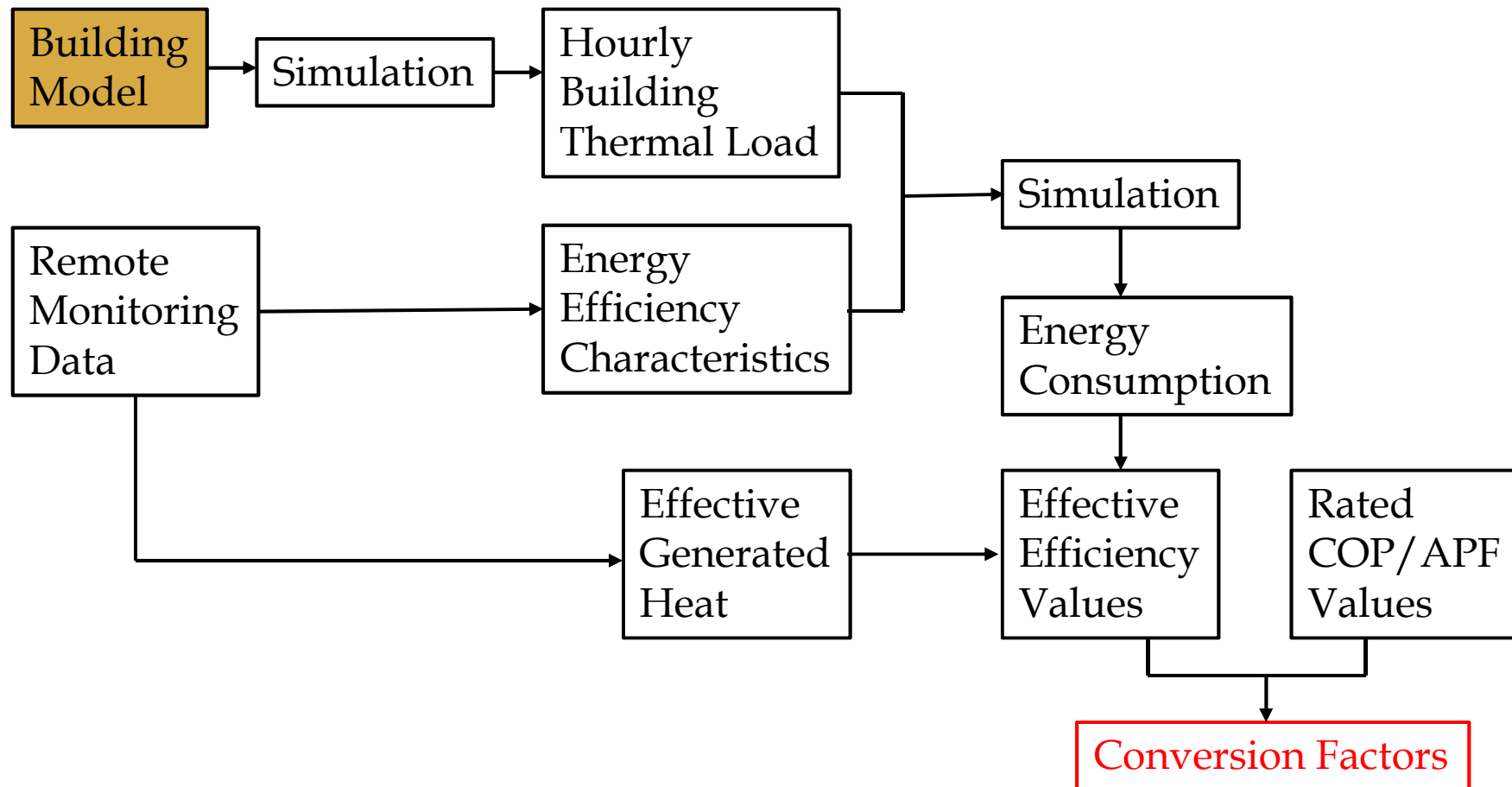
Floor plan (1st)



Floor plan (2nd-6th)

