

10-2007

An Improved Model for Microperforated Absorbers

J Stuart Bolton
Purdue University, bolton@purdue.edu

Jonathan H. Alexander
3M

Taewook Yoo

David F. Slama

Follow this and additional works at: <https://docs.lib.purdue.edu/herrick>

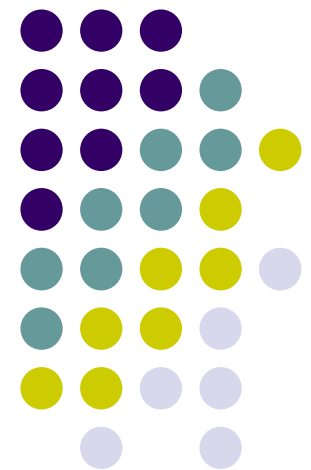
Bolton, J Stuart; Alexander, Jonathan H.; Yoo, Taewook; and Slama, David F., "An Improved Model for Microperforated Absorbers" (2007). *Publications of the Ray W. Herrick Laboratories*. Paper 79.
<https://docs.lib.purdue.edu/herrick/79>

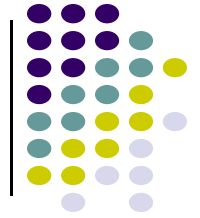
This document has been made available through Purdue e-Pubs, a service of the Purdue University Libraries.
Please contact epubs@purdue.edu for additional information.

An Improved Model for Microperforated Absorbers



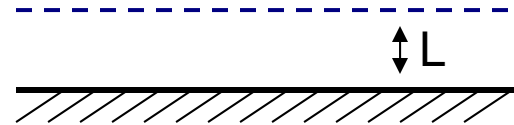
Taewook Yoo
J. Stuart Bolton
David F. Slama
Jonathan H. Alexander



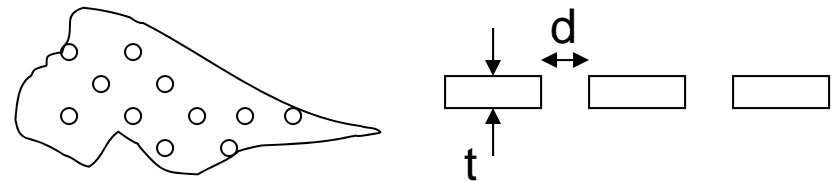


Introduction / Objective

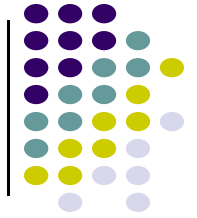
- Microperforated panels work well as sound absorbing materials when backed with finite-depth air space



