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# An Experimental Comparison of the Refrigerant Flow along Adiabatic and Non-adiabatic Coiled Capillary Tubes

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# Main topics of presentation

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- **Introduction**
- **Experimental setup**
- **Results and discussions**
- **Conclusion**



# Functions of expansion devices

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- **Reduces down stream pressure of condenser to up stream pressure of evaporator**
- **Regulate mass flow rate of refrigerant**



# Expansion device classification

(Refrigerant flow control devices)

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- *Fixed Restriction Area*
- *Variable Restriction Area*



# Expansion device classification

(Refrigerant flow control devices)



- *Variable Restriction Area*
  - **Thermostatic Expansion Valve (TXV)**
  - **Automatic Expansion Valve (AEV)**
  - **Hand Expansion Valves**
  - **Electronic Expansion Valve (EEV)**
  - ...





# Expansion device classification



(Refrigerant flow control devices)

- *Fixed Restriction Area*

- **Long Capillary Tubes**

- Straight	And	- Adiabatic
- Coiled		- Non-adiabatic

- **Short Orifices** → Adiabatic





# Comparison

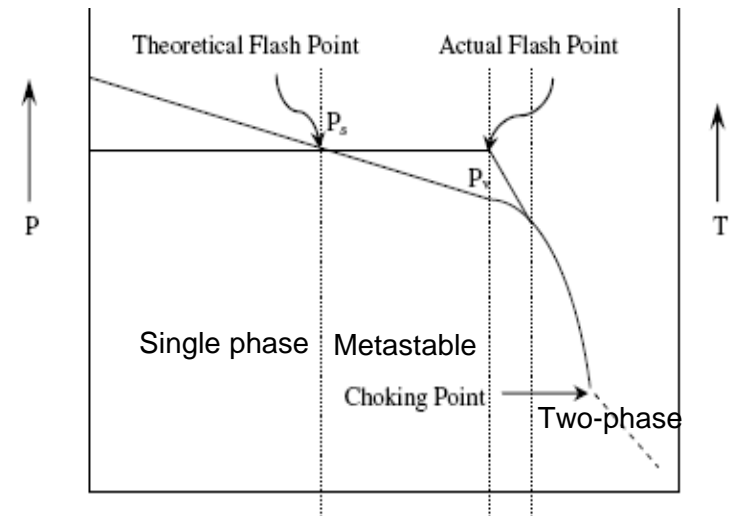
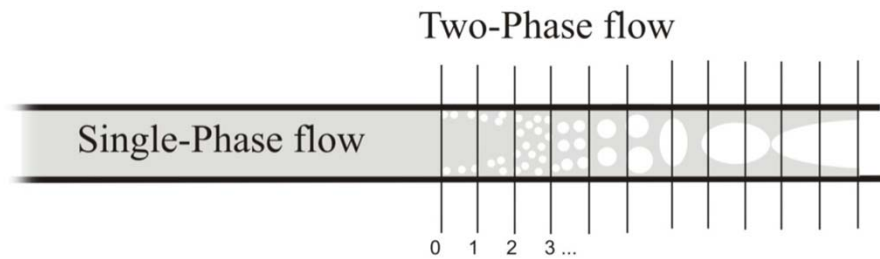
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- Expansion tubes: More durable and simpler
- Expansion Valves: Compatible with wide range of operating condition



# Flow Pattern Through Capillary Tube

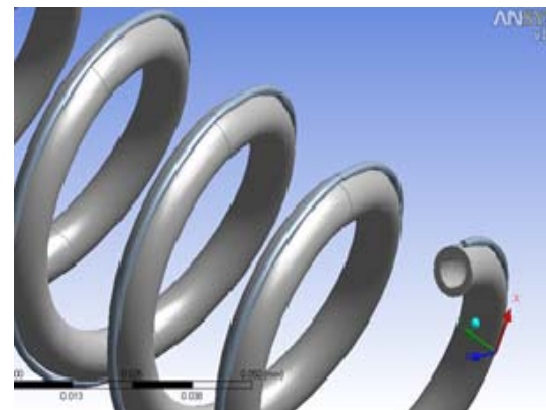
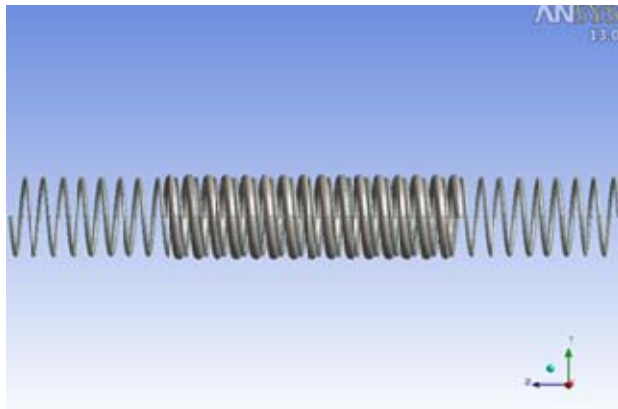






