

# Simultaneous Control of Indoor Temperature and Humidity Using a VS DX A/C System

A PD Law based Fuzzy Logic Controller

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# Simultaneous Control of Indoor Temperature and Humidity Using a VS DX A/C

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- Introduction
- Development of the controller
- The experimental setup and conditions
- The experimental results
- Conclusions



- The application of DX A/C system

Large scaled buildings: Chilled water based central air conditioner (A/C)

Small-to medium-scaled buildings: Direct Expansion (DX) A/C system

Advantages: compact, flexible, energy efficient, cost less.

- Conventional control strategy

ON / OFF control: leave the indoor air humidity uncontrolled

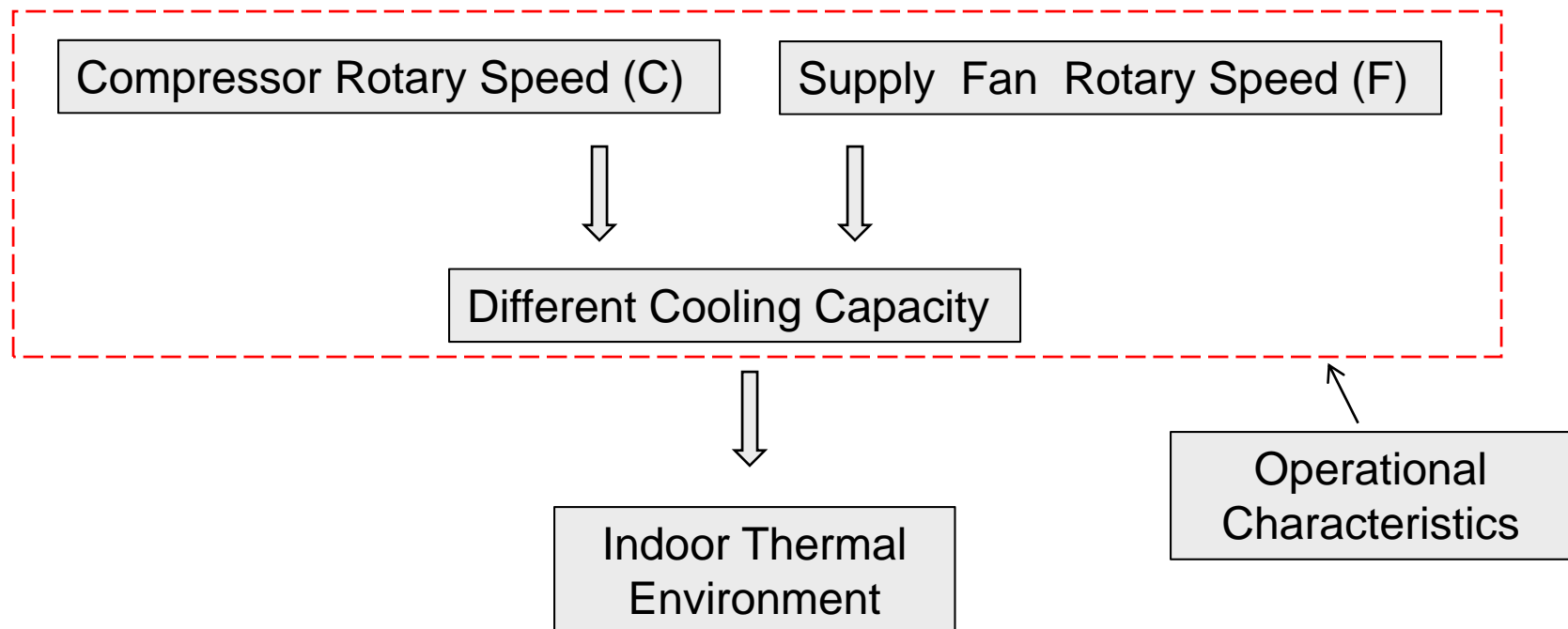
Simultaneous control of indoor temperature and humidity:

Total Cooling Capacity= Indoor cooling load & E SHR=A SHR



- Variable Speed Driven

VSD → Continues speed variation for Fan & Compressor





- Some novel control strategies

Physical model based controller:

DDC based Capacity controller; MIMO Controller

Empirical model based controller:

ANN based Online adaptive Controller

Simple PID controller:

Control algorithm: C → Indoor air temperature  
F → Indoor air humidity

H-L Control:

Control algorithm: No OFF period for Compressor,  
but high and low rotary speed

Performance  
promised but  
complicated



Conflict

Structure  
Simplified but  
without strict  
control of  
parameters



## ● Proposition

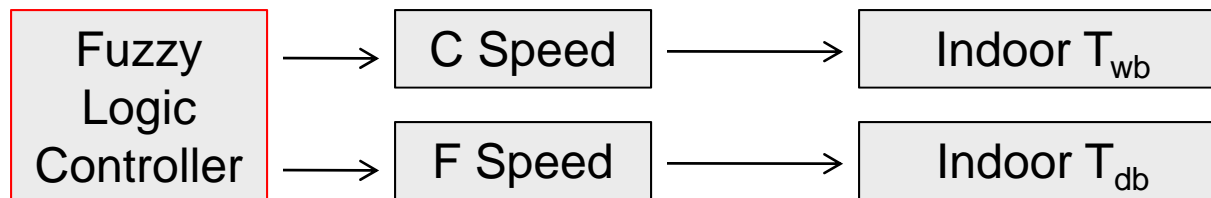
Certain logic : Using accurate mathematical and physical model

Fuzzy logic : Using common sense and experience of people



Effective means of capturing the approximate and inexact nature of the real world