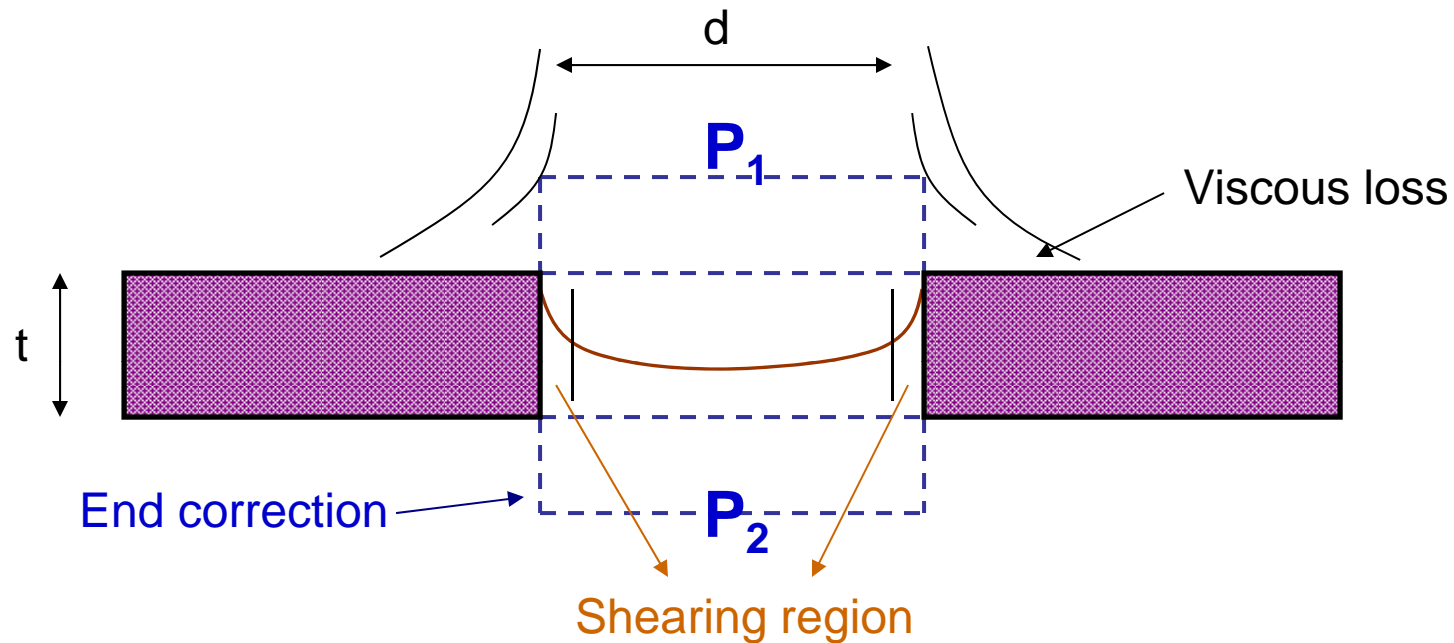
- Controlling parameters: hole diameter, hole depth, number of holes per unit area



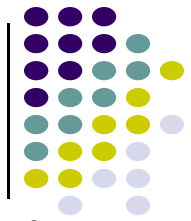
- D. Y. Maa models
 - Referred to by many other authors
 - Used as basis for design of microperforated absorbers
 - Various versions (1975, 1987, 1998)
- Objective
 - Perform measurements with precisely manufactured brass samples
 - Find the version of the Maa model that most closely represents the measurements
 - Broaden the range of applicability of the Maa model by modifying the resistive end correction



Maa Model



- Accounts for viscous losses in hole and on flat surface close to hole
- End correction calculated in basis of uniform velocity profile in hole



Maa model (1975)

$$z = \frac{Z_1}{\sigma \rho_0 c} = r + j\omega m$$

Z_1 : specific acoustic impedance of single hole
 σ : porosity
 r : resistance
 m : effective mass per unit area

- Perforation constant $x = 210d\sqrt{f}$ Relate to boundary layer

- Resistance
$$r = \frac{32(\mu + \nu)}{\sigma c} \frac{t}{d^2} \left(\sqrt{1 + \frac{x^2}{32}} + \frac{\sqrt{2}}{8} x \frac{d}{t} \right)$$

d : hole diameter
 f : frequency
 t : hole depth
 c : speed of sound
 μ : kinematic viscosity
 ν : thermal conductivity
 L : backing depth

Contribution from hole (green box) End corrections (effect from flow over outer surface and convergence into and out of holes) (orange box)

- Reactance
$$m = \frac{t}{\sigma c} \left(1 + \frac{1}{\sqrt{9 + \frac{x^2}{2}}} + 0.85 \frac{d}{t} \right)$$

- Absorption Coefficient
$$\alpha_n = \frac{4r}{(1+r)^2 + \left(\omega m - \cot\left(\frac{\omega L}{c}\right) \right)^2}$$

