

2016

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Queiroz, Marcus Vinicius Almeida; Panato, Victor Hugo; Antunes, Arthur Heleno Pontes; Parise, Jose Alberto Reis; and Bandarra Filho, Enio Pedone, "Experimental comparison of a cascade refrigeration system operating with R744/R134a and R744/R404a." (2016). *International Refrigeration and Air Conditioning Conference*. Paper 1785.
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Experimental comparison of a cascade refrigeration system operating with R744/R134a and R744/R404A

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ABSTRACT

This study evaluates the performance of a cascade system in subcritical operation using the pair R744 / R134, as an option to conventional systems in supermarkets, which usually uses R404A, or R22. The experimental apparatus consists of a variable speed reciprocating compressor for R744 and an electronic expansion valve that promotes direct evaporation of the CO₂ inside a cold room (2.3m x 2.6m x 2.5m) to maintain the internal air temperature stable. The high-temperature cycle consists of a reciprocating compressor for R134a, a thermostatic expansion valve, and an air-cooled condenser. A plate heat exchanger, which is at the same time, the condenser for the R744 and evaporator to R134a completes the setup. Two parameters were manipulated: The superheating degree of the R744, 5-20 K, and the R744 compressor operation frequency, 40-65 Hz. In order to contribute to the improvement of the cooling processes, mainly about the sustainability and energy efficiency, a drop-in has been made at the high-temperature cycle, whose R134a load has been replaced by R404A. The alternative cooling system (R744 / R404A) was subjected to two stages of tests which allowed the energy comparison between the two refrigerant sets. Through the obtained results, it was estimated maximum COP equivalent to 1.36 and a minimum value of 1.06 for the R744 / R134a pair, demonstrating the applicability of this cascade system of variable thermal load conditions, the values of capacity cooling, settled between 4.09 and 5.13 kW. The minimum value of the air temperature within the cold room was -28 °C and -5 °C the maximum. Finally, it was found that the results obtained for R744 / R404A pair attended the air temperature condition inside the cold room with similar COP values. However, R744 / R404A pair operated at lower refrigeration capacities, these settled between 2.16 and 3.00 kW. This reduction is due to the large difference between the volumetric cooling capacity values of these HFCs in question, resulting in adaptation problems of the R404A to the compressor and the expansion mechanism.

1. INTRODUCTION

The global climate situation takes all efforts in the refrigeration area to develop systems that are able to combine a worth energy efficiency with reduced environmental impact. This impact is directly related to the use of HCFC and HFC refrigerants.

The destruction of the ozone layer is associated with the use of refrigerants HCFCs (in which is included the R22) and HFCs whose effect is measured / quantified by the ODP index (Ozone Depletion Potential). Although the HFCs gases with null potential for destruction of the ozone layer, ODP = 0, is necessary to note that these same gases (HFCs) imply a high atmospheric warming potential, given by the amount of CO₂ resultant from these productions. In some cases the global warming potential (GWP) of those gases are higher than the HCFCs. Natural fluids such as hydrocarbons and carbon dioxide are seen as possible solutions to those issues, with OPD = 0 and GWP the lowest possible.

The quantitative appraisal related to the greenhouse impact is given by GWP100a defined by the mass of CO₂ that would produce the same impact for 100 years on global warming as the release to the atmosphere of a single unit (kg) of the given component.

The refrigerants R22 and R404A are widely used in the commercial sector, specifically in supermarkets. The R22 is an HCFC that has ODP = 0.055 and GWP100a = 1810 while the R404A is an HFC with ODP zero and GWP100a of 3922. Therefore, those refrigerants have a major impact in global warming. The R744 has a great potential for applications in supermarkets, despite being a fluid with high working pressures. In environmental terms, the R744 has a null ODP and unitary GWP100a, being therefore a good alternative to solve environmental issues. R134a as well as R404A is an HFC, has a zero ODP and according to IPCC (2007), the equivalent value of GWP for 100 years to this refrigerant is 1430, calculated according to AR4 (Assessment Report 4) from IPCC (Intergovernmental Panel on Climate Change).