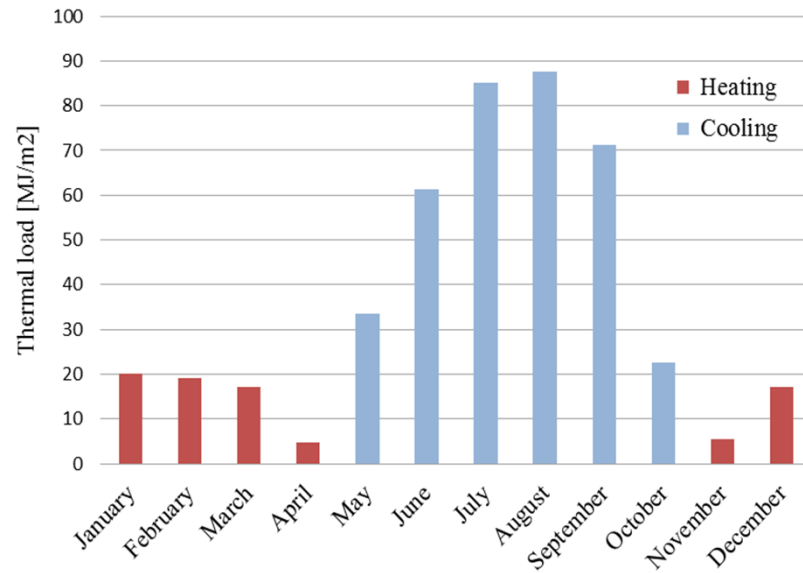


Study flow

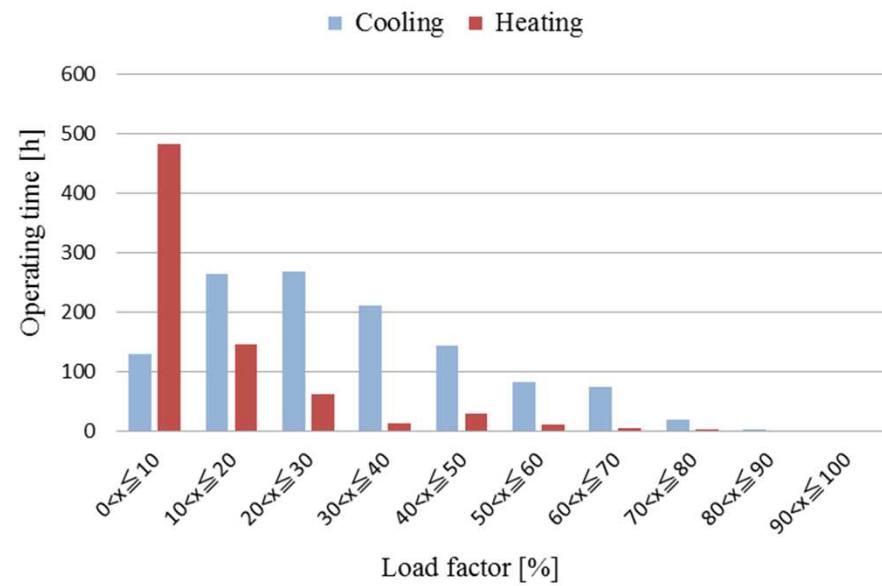




Simulation results



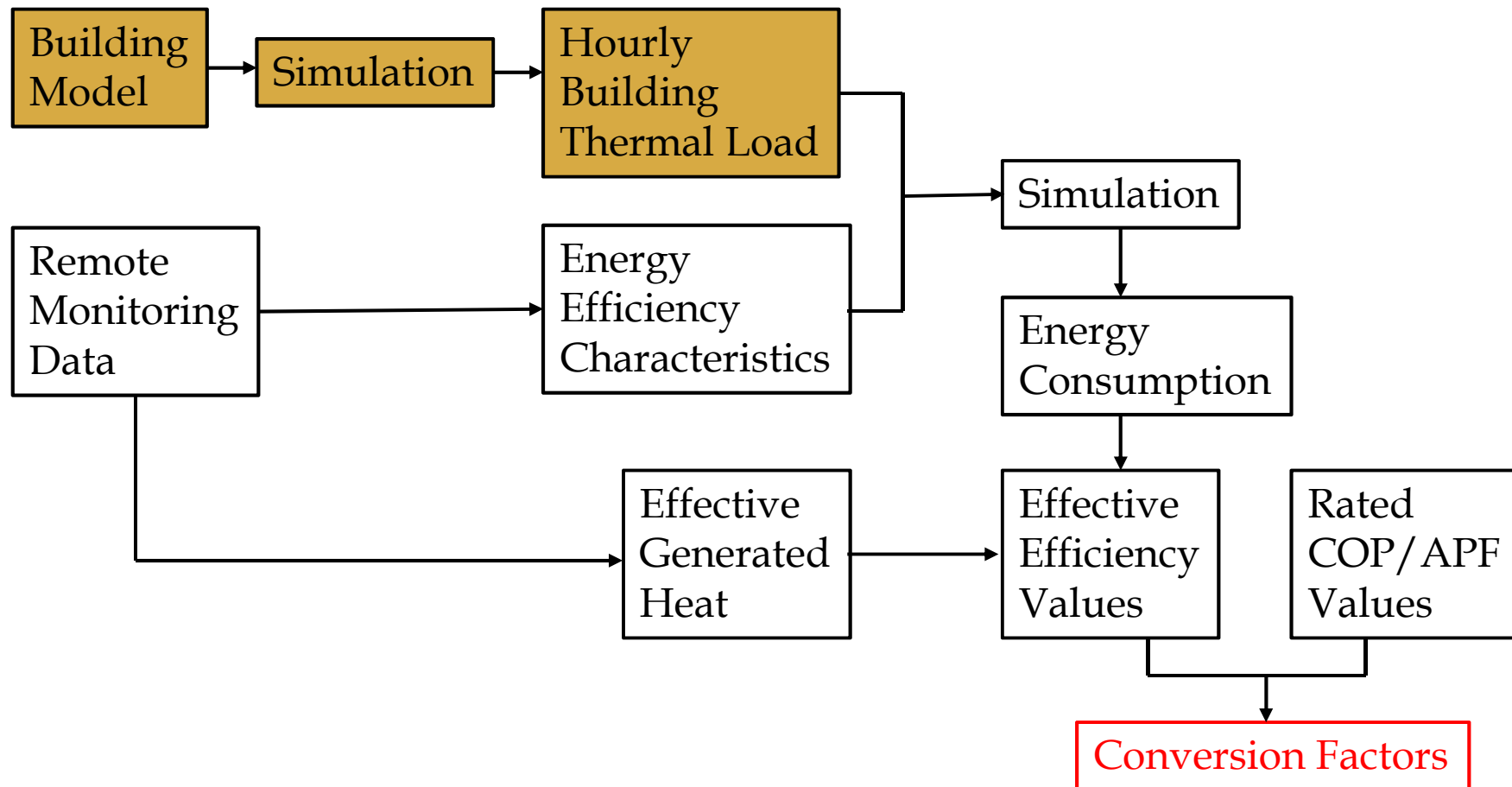
Monthly thermal load



Load factor



Study flow





Remote monitoring data

- Total energy consumption data
- Outdoor unit energy consumption data
- Indoor unit effective generated heat
- Refrigerant mass flow of a compressor per unit time
- Indoor suction temperature
- Preset temperature
- Refrigerant physical properties
- On-off time of defrost
- On-off time of thermostat
- Standby power