Panel is assumed **RIGID** in Maa models: no flexural motion is considered

Maa models

Substantial change
in Resistance

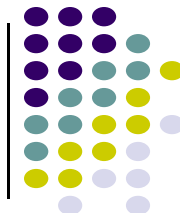


Not much change
in Reactance



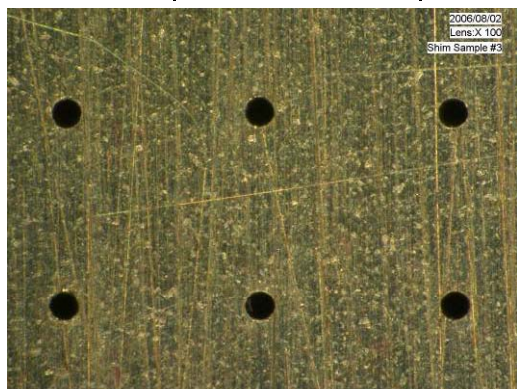
Models	Perforation constant	Resistance	Reactance
1975 High thermal conductivity model- Scientia Sinica	$x = 210d\sqrt{f}$	$r = \frac{32(\mu+\nu)}{\sigma} \frac{t}{d^2} \left(\sqrt{1 + \frac{x^2}{32}} + \frac{\sqrt{2}}{8} x \frac{d}{t} \right)$	$m = \frac{t}{\sigma} \left(1 + \frac{1}{\sqrt{9 + \frac{x^2}{2}}} + 0.85 \frac{d}{t} \right)$
1975 Low thermal conductivity model- Scientia Sinica	$x = 316d\sqrt{f}$	$r = \frac{32\mu}{\sigma} \frac{t}{d^2} \left(\sqrt{1 + \frac{x^2}{32}} + \frac{\sqrt{2}}{8} x \frac{d}{t} \right)$	$m = \frac{t}{\sigma} \left(1 + \frac{1}{\sqrt{9 + \frac{x^2}{2}}} + 0.85 \frac{d}{t} \right)$
1987 Noise Control Engineering Journal	$x = d \sqrt{\frac{\omega\rho_0}{4\eta}}$	$r = \frac{32\eta}{\sigma\rho_0 c} \frac{t}{d^2} \left(\sqrt{1 + \frac{x^2}{32}} + \sqrt{\frac{2xd}{8t}} \right)$	$m = \frac{t}{\sigma} \left(1 + \frac{1}{\sqrt{9 + \frac{x^2}{2}}} + 0.85 \frac{d}{t} \right)$
1998 Journal of Acoustical Society of America	$x = d \sqrt{\frac{\omega\rho_0}{4\eta}}$	$r = \frac{32\eta}{\sigma\rho_0 c} \frac{t}{d^2} \left(\sqrt{1 + \frac{x^2}{32}} + \frac{\sqrt{2}}{32} x \frac{d}{t} \right)$	$m = \frac{t}{\sigma} \left(1 + \frac{1}{\sqrt{9 + \frac{x^2}{2}}} + 0.85 \frac{d}{t} \right)$

d : hole diameter, f : frequency, t : hole depth, c : speed of sound, μ : kinematic viscosity, ν : thermal conductivity, η : viscosity coefficient ($=\mu\rho_0$)

