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# Impact of Charge Degradation on the Life Cycle Climate Performance of a Residential Air-Conditioning System

## Paper 2277

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# INTRODUCTION



# Introduction

- Every vapor compression system has an optimum refrigerant charge
- A portion of this charge leaks out of the system annually
  - Performance degradation
  - System is charged during servicing

# LITERATURE REVIEW

# Literature Review

- Limited number of studies reporting the effect of charge degradation on vapor compression system performance
- Systems investigated include:
  - Residential and mobile air conditioning
  - Heat pumps (including air-to-air and water-to-water)
  - Water chiller systems
  - Walk-in coolers and freezers

# Literature Review

- Primary refrigerants tested include R-22 (most common), R-134a, R-32, R-404A, R-1234yf, and R-410A
- General trends observed with a drop in the system's charge include:
  - Decrease in system capacity, COP, and energy consumption
  - Increase in the suction superheat

# Literature Review

- ❖ Ambient temperature affects all systems for all expansion devices used
- ❖ Accumulator helps to maintain the system performance at its rated value for a charge reduction of up to 20%
- ❖ Systems with a capillary tube are very sensitive to a decrease in refrigerant charge, as opposed to orifice, TXV, and EEV



# CORRELATION DEVELOPMENT

# Correlation Development

- 🌱 Predict system capacity and power fractions in terms of the charge fraction and the ambient dry bulb temperature
- 🌱 The general trends fit well to a second degree polynomial

$$z = a + b \times x + c \times y + d \times x^2 + f \times y^2 + g \times x \times y$$