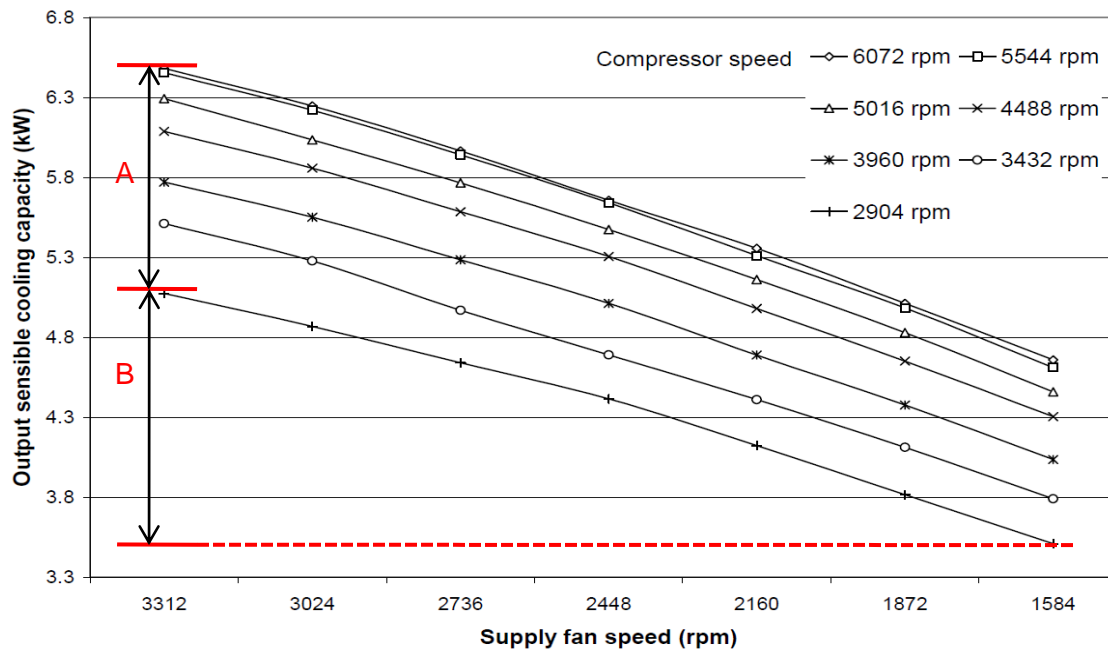
Proposed Control Strategy:





## ● Proposed control principle

Operational characteristics:



A: Sensible cooling capacity change due to C speed variation

B: Sensible cooling capacity change due to F speed variation

$$A < B$$

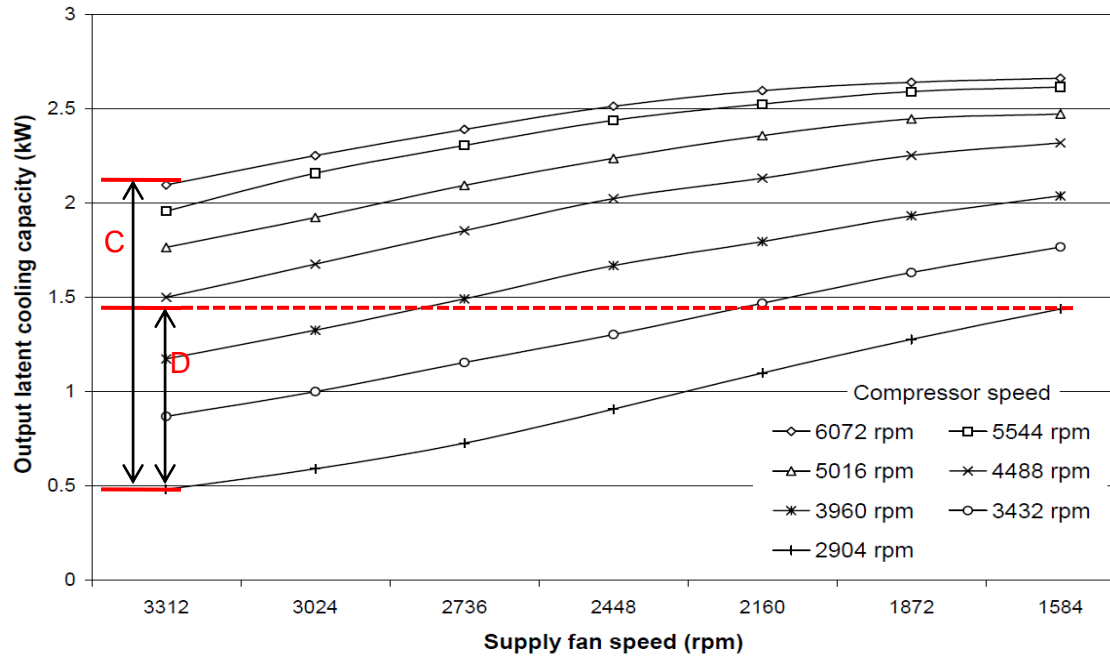
Output sensible cooling capacity ( $Q_s$ ) at different speeds combination

The influence caused by varying supply fan speed on the output sensible cooling capacity is more considerable than that caused by varying compressor speed



## ● Proposed control principle

Operational characteristics:



C: Latent cooling capacity change due to C speed variation

D: Latent cooling capacity change due to F speed variation

$$C > D$$

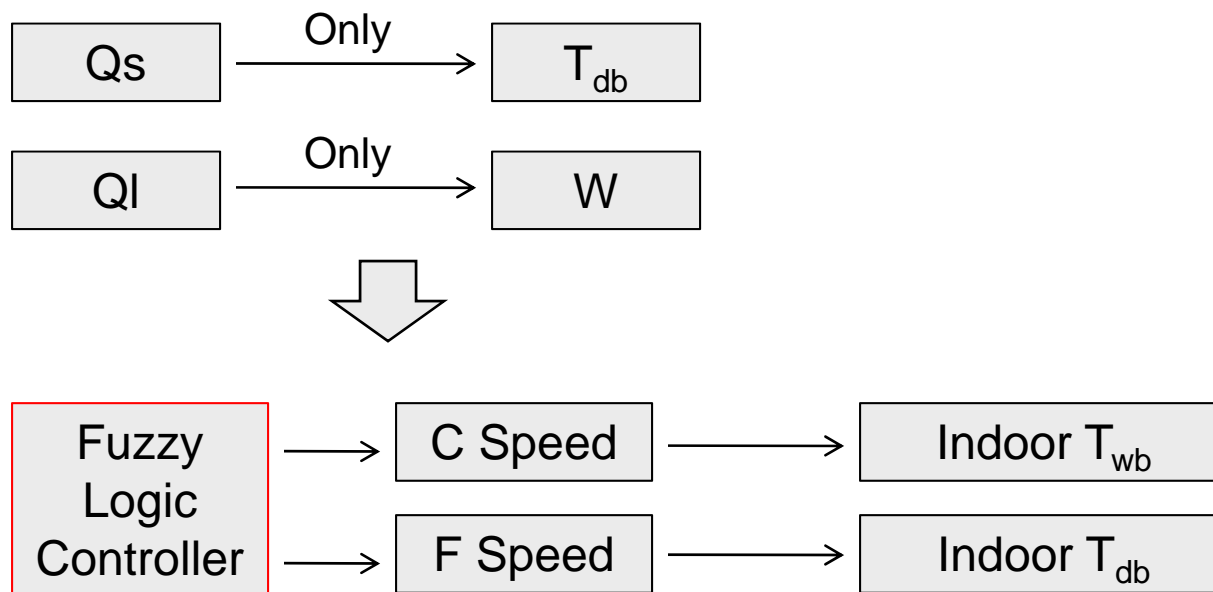
Output latent cooling capacity (Ql) at different speeds combination

The output latent cooling capacity will be influenced more by varying compressor speed than by varying supply fan speed





- Proposed control principle

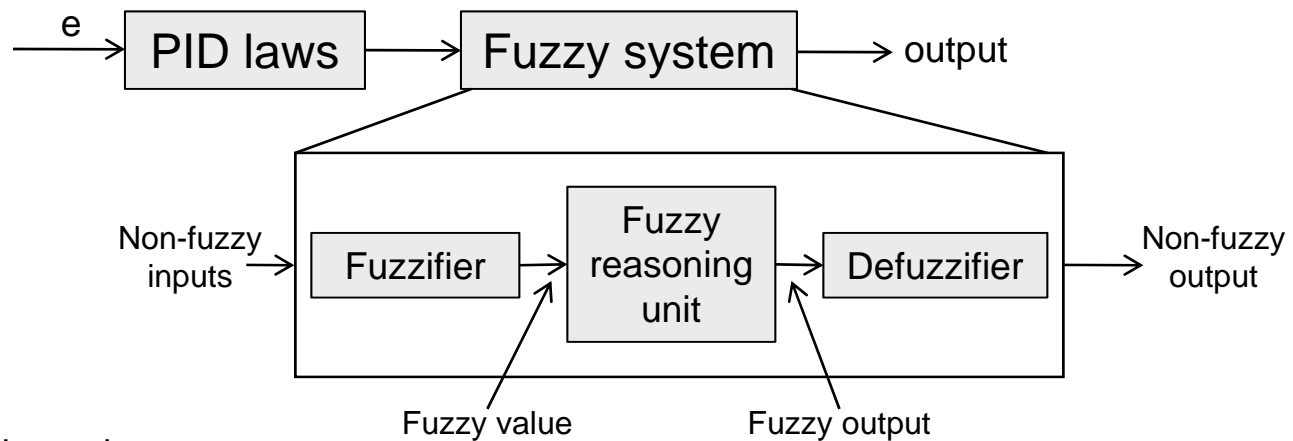


Controlled indoor  $T_{db}$  &  $T_{wb}$  determine the indoor thermal environment, realizing simultaneous control of temperature and humidity.



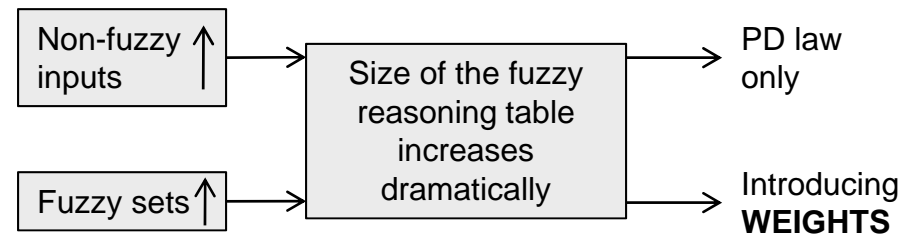
## ● Control algorithm

Conventional PFC:



Fuzzy reasoning unit:

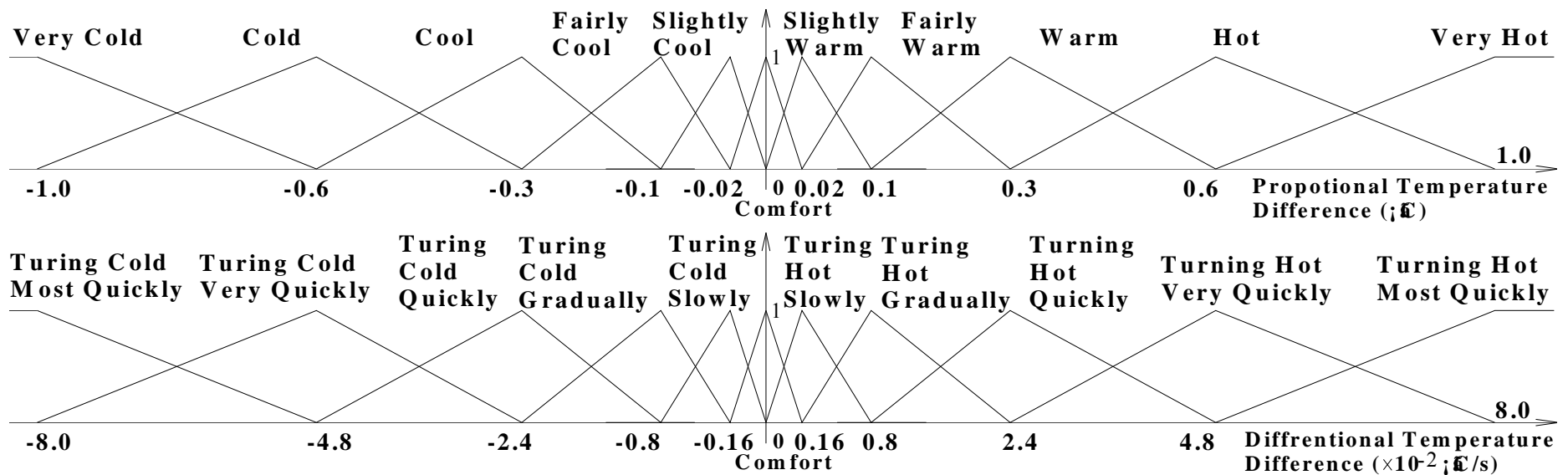
$e \downarrow \Delta e \rightarrow$	NB	NM	NS	ZE	PS	PM	PB
NB	NB	NB	NB	NM	NS	ZE	PS
NM	NB	NB	NM	NS	ZE	PS	PM
NS	NB	NM	NS	NS	PS	PM	PB
ZE	NB	NM	NS	PS	PS	PM	PB
PS	NB	NM	NS	PS	PS	PM	PB
PM	NM	NS	ZE	PS	PM	PB	PB
PB	NS	ZE	PS	PM	PB	PB	PB





## ● Control algorithm

The membership function for  $T_{db}$  control:



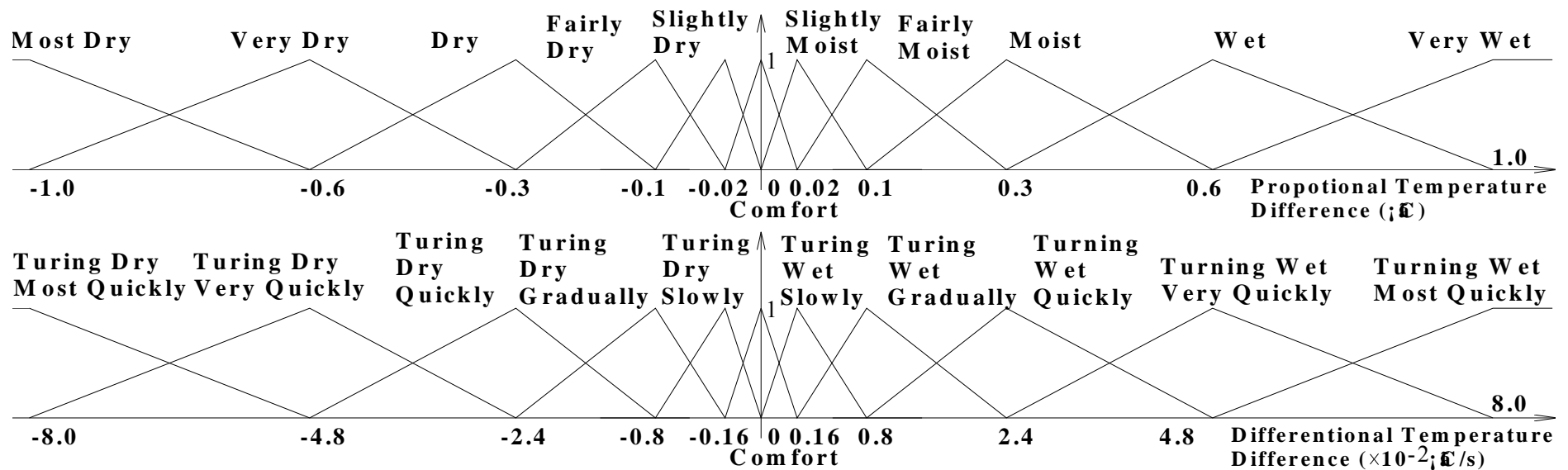
Very Cold...: Linguistic Variables, defined in different fuzzy sets

One membership function for each inputs of the fuzzifier: proportional temperature difference & differential temperature difference



## ● Control algorithm

The membership function for  $T_{wb}$  control:



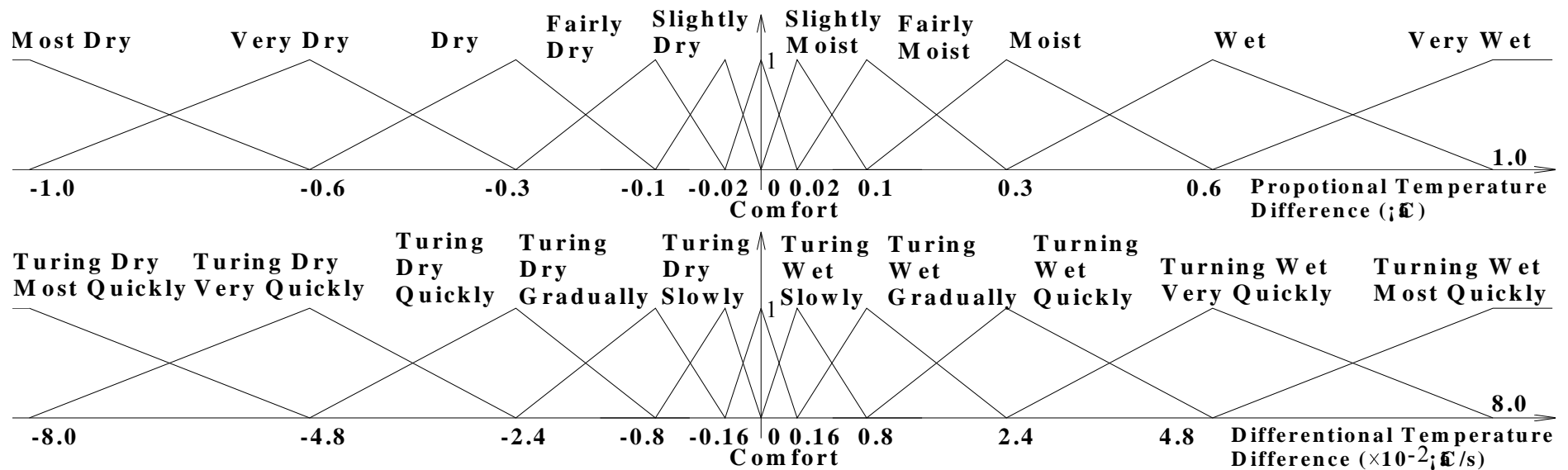
Most Dry...: Linguistic Variables, defined in different fuzzy sets