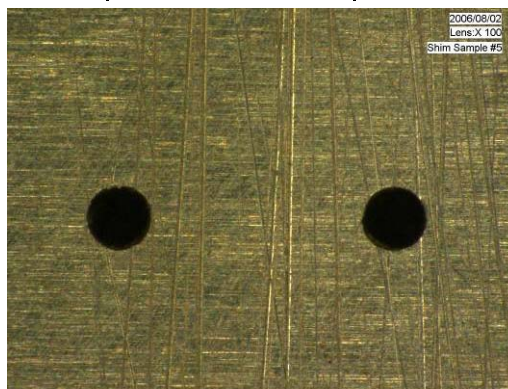


Brass samples

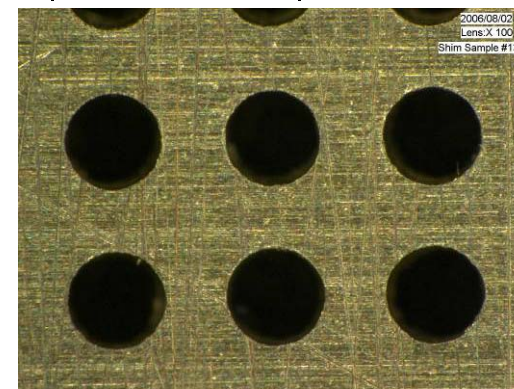
Sample number	Hole diameter d [mm]	Hole depth t [mm]	Number of holes per m^2	t/d	Mass/area [kg/m ²]	Porosity [%]
1	0.197	0.813	3.06×10^5	4.1	6.5	0.9
2	0.185	1.27	6.20×10^5	6.8	9.8	1.7
3	0.41	0.406	3.03×10^5	1	3.2	4
4	0.413	0.813	6.07×10^5	2	5.9	8.1
5	0.41	0.432	9.47×10^5	1.1	2.9	12.5
6	0.419	1.27	9.11×10^5	3	8.9	12.6
7	0.622	0.813	3.11×10^5	1.3	5.9	9.5
8	0.645	0.432	6.11×10^5	0.7	2.6	20
9	0.625	1.27	6.26×10^5	2	8.1	19.2
10	0.625	0.813	9.22×10^5	1.3	4.7	28.3



Sample 2 (x100)

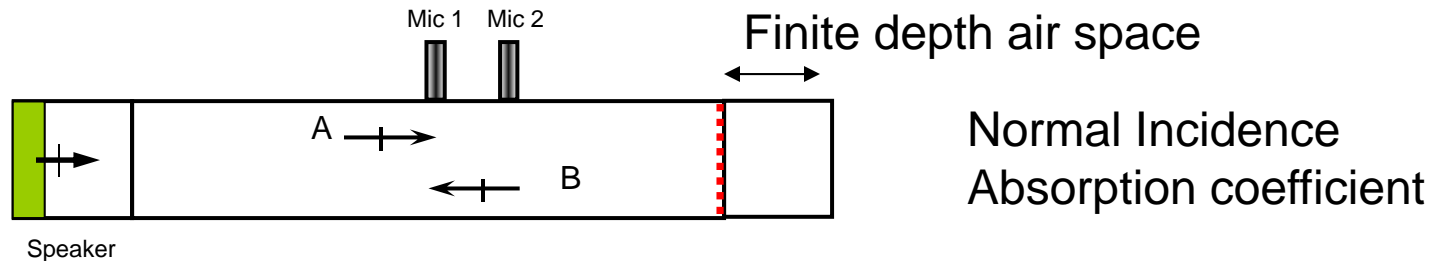
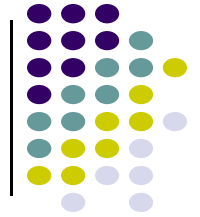


Sample 3 (x100)



Sample 10 (x100)