There are many works that reference the use R744 as the working fluid. In 1994 the Norwegian teacher Gustav Lorentzen first proposed that the refrigerant R744 could be used as a working fluid for vapor compression cycles. From that time many other authors have studied extensively such applications.

Montagner and Melo (2011) analyzed experimentally thermodynamic cycles of R744 for commercial refrigeration. It had been analyzed the effect of refrigerant charge and the type of expansion device (thermostatic valve or capillary tube). The results showed that the system performance depends on both parameters. The thermostatic valve promotes a better control of the conditions of evaporation, i.e. the evaporation temperature and the degree of superheating has reached optimum values.

Finckh et al. (2011) calculated numerically, and compares with the experimental data available in the literature, the influence of various parameters on the efficiency of the system using R744. These systems, generally transcritical, were compared with HFC refrigeration systems. In the model developed by the authors it is possible to modify parameters, such as superheating or the discharge pressure and then to optimize the system for maximum efficiency. The results showed that the transcritical CO₂ systems operated at ambient temperatures of 26 °C, obtained maximum COP 7. It was considered efficient and competitive as compared with HFC systems.

Silva et al. (2012) shows an experimental work related to energy efficiency in two different systems used in supermarket applications. The systems consist of a subcritical cascade R744 / R404A and R744 / R22. The results obtained by the authors show the reduction in energy consumption, 13-24%, when comparing the cascade system to direct expansion systems. Another important fact emphasized by the authors, is related to the load of the system. The cascade system uses 32 kg of CO₂ and 15 kg of R404A, whereas the direct expansion system of R404A uses 125 kg and the R22 uses 115 kg. The refrigerant amount influences the operation final cost, since it must take into account the fluid replacement in the system.

The goal of this experimental study is to evaluate the cascade system performance in subcritical operation using the pair R744 / R134a, which was originally installed at the apparatus, and then analyze the cascade cycle behavior replacing the R134a refrigerant by R404A through a drop-in operation.

The choice of R404A refrigerant was due to its extensive use in the refrigeration sector in supermarkets, even with high values of GWP.

2. MATERIALS AND METHODS

The experimental apparatus consists of a variable speed reciprocating compressor for R744 and an electronic expansion valve that promotes direct evaporation of the CO₂ inside a cold room (2.3m x 2.6m x 2.5m) to maintain stable the internal air temperature. Thus, the R744 leaves the compressor in superheated vapor state, is condensed and then stored in a liquid tank. By letting the liquid tank toward the electronic expansion valve, it undergoes a flow Coriolis meter, and expands in the evaporator unit, subsequently superheated R744 through an intermediate heat exchanger, further increasing the superheating degree. Finally, the CO₂ is returned to the suction of the compressor. Figure 1 illustrates the experimental facility.

The high-temperature cycle consists of a reciprocating compressor for R134a / R404A, a thermostatic expansion valve, and an air-cooled condenser. A plate heat exchanger, which is at the same time, the condenser to R744 and evaporator to R134a / R404A, completes the bench.

Three basic parameters can be changed in the system, first the superheat degree of the R744, second the operation frequency of the R744 compressor and, finally, the thermal load simulation within the cold room, carried by a bank of electrical resistances, which dissipates 1.5 kW of power. Although the cold room walls being well insulated, the heat exchanges by conduction therein, floor and ceiling, in addition to heat radiation are inherent to R744 evaporation process, therefore, the cooling capacity of the cascade system is superior to 1.5 kW.

$$COP_{R744} = \dot{Q}_{R744} / \dot{W}_{R744} \quad (2)$$

The COP of the cascade system is expressed by the Equation (3).

$$COP_{Sistema} = \dot{Q}_{R744} / (\dot{W}_{R744} + \dot{W}_{R134a/R404A}) \quad (3)$$

Beyond the proposed study of the energy performance of the cascade refrigeration cycle in different operating conditions, it is also an goal of this study, a comparison between the operation of the original cascade cycle (R744 / R134a) and the alternative cascade cycle (R744 / R404A). For this purpose, it was made a drop-in at the high temperature cycle, replacing the R134a by the R404A retaining the same compressor and lubricating oil, as well as heat exchangers and original components.