$z$ : the fraction of capacity, rated energy consumption, or COP

$x$ : is the ambient temperature in K

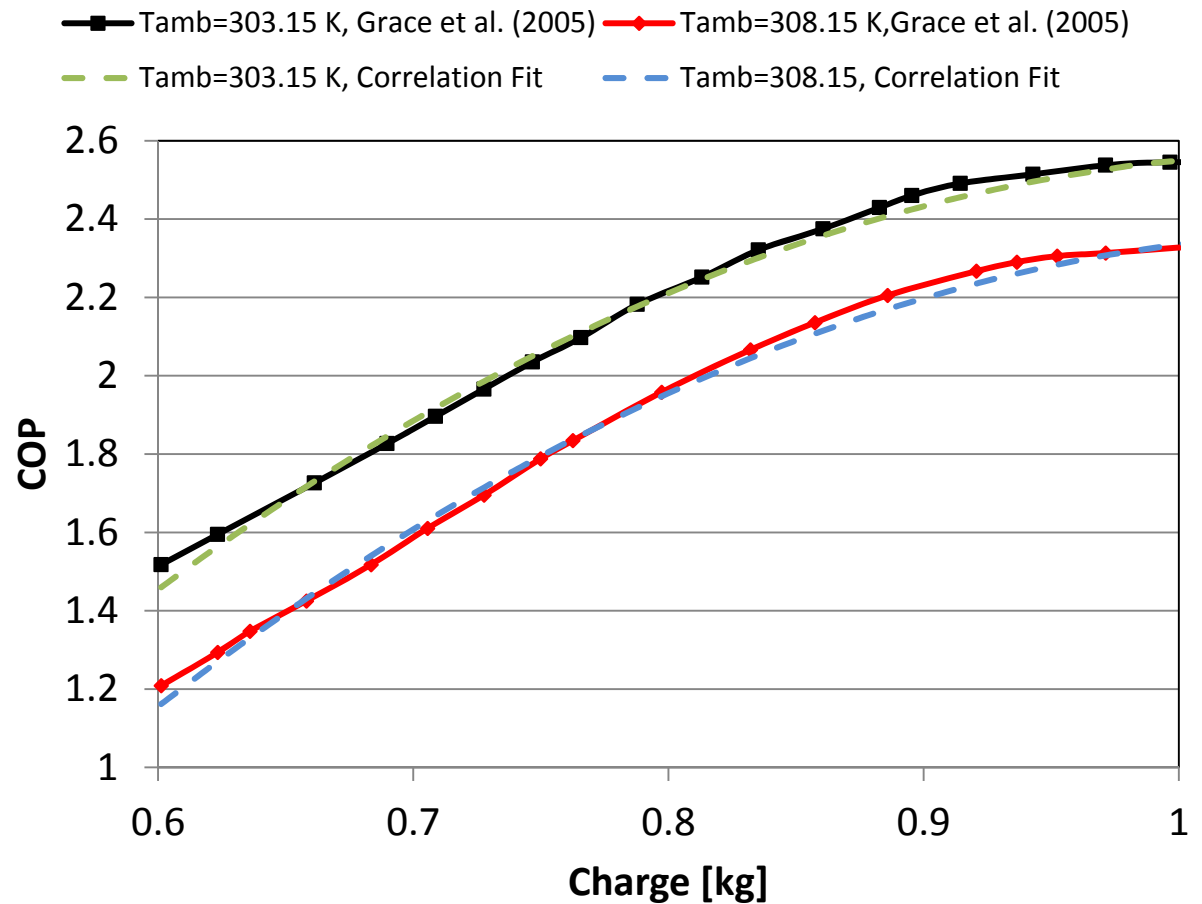
$y$ : is the charge fraction

$a, b, c, d, f$  and  $g$ : regression coefficients

# Correlation Development

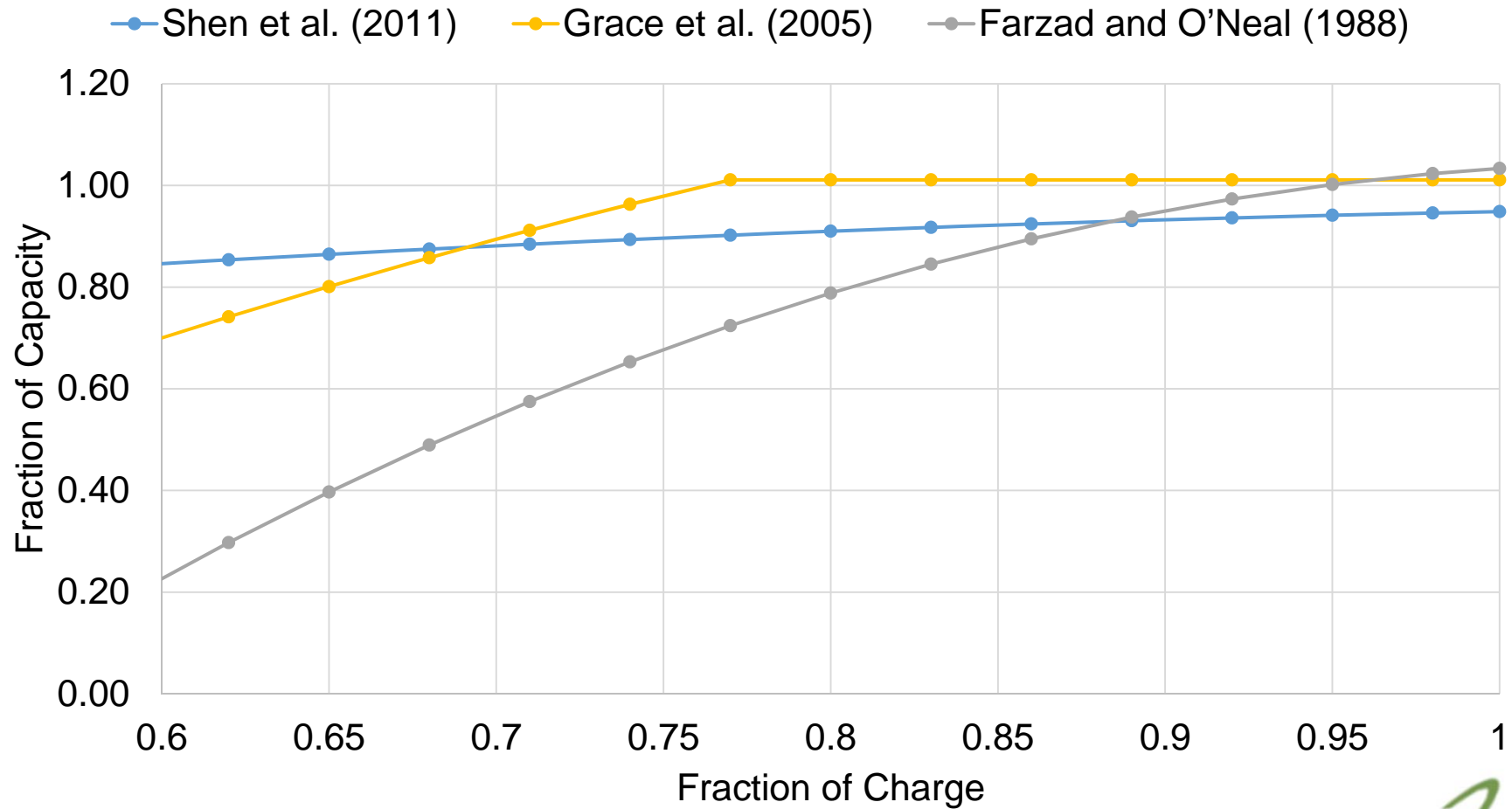
## Charge degradation correlations

Correlation	System/ Exp. Device
Shen et al. (2011)	Heat Pump/ TXV
Grace et al. (2005)	Water to water chiller/ TXV
Farzad and O'Neal (1988)	Air Conditioning/ Capillary Tube