One membership function for each inputs of the fuzzifier: proportional temperature difference & differential temperature difference



## ● Control algorithm

The membership function for  $T_{wb}$  control:



$e \downarrow \Delta e \rightarrow$	NB	NM	NS	ZE	PS	PM	PB
NB	NB	NB	NB	NM	NS	ZE	PS
NM	NB	NB	NM	NS	ZE	PS	PM
NS	NB	NM	NS	NS	PS	PM	PB
ZE	NB	NM	NS	PS	PS	PM	PB
PS	NB	NM	NS	PS	PS	PM	PB
PM	NM	NS	ZE	PS	PM	PB	PB
PB	NS	ZE	PS	PM	PB	PB	PB



11×11=121 fuzzy rules for each control variable and most of them will be the same



## ● Control algorithm

The Weights for different Linguistic Variables:

Inputs	Linguistic Variables and Weights										
$\Delta T_{db}$	VH	H	W	FW	SW	Com	SC	FC	Cool	Cold	VC
	5.0	1.8	1.0	0.6	0.1	0	-0.1	-0.6	-1.0	-1.8	-5.0
$dT_{db}/dt$	THMQ	THVQ	THQ	THG	THS	Com	TCS	TCG	TCQ	TCVQ	TCMQ
	5.0	2.0	1.5	0.8	0.1	0	-0.1	-0.8	-1.5	-2.0	-5.0
$\Delta T_{wb}$	VW	Wet	Moist	FM	SM	Com	SD	FD	Dry	VD	MD
	5.0	1.8	1.0	0.6	0.1	0	-0.1	-0.6	-1.0	-1.8	-5.0
$dT_{wb}/dt$	TWMQ	TWVQ	TWQ	TWG	TWS	Com	TDS	TDG	TDQ	TDVQ	TDMQ
	5.0	2.0	1.5	0.8	0.1	0	-0.1	-0.8	-1.5	-2.0	-5.0

$$dx = \sum [f(n)_i \cdot W]$$

dx: degree of changes in C (dC) & F (dF),  $f(n)_i$ : grade of membership, W: weights for linguistic variables

Different inputs can lead to same dx and there's no need to build the 11x11 fuzzy reasoning table.



## ● Control algorithm

The Weights for different Linguistic Variables:

Inputs	Linguistic Variables and Weights										
$\Delta T_{db}$	VH	H	W	FW	SW	Com	SC	FC	Cool	Cold	VC
	5.0	1.8	1.0	0.6	0.1	0	-0.1	-0.6	-1.0	-1.8	-5.0
$dT_{db}/dt$	THMQ	THVQ	THQ	THG	THS	Com	TCS	TCG	TCQ	TCVQ	TCMQ
	5.0	2.0	1.5	0.8	0.1	0	-0.1	-0.8	-1.5	-2.0	-5.0
$\Delta T_{wb}$	VW	Wet	Moist	FM	SM	Com	SD	FD	Dry	VD	MD
	5.0	1.8	1.0	0.6	0.1	0	-0.1	-0.6	-1.0	-1.8	-5.0
$dT_{wb}/dt$	TWMQ	TWVQ	TWQ	TWG	TWS	Com	TDS	TDG	TDQ	TDVQ	TDMQ
	5.0	2.0	1.5	0.8	0.1	0	-0.1	-0.8	-1.5	-2.0	-5.0

$$dx = \sum [f(n) \cdot \mathbf{W}]$$

$$C(t+1) - C(t) = (C_{\max} - C(t)) / 10 \times dC, \quad dC \geq 0$$

$$C(t+1) - C(t) = (C(t) - C_{\min}) / 10 \times dC, \quad dC \leq 0$$

$$F(t+1) - F(t) = (F(t) - F_{\max}) / 10 \times dF, \quad dF \geq 0$$

$$F(t+1) - F(t) = (F_{\min} - F(t)) / 10 \times dF, \quad dF \leq 0$$

C, F: percentage of the maximum rotary speed for compressor and supply fan, in %.

$C_{\max}$ ,  $F_{\max}$ : maximum output control signals for C and F, set at 90%.

$C_{\min}$ ,  $F_{\min}$ : minimum output control signals for C and F, set at 25%.