Accordingly, the new refrigerant has to provide similar values of the cooling capacity and air temperature inside the cold room, to the values obtained by the original system, thus respecting what would be the design conditions of a cold room for storage in a commercial application. COP provided by the new pair of refrigerants, becomes a consequence, and their values are compared to the original fluids.

However, there is a large difference between the values of volumetric cooling capacity of the two studied fluids. This parameter allows the prediction of the dimensions of the displacement mechanisms required a compressor for a given refrigerant and a specific operating condition.

Figure 2 shows the variation of this capacity by the evaporation temperature for different refrigerants. This parameter is a measure of cooling capacity per unit volume of refrigerant passing through the compressor. It is a property of the refrigerant and the system operating point.

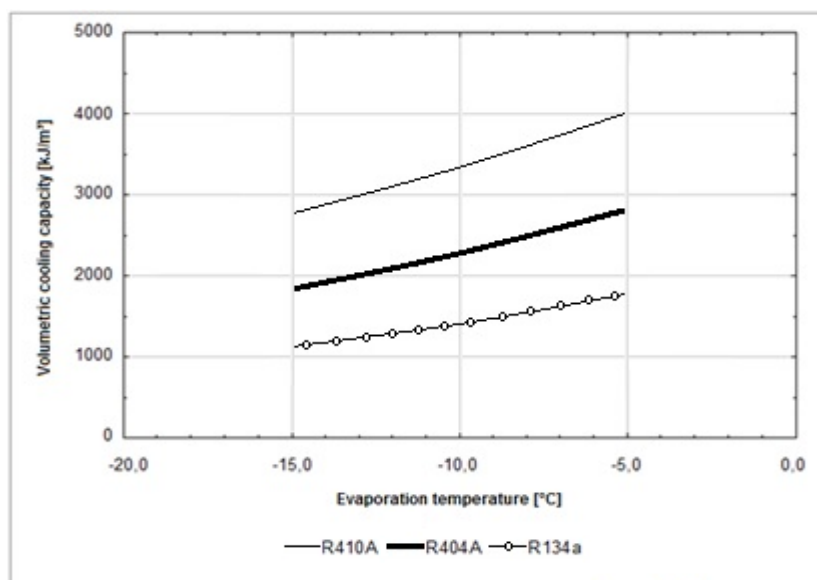


Figure 2: Volumetric cooling capacity relative to different classes of refrigerants. Estimated values at 40 °C condensation temperature, 0 °C of subcooling and 5 °C of superheating

As can be seen, the volumetric cooling capacity decreases with the reduction of evaporation temperature values. This is due mainly to the decrease in vapor density at low temperatures. Therefore, it is easily understood that the necessary compressor for operating a R404A cycle must always be smaller than the compressor required for R134a in fixed rotation.

The R404A moves away from the reference in terms of capacity, this fact makes it unsuitable to work at the same speed of the original system. Thus, in order to better adapt the cooling R404A system, a retrofit action was taken. Thus, a frequency inverter was installed and allowed the reduction of the speed of the high-temperature cycle compressor. This strategy allows the system to achieve higher cooling capacity and smaller air temperatures values inside the cold room, approximating the thermal load conditions to of the values of the original system.

3. RESULTS

The first leg results refers only to the analysis of the original cascade system operating with R134a in the high temperature cycle (HT) and R744 in the low temperature cycle (LT). A total of 24 tests were performed and two basic parameters have changed during these tests: the value of the superheating degree in the LT cycle and the value of the operating frequency of the R744 compressor.