Measurement of normal incidence absorption coefficient



1. Sound pressures

$$P_1 = (Ae^{-jkx_1} + Be^{jkx_1})e^{j\omega t}$$

$$P_2 = (Ae^{-jkx_2} + Be^{jkx_2})e^{j\omega t}$$

3. Solve for R

$$R = \frac{-H_{21}e^{-jkx_1} + e^{-jkx_2}}{H_{21}e^{jkx_1} - Re^{jkx_2}}$$

2. Measuring transfer function

$$H_{21} = \frac{Ae^{-jkx_2} + Be^{jkx_2}}{Ae^{-jkx_1} + Be^{jkx_1}}$$

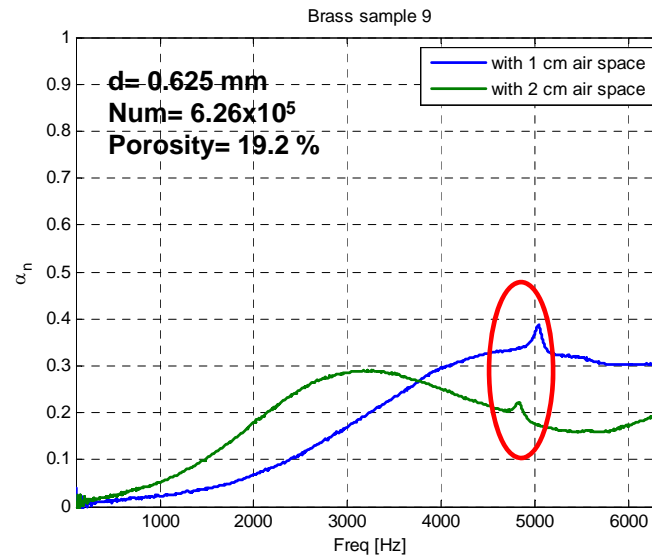
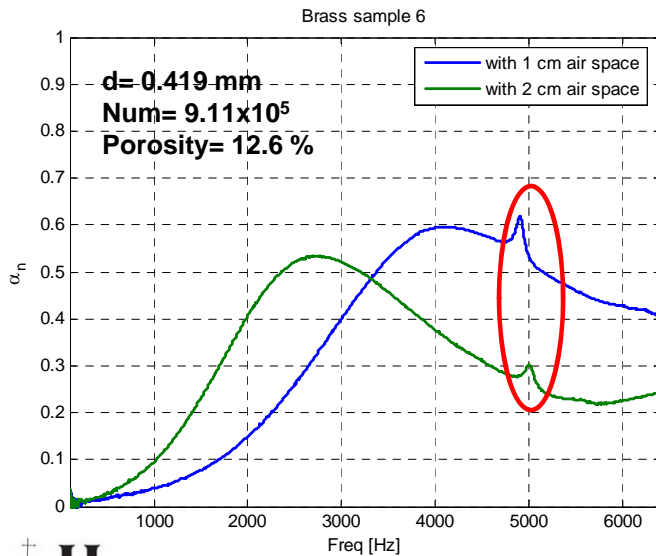
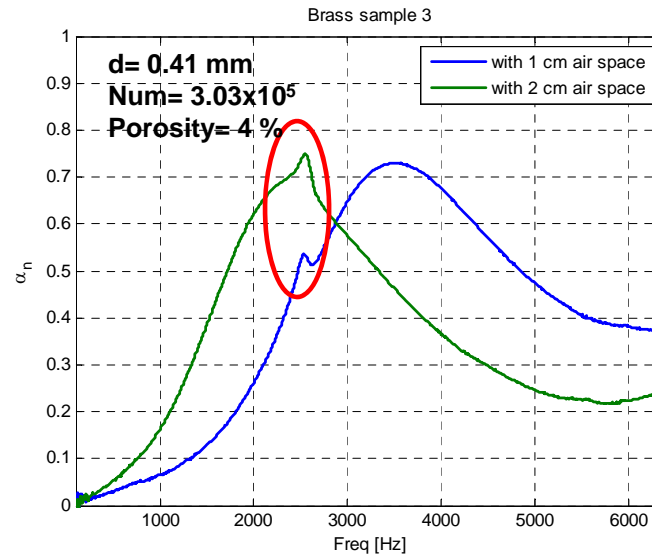
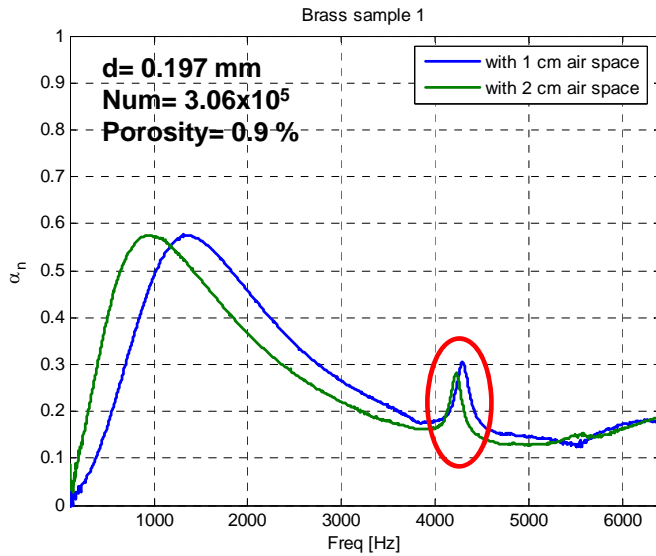
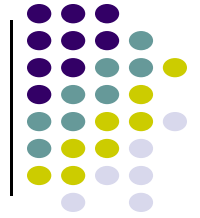
$$H_{21} = \frac{e^{-jkx_2} + Re^{jkx_2}}{e^{-jkx_1} + Re^{jkx_1}}$$