# Objectives of Experiment

- Effect of heat transfer on length of metastable region by :
- Soldering a suction line with counter flow to the coiled capillary tube to cool the flow
- Using discharged vapor from evaporator





# Objectives of Experiment

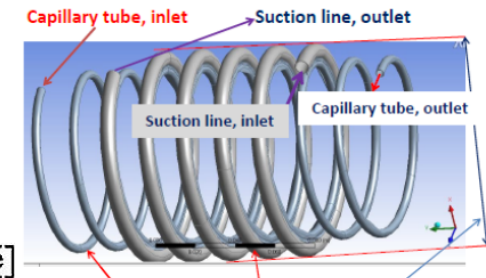
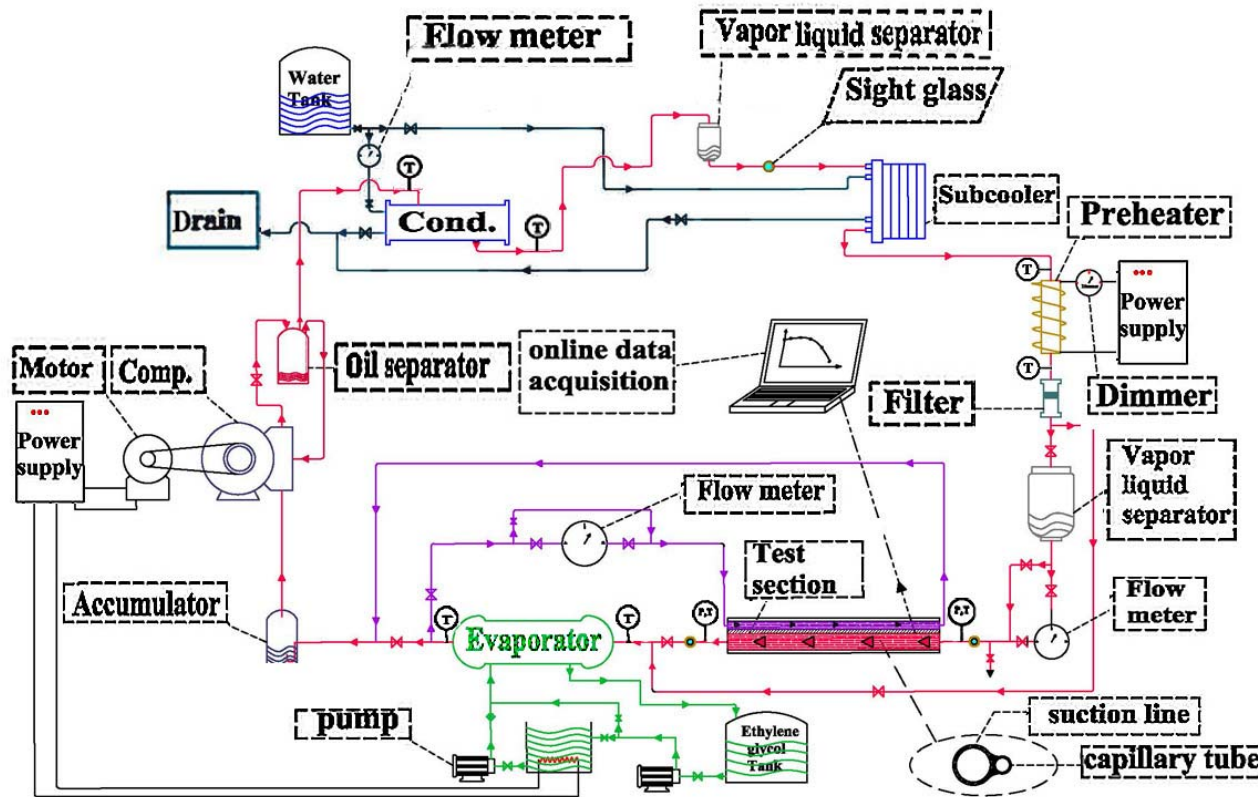