Obtained by
the Compressor Curve method



Compressor Curve method

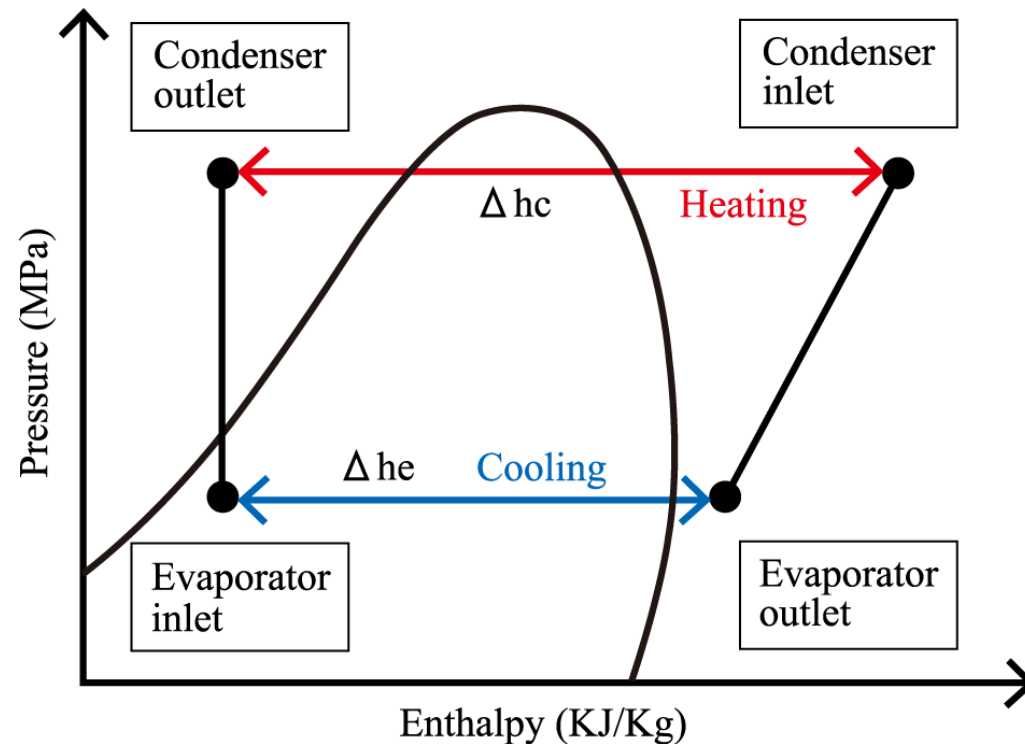


$$Q = G_{\text{comp}} \times \Delta h$$

Q : Capacity (kW)

G_{comp} :
Refrigerant mass flow
(kg/min)

Δh :
Specific enthalpy