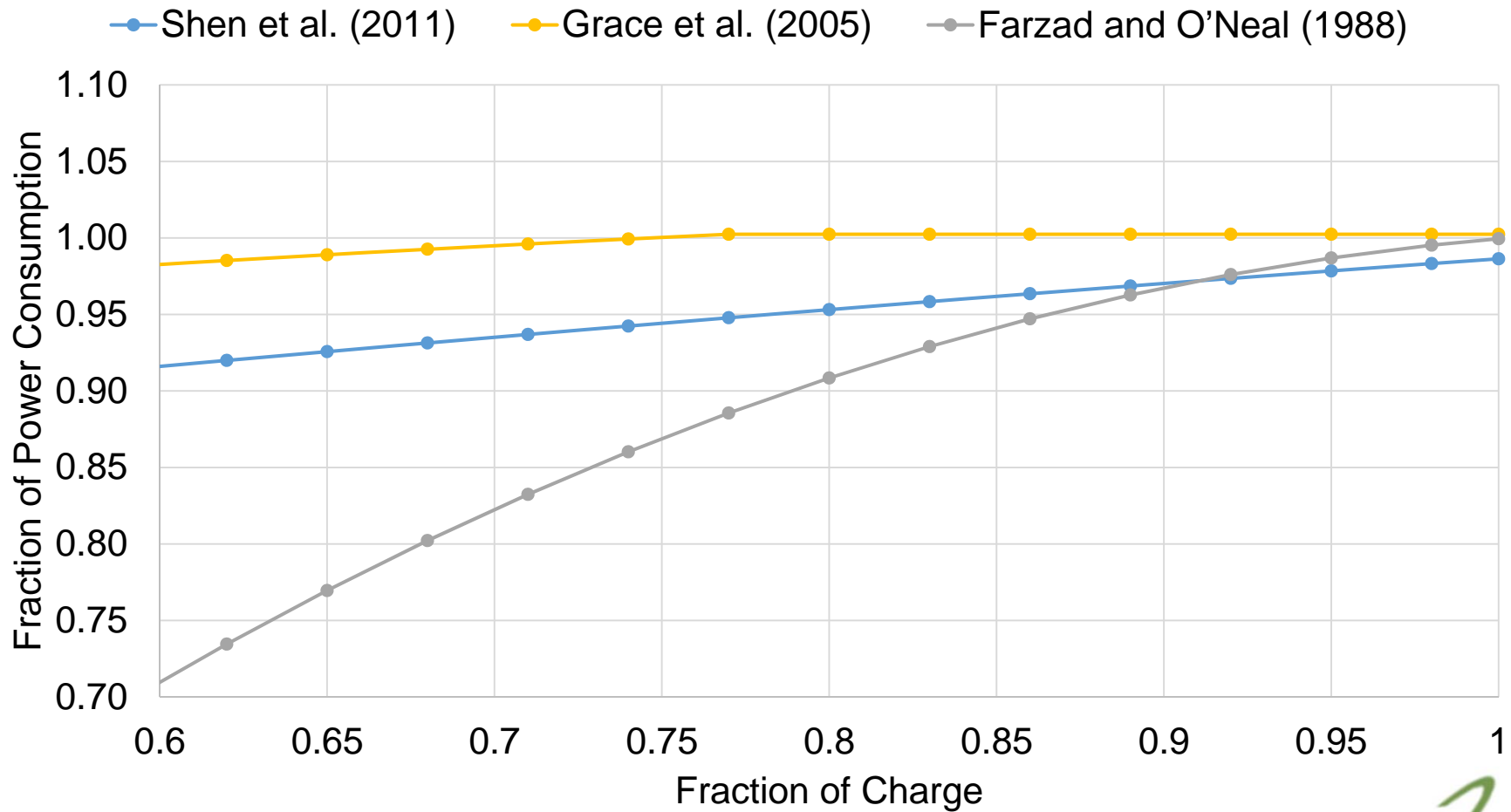
# Correlation Development

$T_{\text{amb}} = 310 \text{ K}$



# Correlation Development

$T_{\text{amb}} = 310 \text{ K}$



# LCCP ANALYSIS

# LCCP Analysis - System

<b>Manufacturer</b>	Goodman
<b>Outdoor Unit Model Number</b>	SSZ140361BA
<b>Indoor Unit Model Number</b>	ARUF374316
<b>Compressor</b>	Single Speed, Scroll Compressor
<b>Nominal Cooling Capacity</b>	3 Tons
<b>Rated SEER</b>	14
<b>Rated HSPF</b>	8.7
<b>Expansion Device</b>	TXV for Cooling/ Orifice for Heating