Figure 3 shows the power values consumed by each compressor. It is evident that the R744 compressor has lower power consumption than the R134a compressor, this is due to the high vapor density of R744, even with the pistons displacing a high volume in the process. R744 such properties are also the cause of a compressor of reduced dimensions.

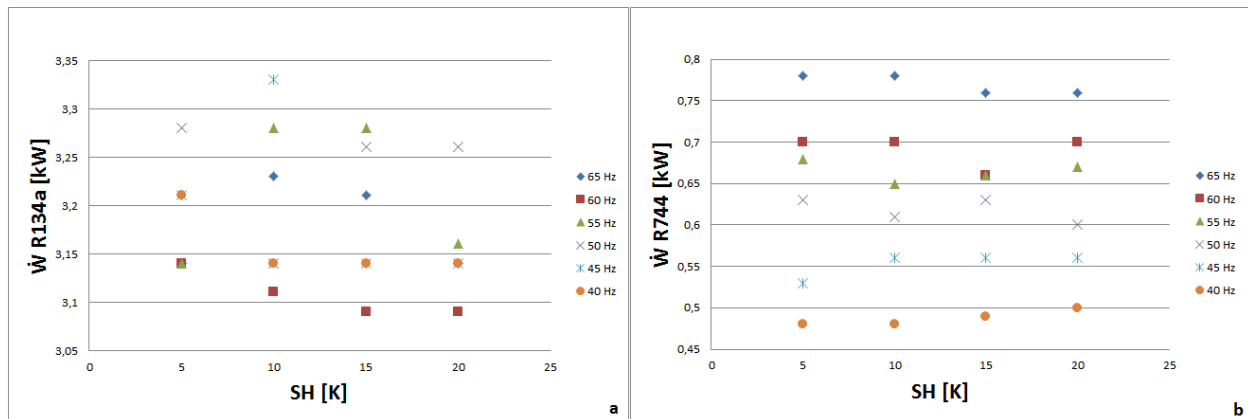


Figure 3: Power consumption of the LT (a) and HT (b) compressors, for different operating frequencies and different superheating degrees

The R744 compressor power consumption increases with the increase of the frequency and the consequent rotation (see Fig. 3a), as such proportionality cannot be observed by R134a compressor (Fig. 4b). Another factor that confirms the lowest consumption of the system R744 is the compression ratio of 2:1. In the case of R134a compressor this reason, in some comes to be 10: 1.

Figure 4 illustrates the COP values related to the cooling capacity values.