difference
(kJ/kg)





Compressor Curve method



$$G_{\text{comp}} = \rho \times V \times N \times \eta$$

ρ : Suction refrigerant density (kg/m³)

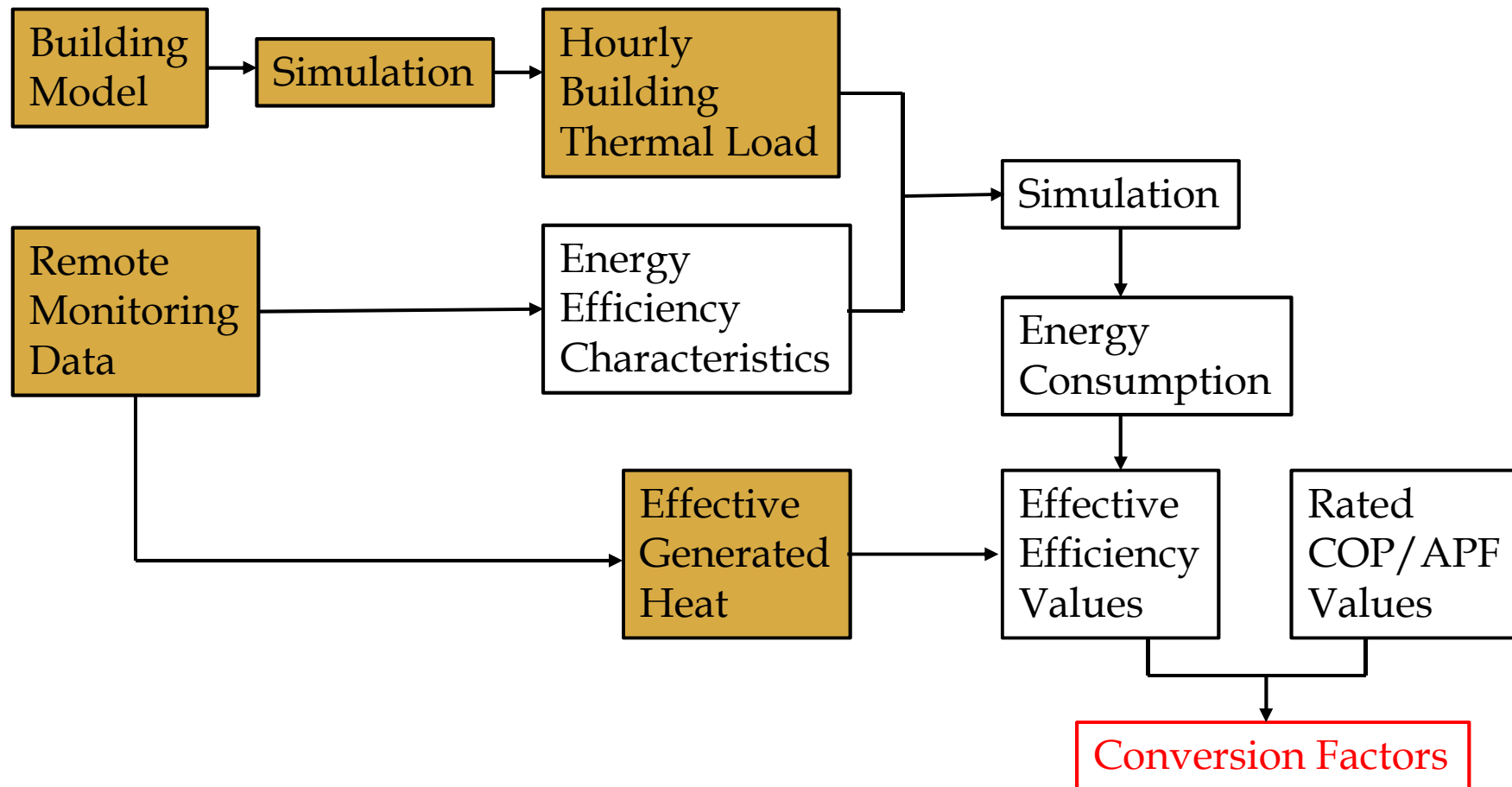
V : Compressor displacement (m³)