4. Absorption coefficient

$$\alpha = 1 - |R|^2$$

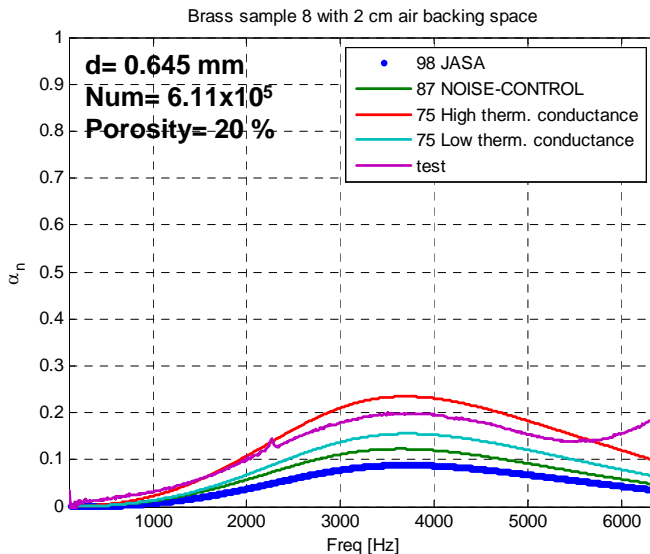
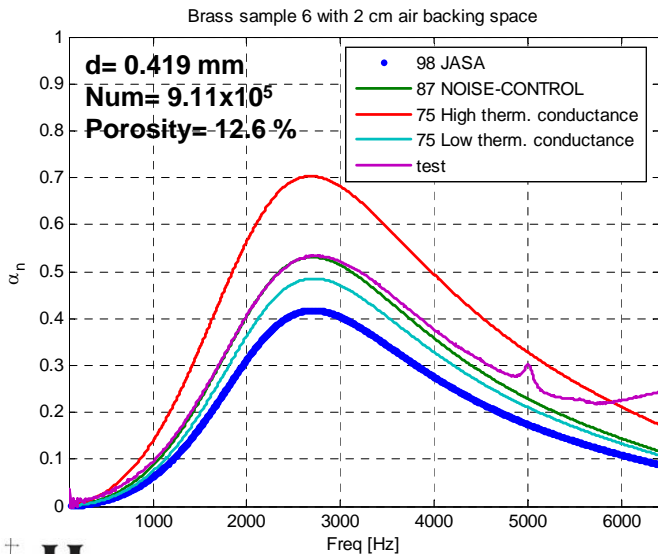
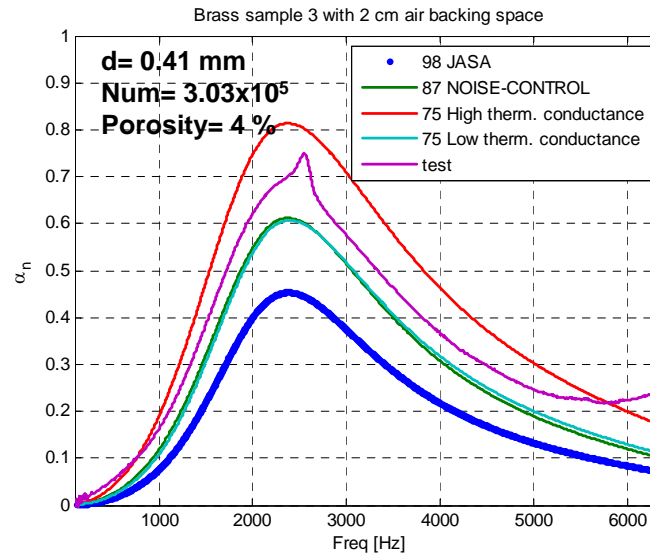
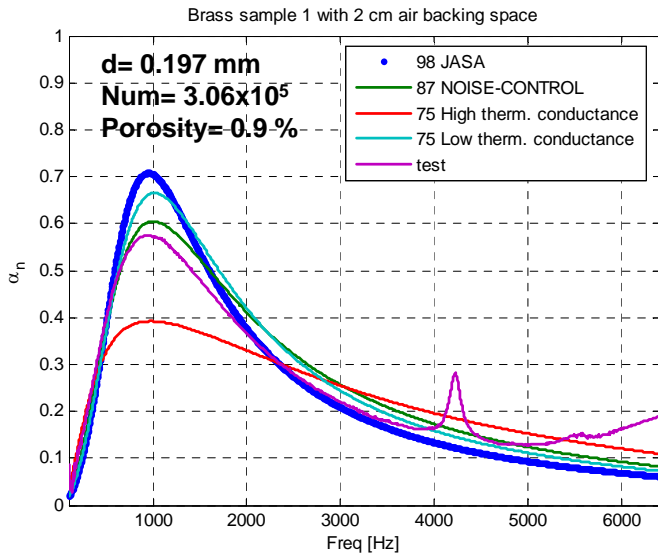
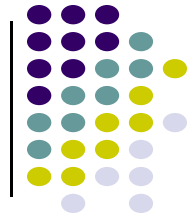
- Two backing depths: 1 cm and 2 cm
- B&K small impedance tube (d=2.9 cm) was used with Pulse 10
- Rubber O-ring was used to avoid the leakage from the contact between sample and tube surface