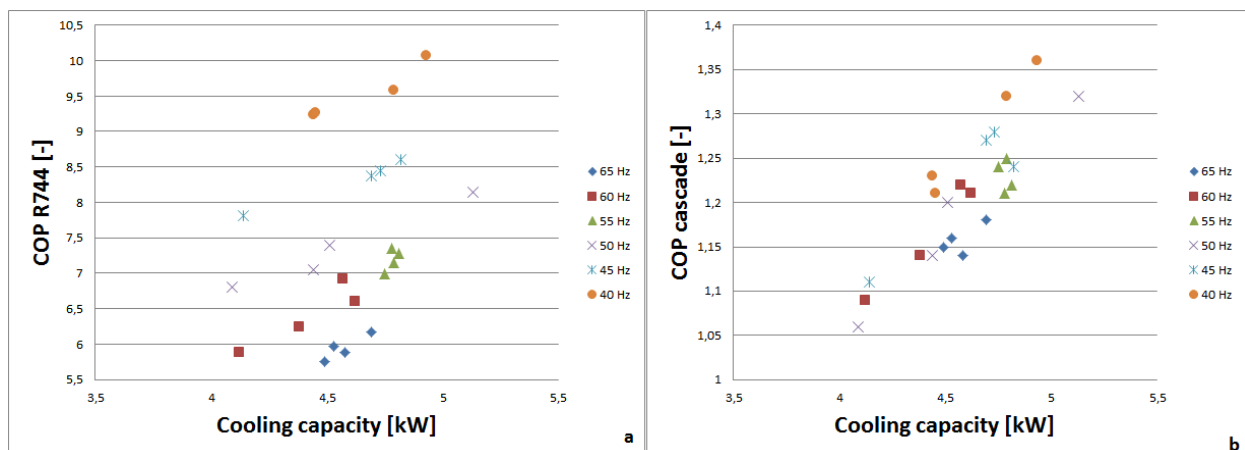


Figure 4: The behavior of the cooling capacity, by the COP values for the LT (a) and the cascade systems R744 / R134a (b)

Note that the COP behavior was physically consistent, the four tests at 40 Hz obtain the highest COPs from the R744 cycle, as the highest 10.07. The tests at 65 Hz equivalent to the worst performances.

The highest cooling capacity developed for the operating frequency of 50 Hz with 15K superheating, 5.13 kW. The values of COP cascade system (see Fig. 4b) follow the same trend COP proportionality for the isolated cycle R744 (Fig. 4a). The greatest value of this amounts to 1.36, operating at 40 Hz and 15 K superheat. The lowest COP of the system obtained was 1.06 for the condition of 65 Hz to 20 K superheating.

According to Figure 5, the operating point can be determinate from the temperature that is desired to have inside the cold room. It is noticed that with higher frequencies and smaller degrees of superheating, we obtain the lower air temperatures.

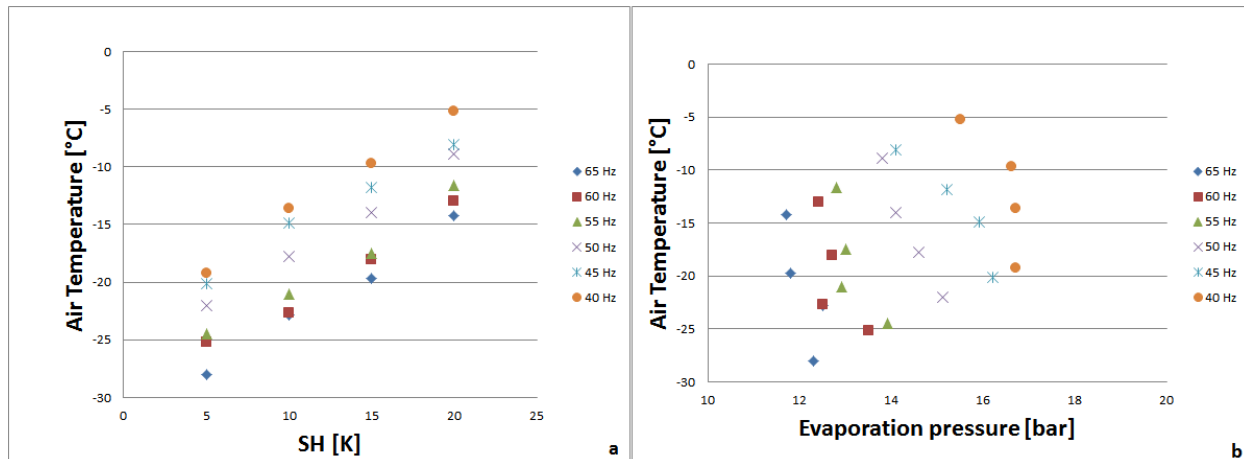


Figure 5: Air temperature behavior within cold room for different operating frequencies depending on the degree of superheat (a) and due to the R744 evaporation pressure (b)

Cooling capacity values were established between 4.09 and 5.13 kW, demonstrating the applicability of this cascade system for different thermal load conditions. The minimum value of the operating temperature of the air within the cold room was -28°C and -5°C maximum.