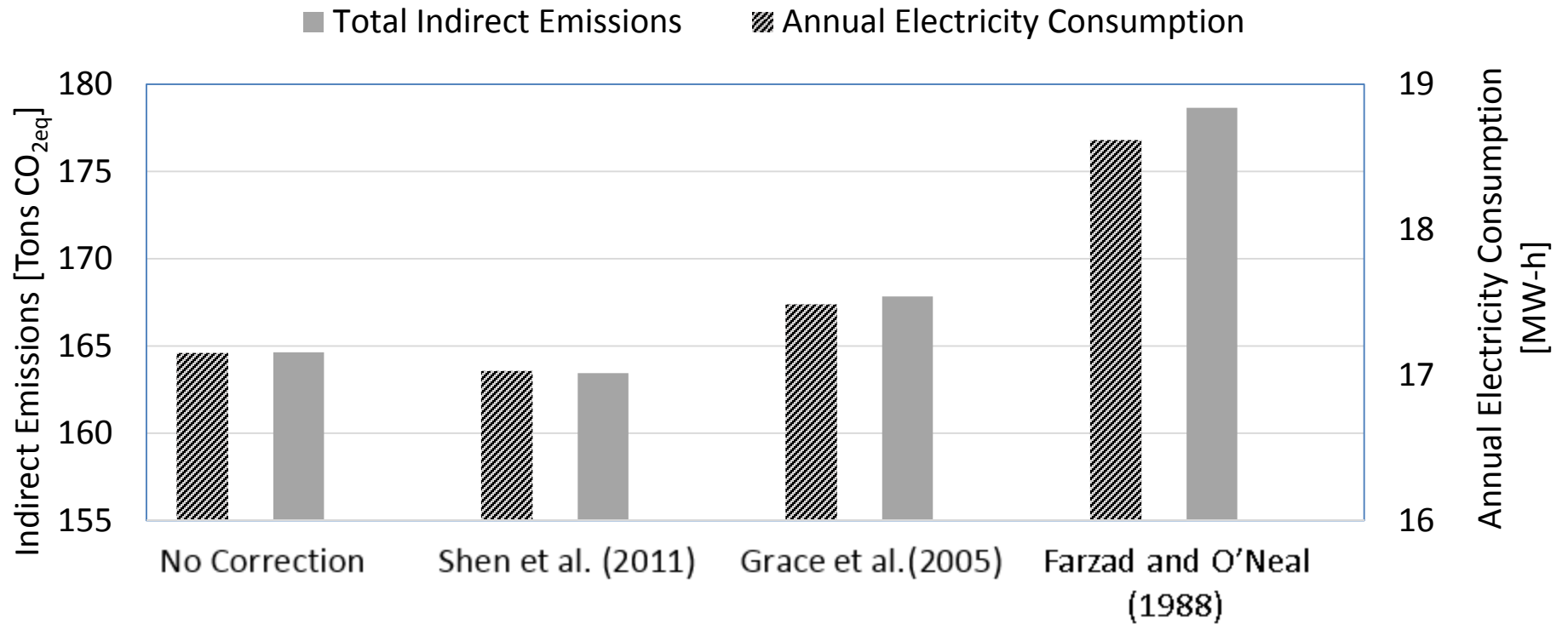
# LCCP Analysis - Inputs

- ASHP\* located in Chicago, IL
- Refrigerant: R410A with charge of 4.54 kg
- AHRI standard 210/240 for load calculations
- An in house vapor compression system simulation tool for system simulation

System Property	Value
Annual Leak Rate (%)	5
Ref Loss EOL (%)	15
System Lifetime (years)	15
Service Interval (years)	5
Reused Refrigerant (%)	85
Aluminum (kg)	19.1
Steel (kg)	75.9
Copper (kg)	31.2
Plastics (kg)	37.8



# LCCP Analysis - Results



$$\text{energy consumption} = \frac{\text{load}}{\text{capacity}} \times \text{power}$$