- Effect of **mass flux ratio** on **length of meta-stable region** and **inception of flashing point** through coiled tubes
- Mass flux ratio:

$$R = \frac{G_{capillary}}{G_{suction}}$$



# Experimental setup

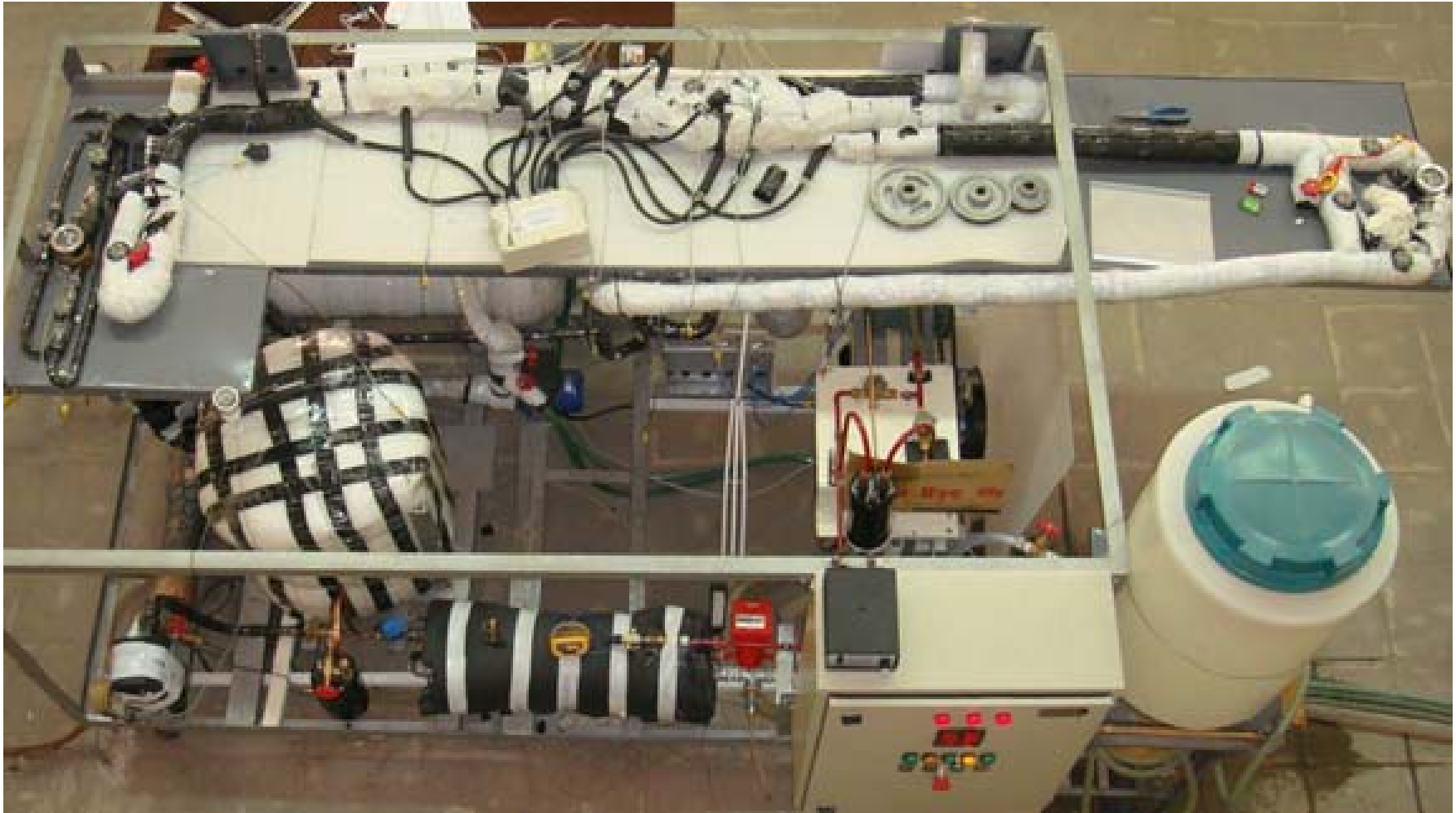