Sanz-Kocket et al. (2014) experimental data, were used as benchmarks for performance values found in this study. They evaluated experimentally, a cascade system R744 / R134a for commercial refrigeration. The experimental apparatus operated under different conditions and has less cooling capacity when compared to, however, the behavior of the main thermodynamic parameters were similar. The authors estimated values for the cascade system COP between 1.05 and 1.65. The cooling capacity values were 7.5 kW to 4.5 kW.

The second step results refers to the comparative analysis of the original cascade system and the alternative system, which operates with R404A in the high temperature cycle (HT) and R744 in the low temperature cycle (LT). A limited number of tests were carried out and three basic parameters were changed during the tests: the value of the superheating degree of the R744, the value of R744 compressor operating frequency and, finally, the compressor operation frequency at the R404A cycle.

Table 1 provides a comparison between the systems studied, this refers to three experimental tests, which the R744 compressor operated at nominal frequency of 60 Hz.

Table 1: Comparison between the pair of refrigerants, results in condition in which R744 compressor operated at nominal frequency of 60 Hz

Systems	T_{EVLT} [°C]	T_{CDLT} [°C]	T_{DCLT} [°C]	T_{LLT} [°C]	\dot{m}_{LT} [kg/s]	\dot{W}_{LT} [kW]	\dot{W}_{HT} [kW]	F_{HT} [Hz]	T_{AIR} [°C]	\dot{Q}_{SYS} [kW]	COP_{SYS} [-]
R744/R134a	-31,70	-8,16	96,1	-8,30	0,0167	0,70	3,14	60	-25,2	4,38	1,14
R744/R404A	-34,75	-25,74	73,7	-25,79	0,0083	0,36	2,30	60	-26,0	2,49	0,94
R744/R404A	-33,37	-19,29	82,1	-19,34	0,0098	0,47	1,80	35	-26,0	2,80	1,24

The first test refers to the pair of refrigerants (R744 / R134a), note that the HT compressor cycle ran at 60 Hz and the temperature of the air in the cold room stabilized at $-25.2\text{ }^{\circ}\text{C}$. The second test, which was conducted after the operation of the drop-in air met the condition of temperature inside the freezer, $-26\text{ }^{\circ}\text{C}$, however, the pair R744 / R404A has a cooling capacity value 43% less than the original system.

The frequency inverter installed in the HT compressor cycle has allowed the realization of the third test of Tab. 1 in which R404A system operated below the nominal speed (60 Hz) at a frequency of 35 Hz, making it possible to increase the value of the cooling capacity, 2.80 kW. In this condition, the alternative system COP exceeded 8% of the R744 / R134a COP to the same operating cold room temperature, $-26\text{ }^{\circ}\text{C}$.

The reduced cooling capacity is a result of the large difference between the values of volumetric cooling capacity of the two HFCs. It can be also seen in Table 1, the mass flow rate of R744 is greatly reduced for the system R404A. In addition, the value of the liquid temperature (T_L) for the second test in the table ($-25.79\text{ }^{\circ}\text{C}$) is very low compared to the value of the original system ($-8.30\text{ }^{\circ}\text{C}$). By reducing the frequency, the liquid temperature rises at $5.5\text{ }^{\circ}\text{C}$ and the flow rate increases by 10%, however, is not sufficient to promote evaporation condition compatible to the original system. This comparison can be seen in Fig. 6b, where it is seen that the cooling capacity for all tested conditions fail to achieve similar operation points due to temperature of the air in the cold room.