# CONCLUSIONS

# Conclusions

- Three main factors affect the sensitivity of system performance relative to refrigerant charge:
  - Ambient temperature
  - Use of an accumulator
  - Type of expansion device
- Three correlations for the capacity and rated energy consumption fractions as a function of charge fraction and ambient temperature are presented

# Conclusions

- ❖ Applying these correlations to an ASHP tends to give much higher total emissions of the system
- ❖ Farzad and O'Neal (1988) correlation gives the highest energy consumption as it is developed for systems with a capillary tube

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# Thank You



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# Literature Review

Study	System	Charge	Conclusion
Trane Co.	A/C	90%	Efficiency is 20% lower
Domingorena	3-ton air-to-air heat pump, capillary tube, no accumulator	63-127%	<ul style="list-style-type: none"> <li>• Highly sensitive to undercharging</li> <li>• Almost insensitive to overcharging</li> <li>• COP and capacity decrease almost linearly</li> </ul>
Domingorena and Ball	3-ton air-to-air heat pump, orifice, suction line accumulator	63-127%	Using the accumulator helped to maintain the COP of the system at its rated value for more charge reduction (around 20%)
Houcek and Thedford	1.5 ton split A/C	77-123%	For the undercharging conditions, the capacity dropped by 23% at 301 K and 34% at 308 K
Farzad and O'Neal	3-ton A/C, capillary tube, no accumulator	75-125%	Total capacity, EER, and SEER are more sensitive to undercharging than overcharging conditions
Farzad	3-ton A/C, no accumulator, three expansion devices: a short-tube orifice, a thermostatic expansion valve, and a capillary tube	80-120%	<ul style="list-style-type: none"> <li>• System using a capillary tube is more sensitive to the off-design charging than the system using either the TXV or the short-tube orifice</li> <li>• The performance of the system with the TXV and the short tube orifice is strongly dependent on the outdoor air temperature</li> </ul>
Robinson	3-ton A/C, orifice	80-120%	

# Literature Review

Study	System	Charge	Conclusion
Rodriguez	3-ton A/C, no accumulator, a short-tube orifice 3.5-ton A/C, no accumulator, a thermostatic expansion valve	60-130%	<ul style="list-style-type: none"> <li>Charge reduction more than 20% causes a sharp decrease in capacity and EER</li> <li>Overcharging the system more than 20% did not have much impact on performance</li> </ul>
Bailey	70-ton dual circuit, air-cooled chiller	40-115%	Chiller power per ton of refrigerating capacity is directly proportional to the outdoor air temperature and the refrigerant charge level
Goswami et al.	3-ton A/C	50%	For a 10% reduction in charge, the capacity decreases by 3.5% while the COP increases by 2%
Choi and Kim	3.5 ton water-to-water heat pump, electronic expansion valve (EEV) and a capillary tube	80-120%	<ul style="list-style-type: none"> <li>The system using the EEV was almost insensitive to the change in charge while it was sensitive to the outdoor conditions</li> <li>The system using the capillary tube was sensitive to both the charge and outdoor conditions with the capacity and COP decreasing for both undercharging and overcharging</li> </ul>

# Literature Review

Study	System	Charge	Conclusion
Grace et al.	1.1 ton water-to-water chiller, a thermostatic expansion valve	50-140%	<ul style="list-style-type: none"> <li>Charge degradation has a slight effect on the system performance in the charge range from 75% to 125%</li> <li>Further reduction in the charge caused a sharp drop in the system performance</li> </ul>
Kruse and Palmiter	3-ton heat pump, suction-line accumulator, a short-tube orifice and a thermostatic expansion valve	70-130%	Heat pump without a suction-line accumulator would be much more sensitive to variations in charge level
Shen et al.		60-150%	
Wichman and Braun	Walk-in coolers and freezers, a thermostatic expansion valve and a liquid line receiver	50-240%	<ul style="list-style-type: none"> <li>TXV helped to maintain system performance until the charge reached 80%</li> <li>Further decrease in charge caused a sharp decrease in the capacity</li> </ul>

# Literature Review

Study	System	Charge	Conclusion
Kim, and Braun	six different residential air conditioning systems with different components	40-130%	for systems with a fixed expansion orifice (FXO), the accumulator had little impact on the charge degradation effect on the capacity. for the systems using the TXV, below a charge limit of 70%, the TXV starts to operate similar to the FXO as the TXV becomes wide open
Huyghe	mobile air conditioning system used in a mid-size sedan, thermostatic expansion valve	25-100%	system performance began to significantly decrease when the refrigerant charge dropped below 70%