Capillary tube diameter:  $d=1.397\text{ mm}$   
Coiled diameter:  $D=40\text{ mm}$   
Suction tube inner diameter:  $D=4.0\text{ mm}$

b) Capillary tube with integrated heat exchanger



# Experimental setup

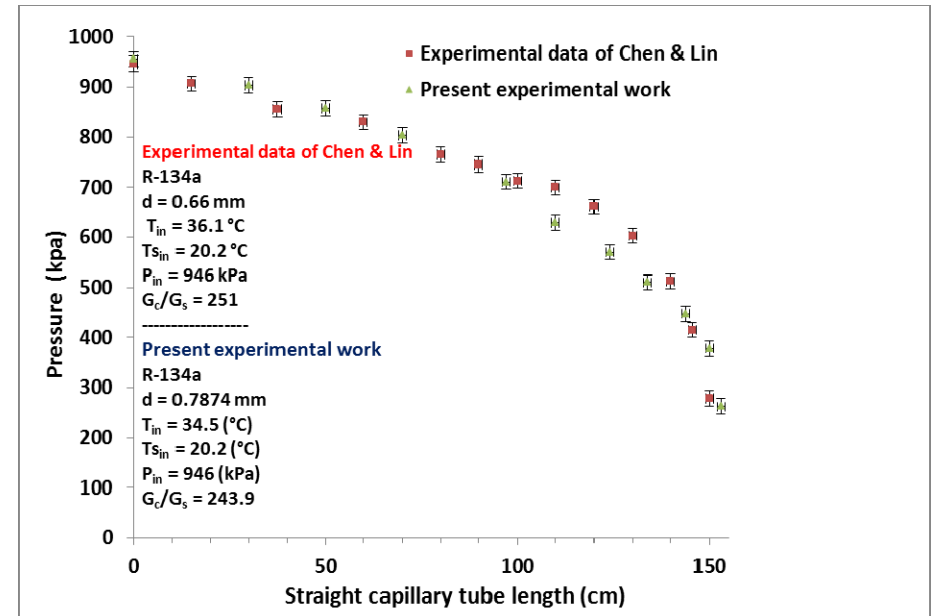
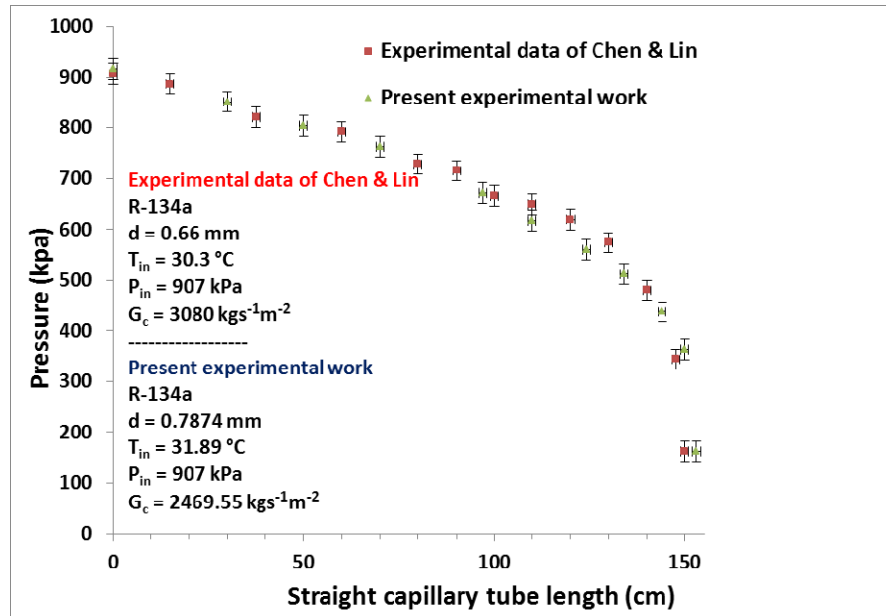