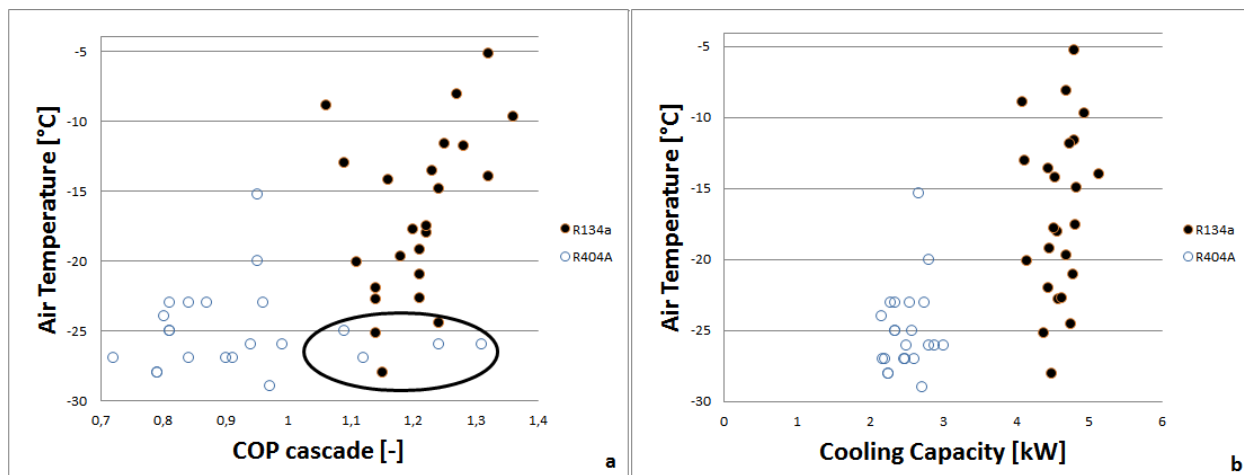


Figure 6: Effect of the COP (a) and cooling capacity (b) on the air temperature, on the overall

Can be observed in the demarcated region of Fig. 6a, which tests for that sector, established a compatibility between the air temperature in the chamber and the COP for both systems. The pair containing R404A can achieve the best COPs (1.31), when the frequency of the compressor operates in 35 and 40 Hz. However, even though the COP and operating temperature conditions are compatible, the cooling capacity of the alternative system R404A is reduced in all analyzed range, Fig. 6b.

Figure 7 provides a comparison for energy performance of the two pairs of refrigerants, it is remarkable the supremacy of the cascade system R744 / R134a. The tests were performed for 4 superheating degrees, showed at the figure besides each experimental point. The same operating frequency values for the compressors were maintained, 60 Hz for the high temperature cycle (R404A and R134a) and 65 Hz for the low temperature cycle (R744).