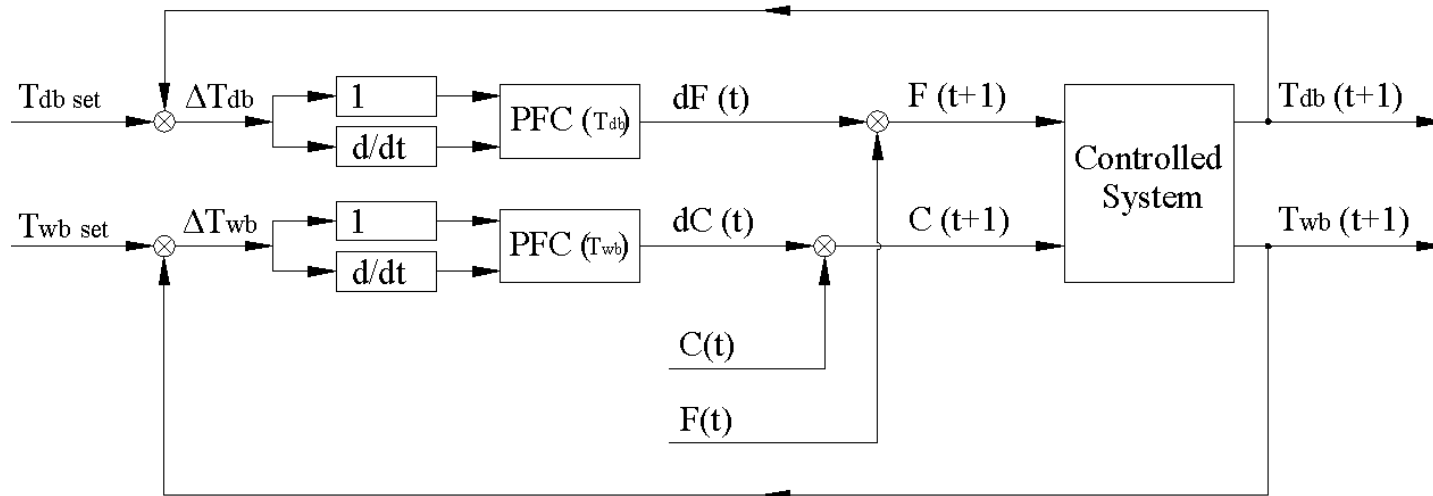
t: present time step.

No need to establish membership functions for defuzzifying stage



## ● Control algorithm

The completed block diagram for the controller developed:





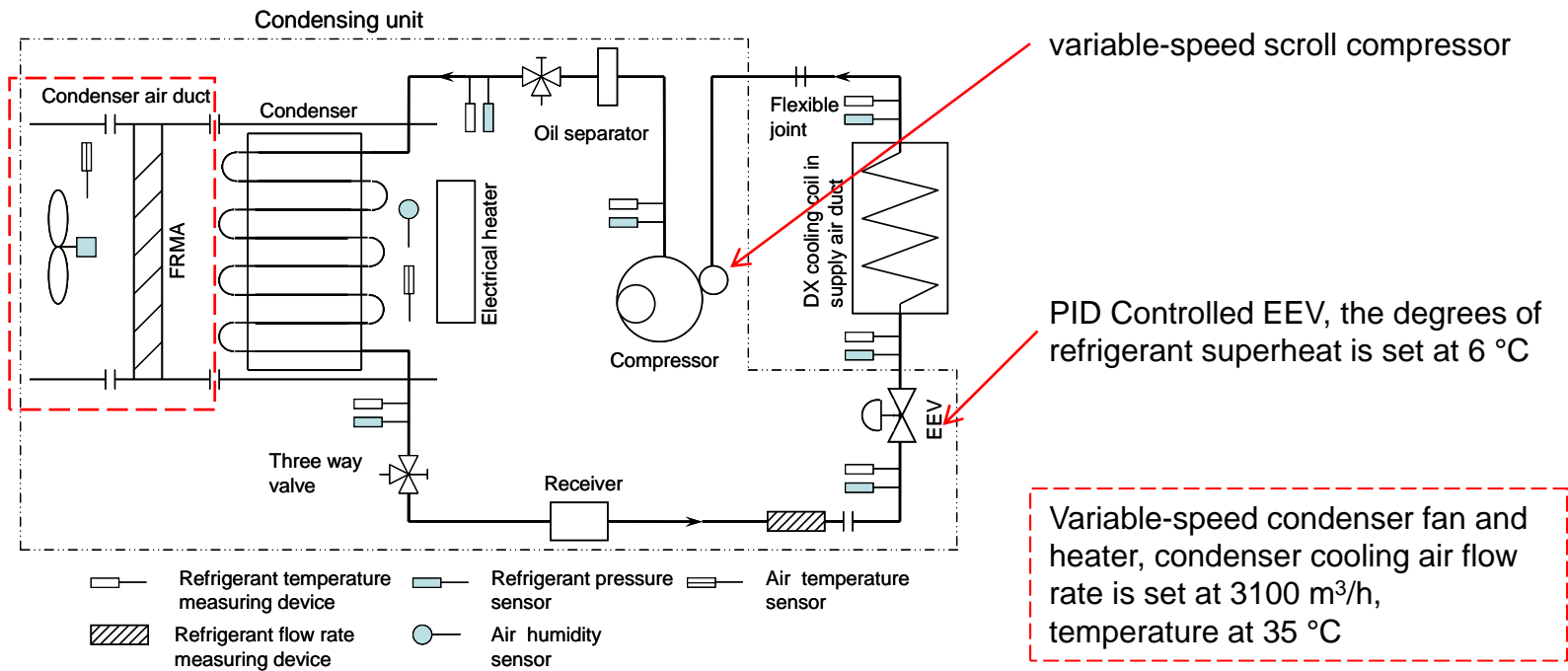


# The experimental setup



## ● The experimental VS DX A/C system

The DX refrigeration plant:



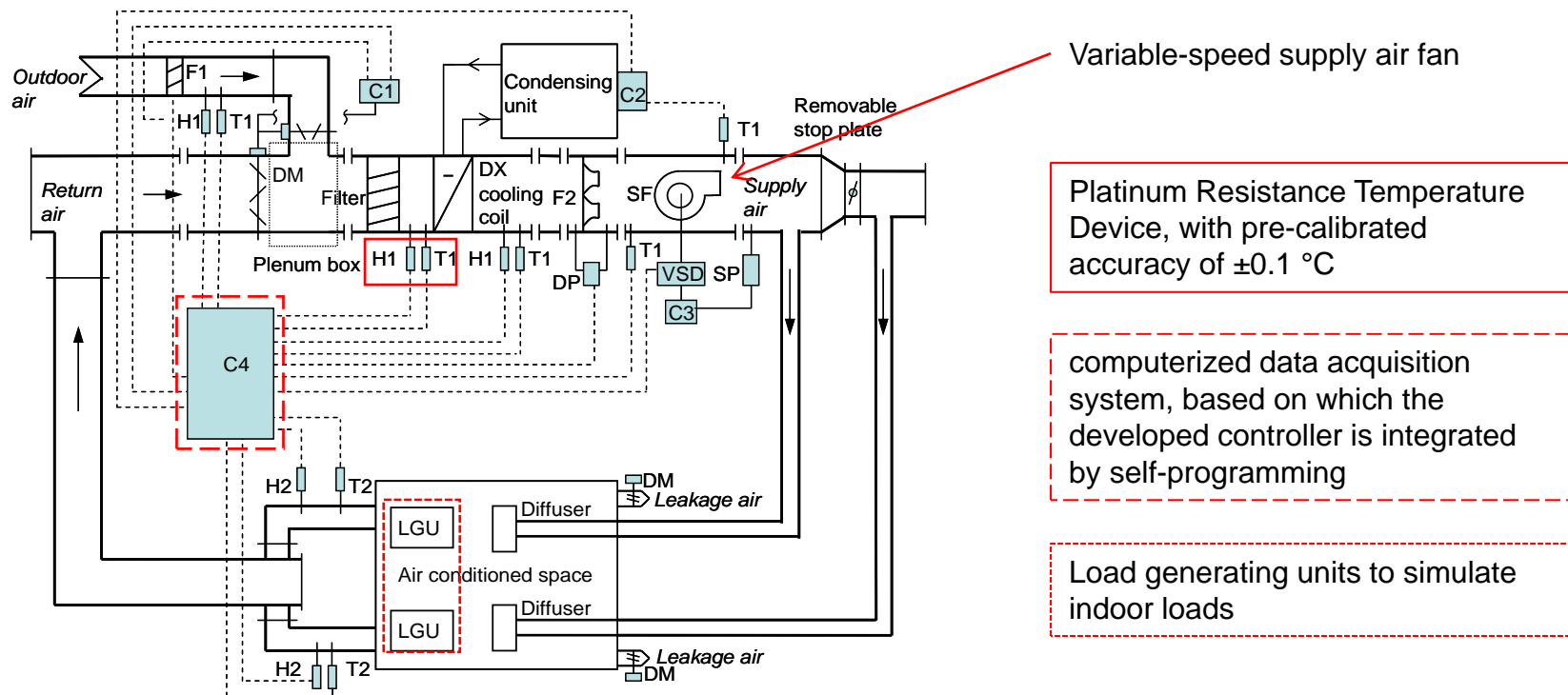


# The experimental setup



## ● The experimental VS DX A/C system

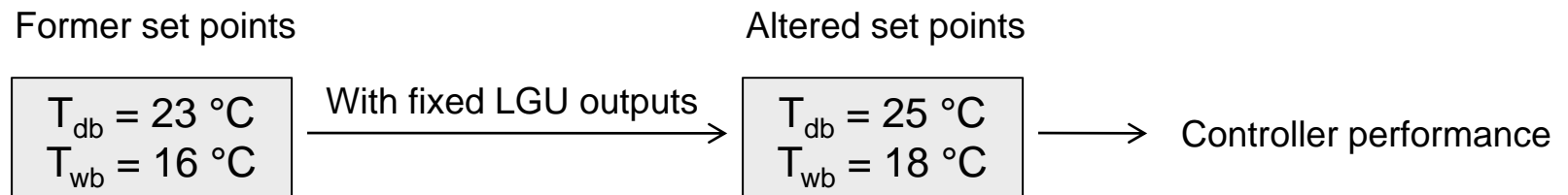
The air distributing sub system:



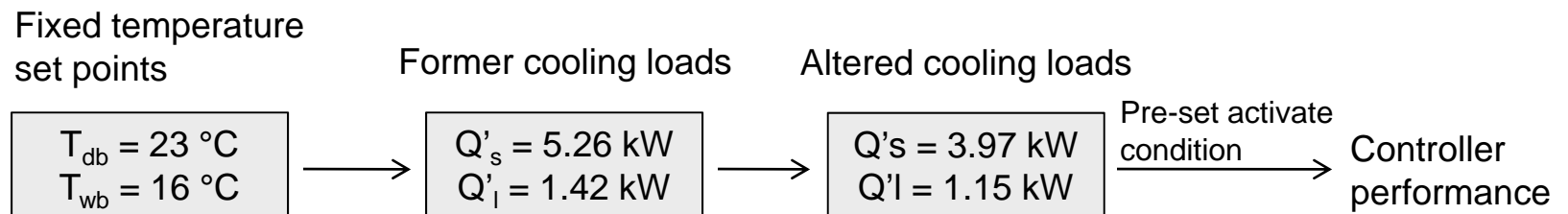


## ● The experimental conditions

The command following test: to test controller's ability of reacting to the changes of set points

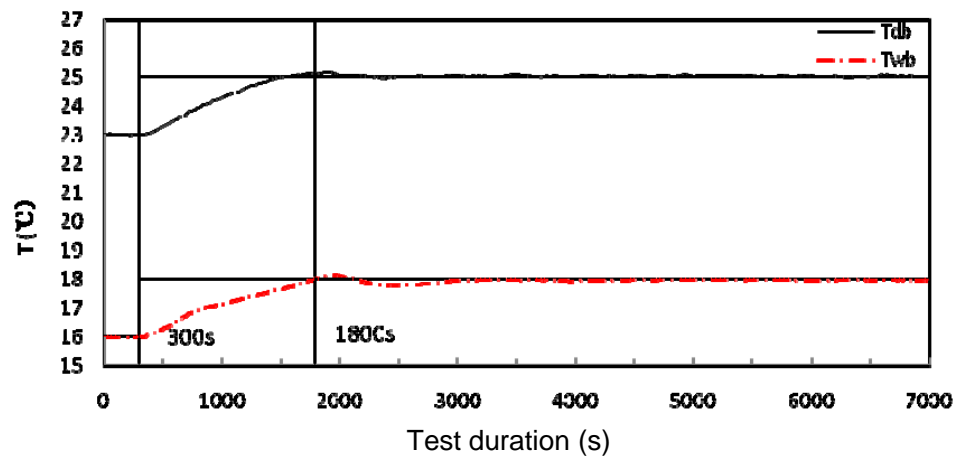


The disturbance rejection test: to test the controller's ability of resisting the disturbance caused by the variation of indoor cooling loads