N : Compressor revolution (rpm)

η : Volumetric efficiency (-)



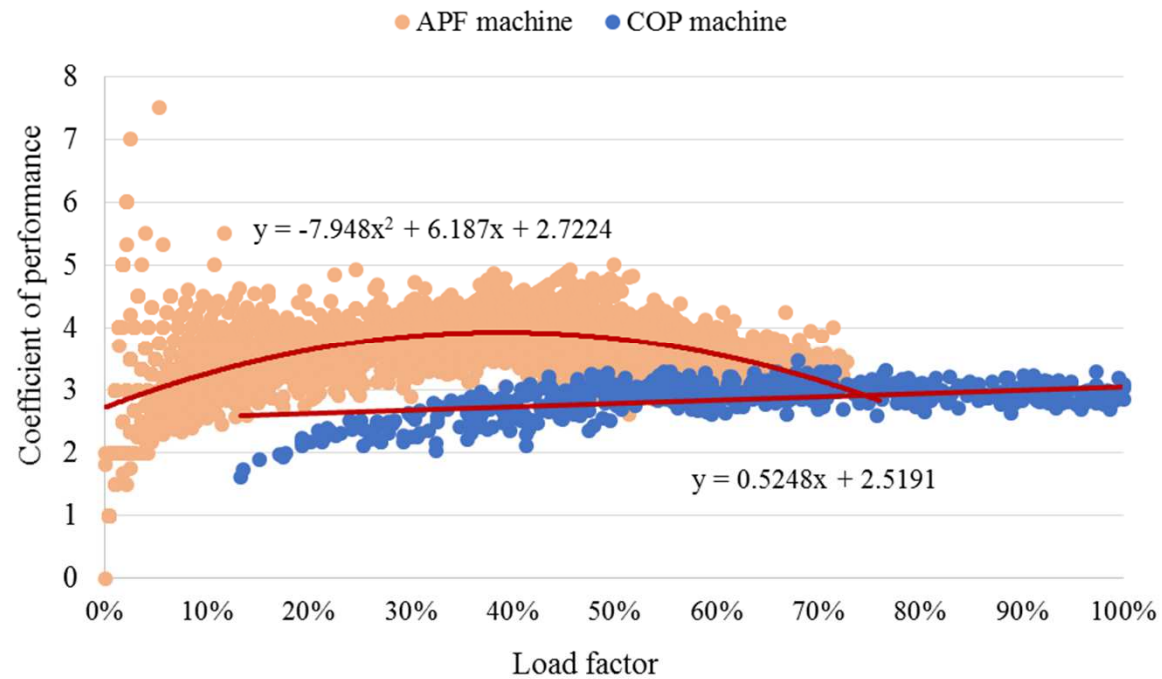
Study flow





Comparison Results

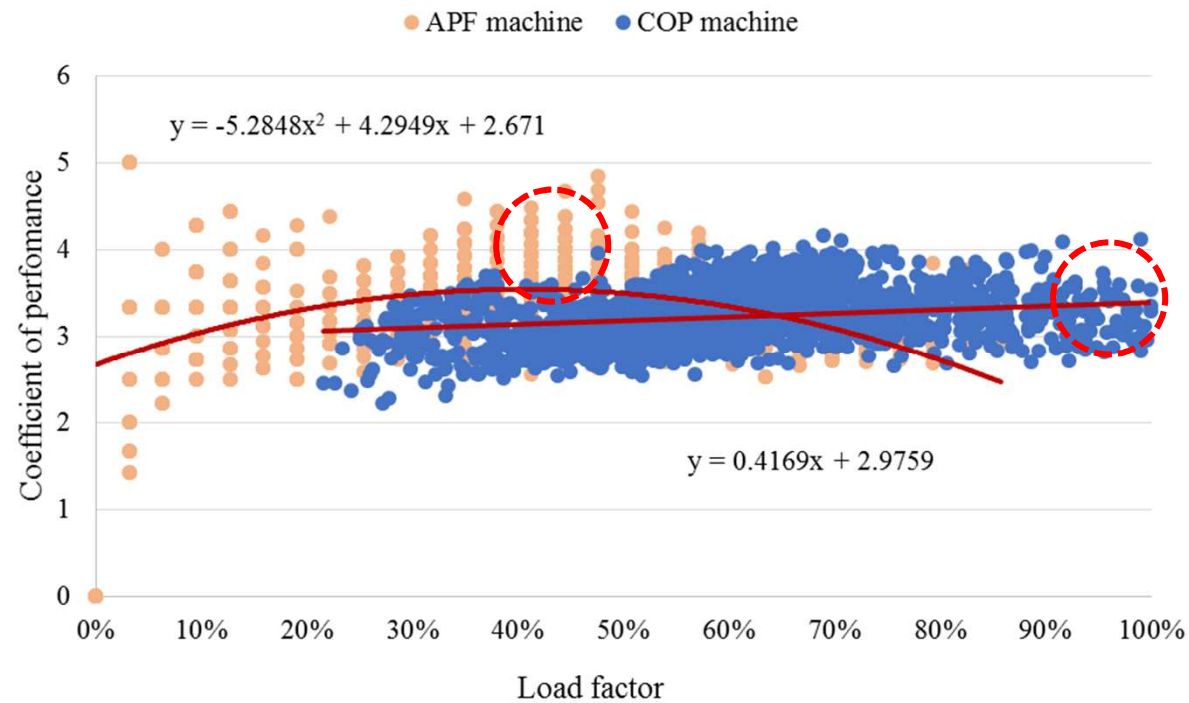
Comparison of the COP and APF machines in the cooling operation