# Results:



## Validation with published data (Chen and Lin, 2001)

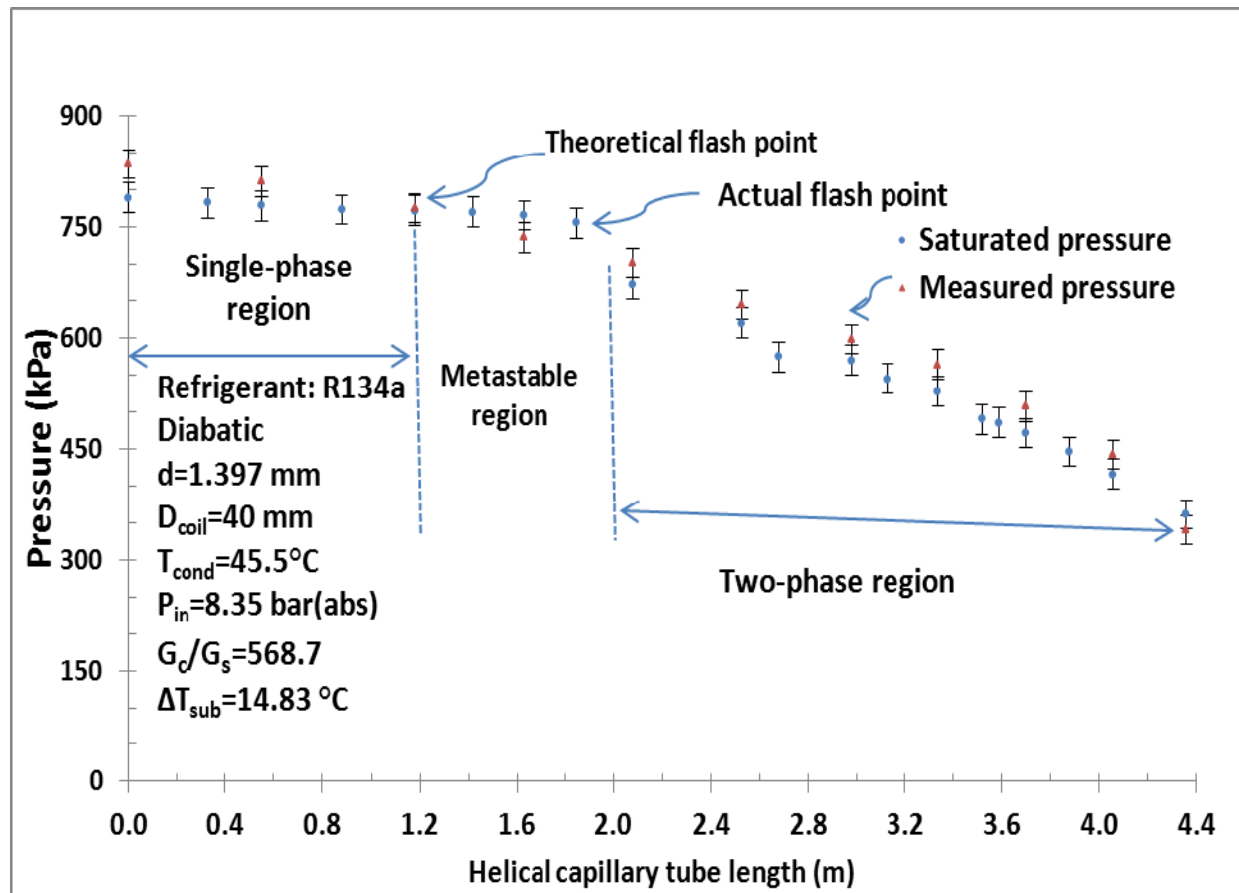




# Results:



## Flash point position on adiabatic flow for mass flux ratio=568

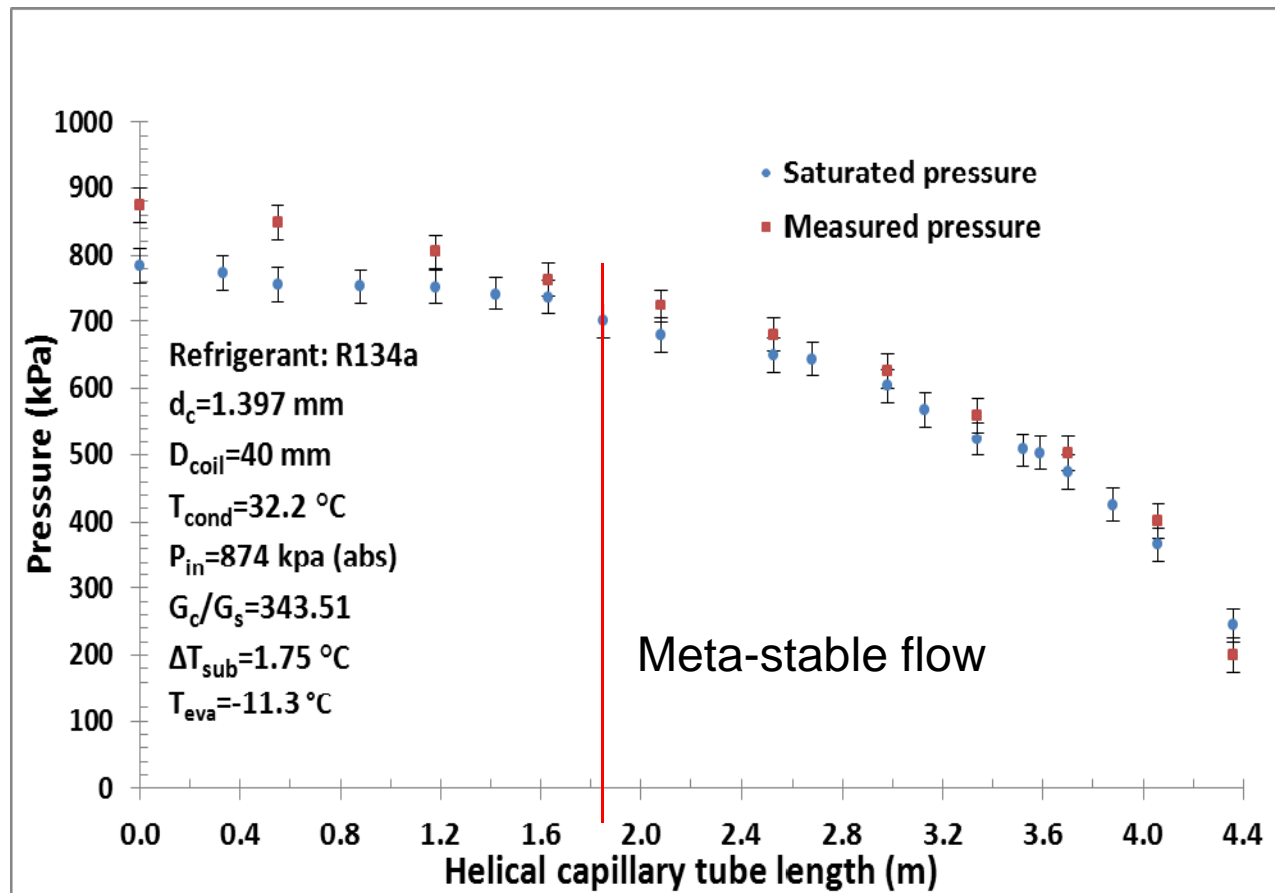




# Results:



## Flash point position on adiabatic flow for mass flux ratio=343





# CONCLUSIONS



1. Experimental study was presented for R134a flow through both non-adiabatic and adiabatic coiled capillary tubes
2. Maximum error of 9.5% between present and previously published data
3. Increase of heat transfer rate by decrease of the mass flux ratio in non-adiabatic condition
4. Weak heat transfer rate and metastable phenomena occurrence in mass flux ratio greater than 343
5. The present model can be used as a suitable tool for design and optimization of the vapor compression refrigeration systems (VCRS) with helical capillary tubes





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**Thank you for your attention**