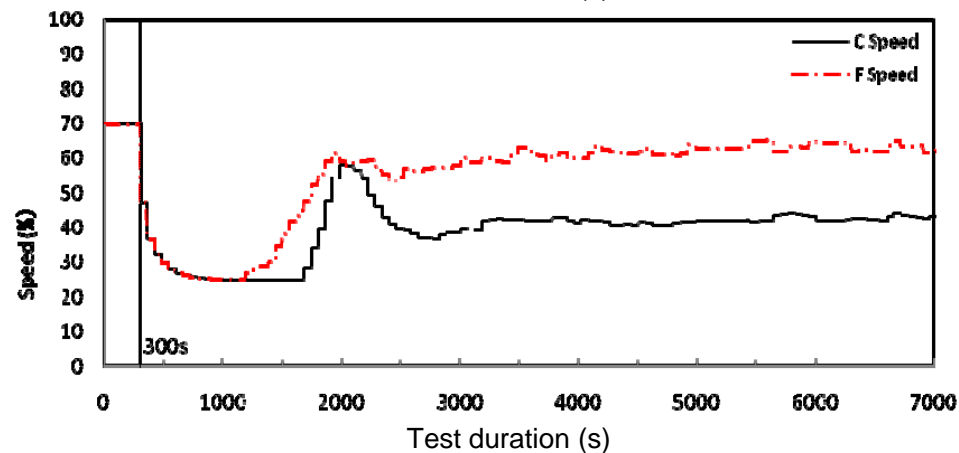


## ● The command following test



Temperature set points were altered at 300 s after achieving steady state

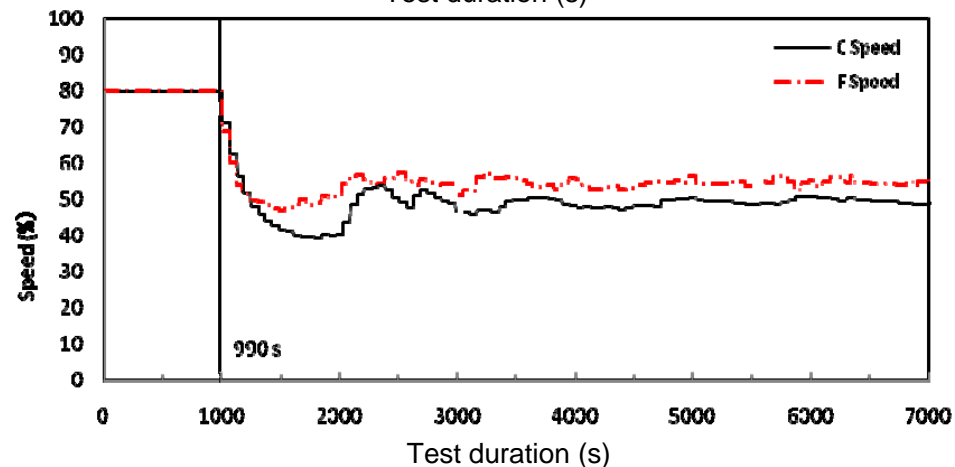
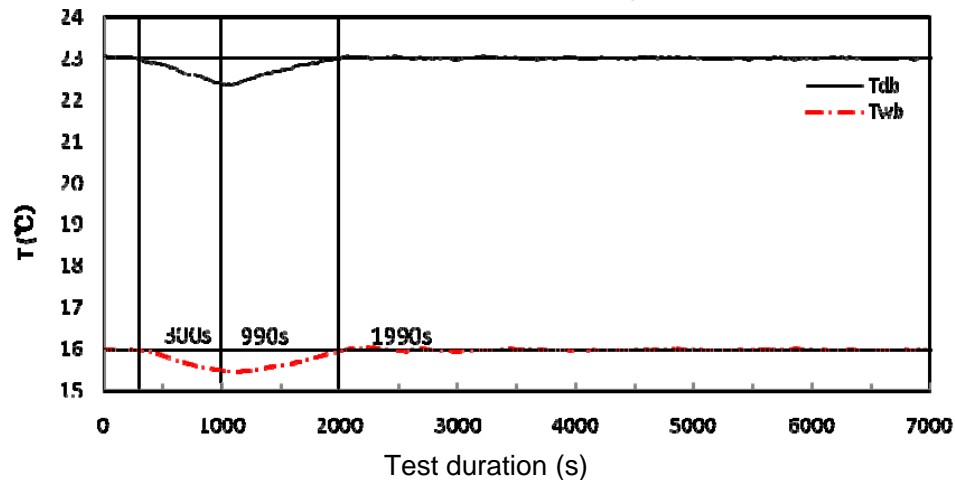
1500 s to settle at the altered set points and last stably for the rest time of test



Controller reacted immediately to the change of temperature set points regulating C and F simultaneously and finally maintained stably at new combination



## ● The disturbance rejection test



Indoor cooling loads were altered at 300 s after achieving steady state

Controller activated when: both temperature changed by 0.5 °C

1000 s for bringing both temperatures back to their fixed set points and maintained stably for the rest time of test

Controller reacted immediately when activated and regulated C and F simultaneously and finally maintained steadily at new combination

New combination of C & F speeds suggested the decreased indoor cooling loads



## Conclusions



- A PD law based fuzzy logic control algorithm for a VS DX A/C system has been developed;
- The controller was experimentally validated by carrying out command following & disturbance rejection tests;
- Simultaneous control of indoor air temperature and humidity using VS DX A/C system is realized with an adequate control accuracy and sensitivity but simpler development;
- However, general methods for determine WEIGHTS should be studied in future.



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# Q & A

The END, Thank you!