Absorption coefficients measured results



- As the backing space increases, the peak shifts to lower frequency
- Flexural resonance features due to finite size of the sample are shown
- High absorption at low frequency can be achieved with smaller hole size
- When the porosity is large, the absorption performance is significantly reduced

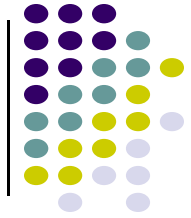
Measurements and predictions from various Maa models



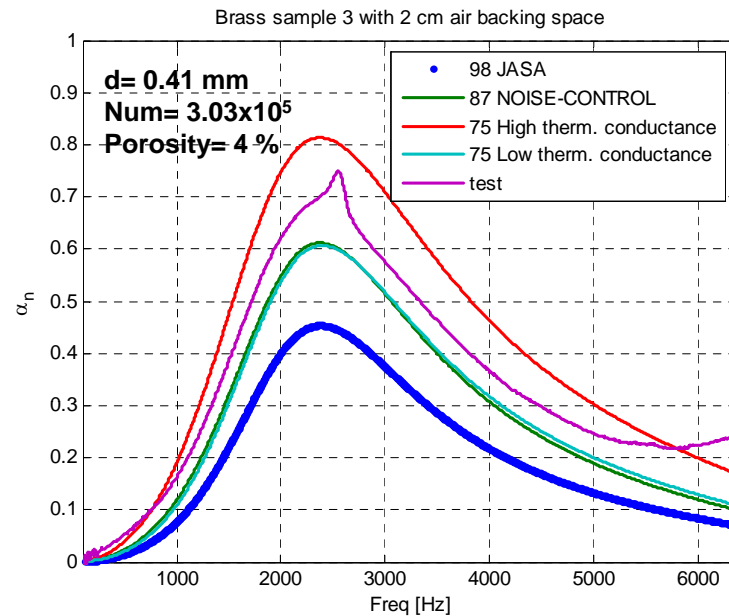
- There are significant differences on absorption predictions depending on model used

- Which one gives the most accurate prediction?