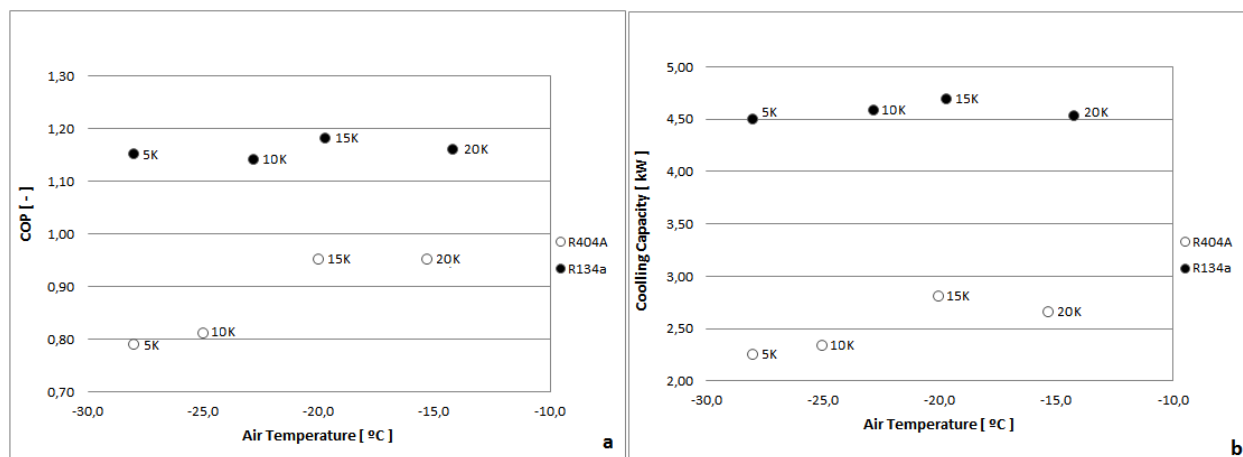


Figure 7: Effect of air temperature on the COP (a) and cooling capacity (b), for a superheating degree range from 5 to 20 K, and same operating frequency values for the compressors of HT and LT

4. CONCLUSION

In order to contribute to the improvement in cooling processes, as well as sustainability and energy efficiency, this experimental study evaluated the performance of a subcritical cascade system using the pair R744 / R134a, which represents the original experimental apparatus installed. On a second step, after replacing the R134a by the R404A, through a drop-in operation, an experimental comparison was performed between the behavior of the original system and the new pair of refrigerants.

Through the first results, it was observed that the cooling capacity values were established between 4.09 and 5.13 kW, demonstrating the applicability of this cascade system working at variable thermal load conditions. The minimum value of the operative temperature of the air inside the cold chamber was -28 °C and the maximum value of this parameter is equivalent to -5 °C.

The evaluation of the original system has shown that the highest COP values were obtained at the lower frequencies of operation of the LT compressor, and these values compared to direct expansion systems with HFCs or HCFCs are generally lower. The maximum COP was equal to 1.36 and the minimum value was 1.06 for the pair R744 / R134a.

Furthermore, the cooling capacity values were similar for the 24 tests, however, the desired temperature of the air inside the chamber must also be evaluated to then, determine the degree of superheat and the frequency used at the LT compressor.

At the sequence, an operation of drop-in was established in the HT cycle, where the charge of R134a was replaced by R404A. Moreover, the compressor of the HT cycle has stopped to operate at 60 Hz and started to operate with variable speed. Through the second step (21 tests) it was found that the results obtained for pair R744 / R404A attended the air temperature condition inside the cold chamber, with similar COP values of the original pair. However, R744 / R404A pair operated at much lower cooling capacities, these values has been established between 2.16 and 3.00 kW. This reduction is due to the large difference between the volumetric cooling capacity of both HFCs, which results in adaptation problems of R404A to the original HT cycle, compressor and expansion device.

Finally, the results confirmed that the cascade system R744 / R134a is superior to the alternative system (R744 / R404A). The direct drop-in operation is inefficient in this case because only four test conditions of 21 carried out were compatible to the original system, and even then the cooling capacity values were reduced.

For future works, the modification of the expansion device of the HT cycle (R404A) will be performed to permit the condensation of R744 at higher temperatures and thus, achieving higher cooling capacity values. In addition, other refrigerants will be evaluated at the HT cycle in order to find the best pair of refrigerants for the original application range.

NOMENCLATURE

GWP	Global warming potential	
ODP	Ozone depletion potential	
Δh	Specific enthalpy difference	kJ/kg
ΔT	Temperature difference	°C
F	Frequency	Hz
h	Specific enthalpy	kJ/kg
\dot{m}	Mass flow rate,	kg/s
p	Pressure	kPa
\dot{Q}	Cooling capacity	kW
SH	Superheating	K
T	Temperature	°C
\dot{W}	Energy consumption	kW

Subscript

CD	Condensation
DC	Discharge
EV	Evaporation
HT	High-Temperature
LT	Low-Temperature
L	Liquid
sys	System

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ACKNOWLEDGMENT

The authors acknowledge the financial support given to the reported investigation by CAPES, CNPq, FAPEMIG and also for the technical support provided by the application engineer Alessandro da Silva, Bitzer Scroll (USA) and Roberto Possebon Junior, Carel (Brazil).