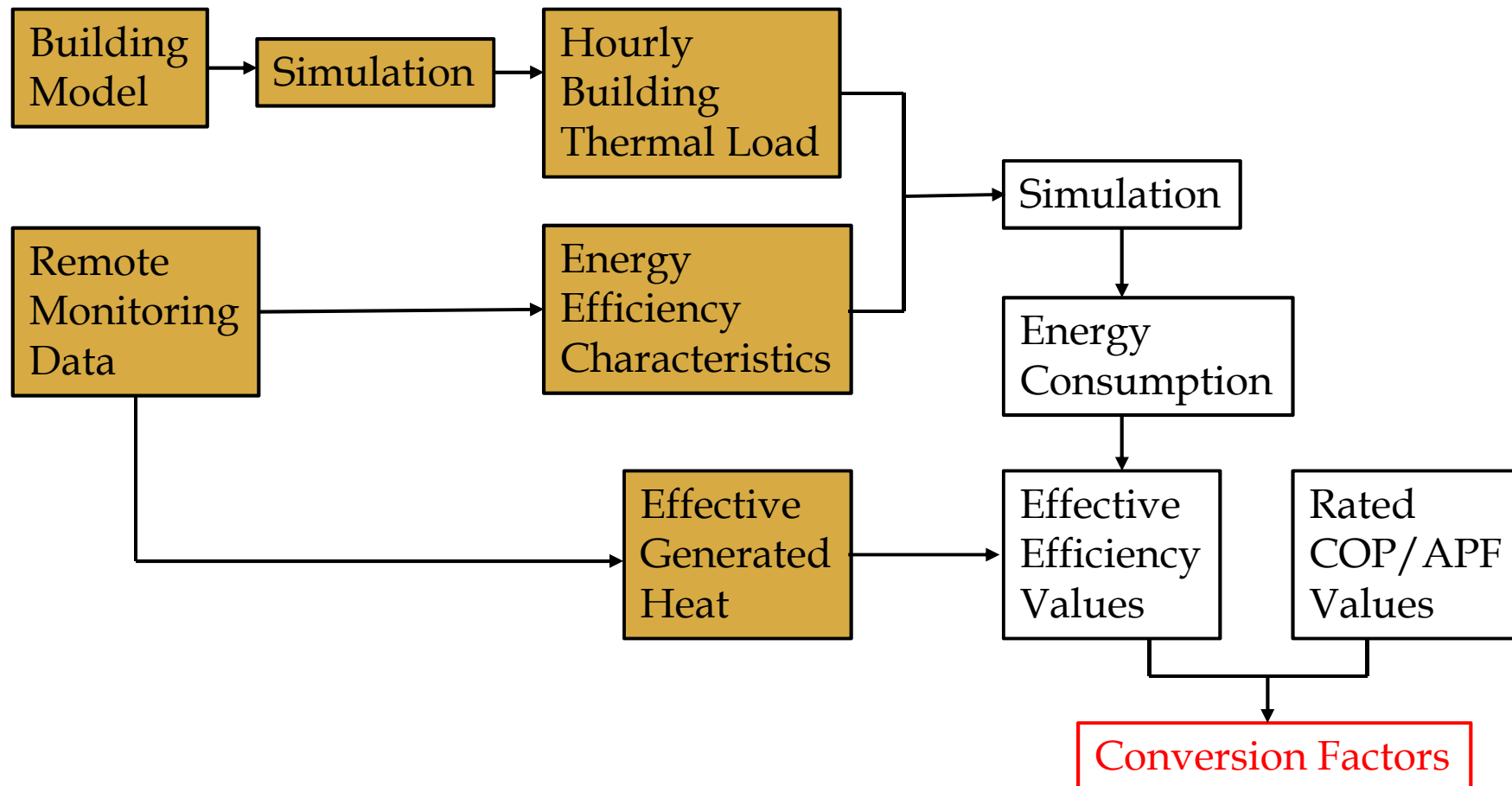
Comparison Results

Comparison of the COP and APF machines in the heating operation



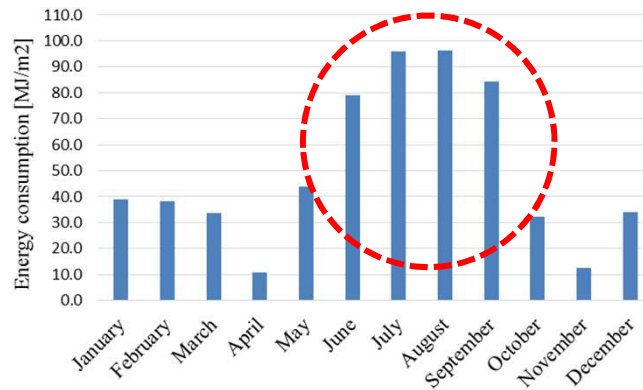


Study flow

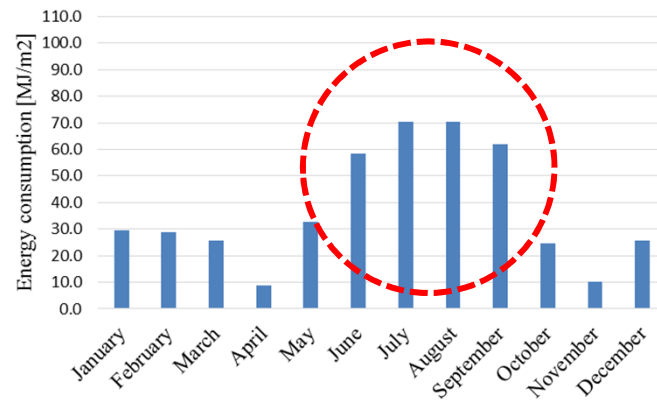




Simulation Results



Energy consumption of the **COP** machine



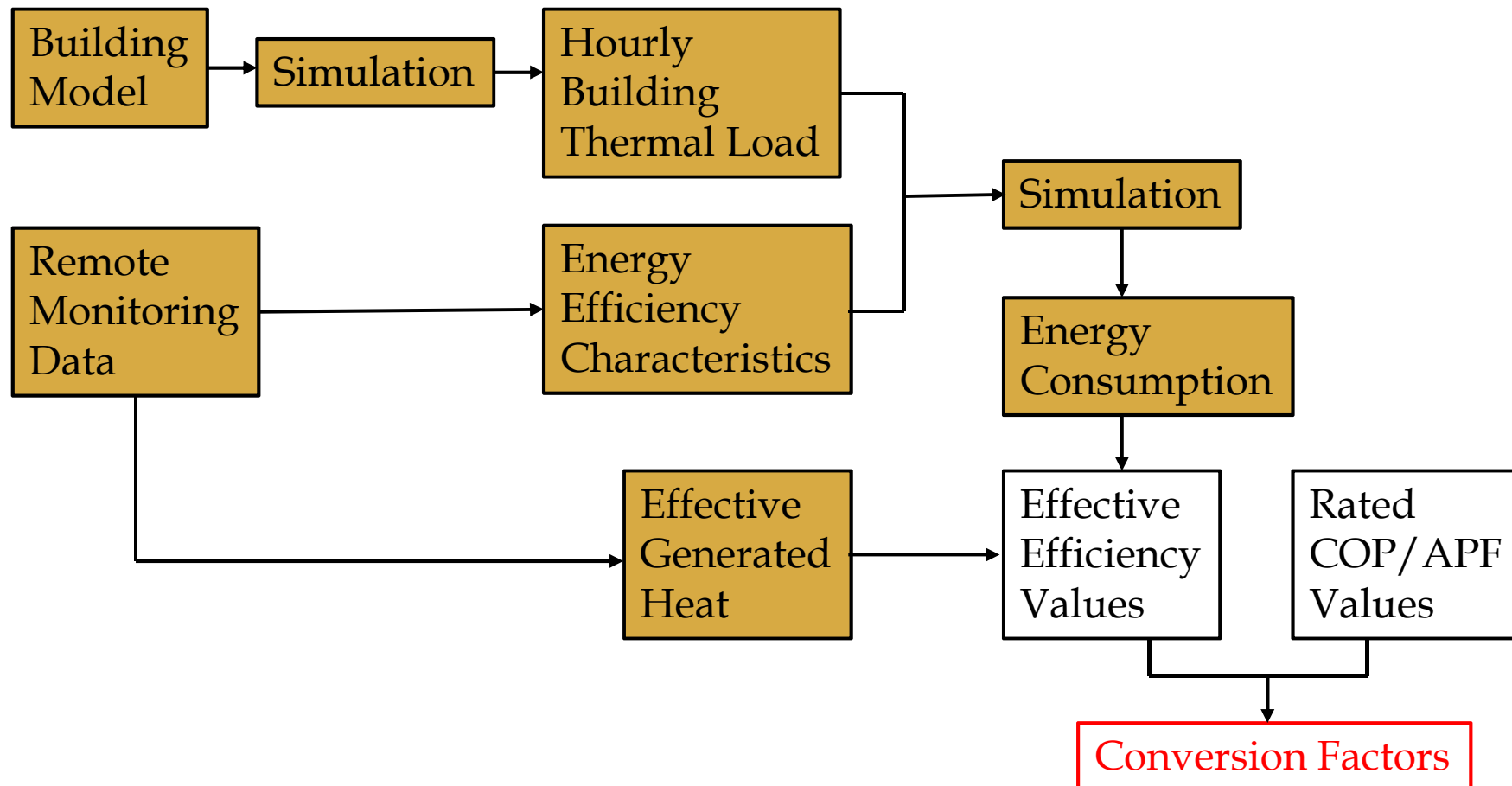
Energy consumption of the **APF** machine

Monthly air conditioning energy consumption

Month	COP machine (MJ/m ²)	APF machine (MJ/m ²)
January	45.6	29.4
February	42.9	29.0
March	39.8	25.8
April	15.9	8.9
May	48.9	32.8
June	87.9	58.4
July	106.9	70.2
August	115.2	70.4
September	92.4	62.0
October	40.9	24.5
November	19.3	10.4
December	46.9	25.8
Total	702.6	447.6



Study flow





Simulation Results



Conversion factors

$$\text{Effective efficiency} = \frac{\text{Annual effective generated heat}}{\text{Annual energy consumption}}$$

$$\text{Conversion factor} = \frac{\text{Effective efficiency}}{\text{Rated average COP or APF values}}$$

Values	COP machine	APF machine
Catalog value	3.17	3.71
Effective efficiency value	0.69	1.08
Conversion factor	0.22	0.29



Conclusions

- The performance of the COP and APF machines were different when comparing the energy efficiency characteristic diagrams.
- The conversion factors of the COP machine and the APF machine showed a difference.



- The performance of the machines can be easily compared for when evaluation standards change and new systems are installed.
- This approach can be applied to GHP (Gas driven heat pump) systems.



Thank you for taking the time

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