- Error calculation

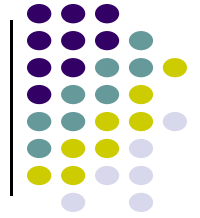


Model performance



- Observations

- Absorption peak locations accurately predicted in all cases- reactive part of model impedance is assumed to be accurate
- Absorption peaks heights are not predicted accurately consistently by any of the models- resistive end correction is assumed to be inaccurate

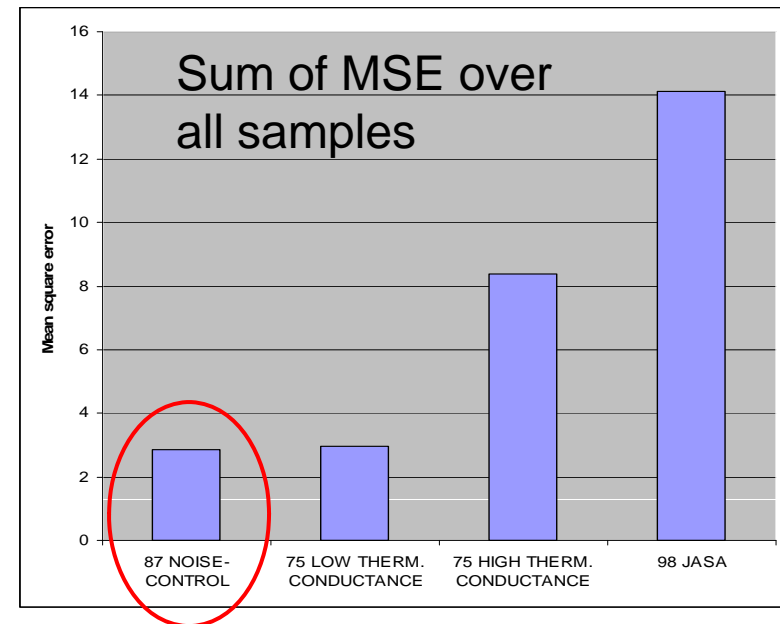
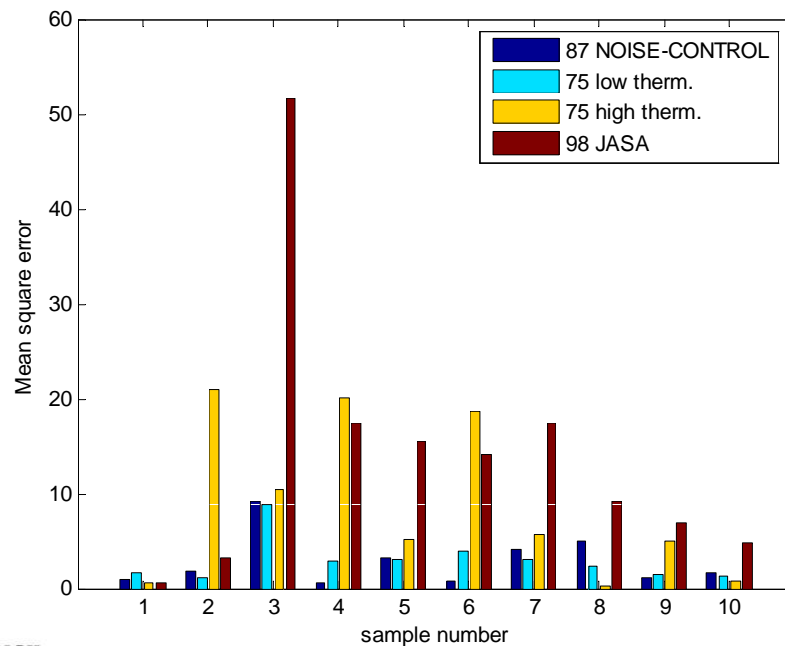


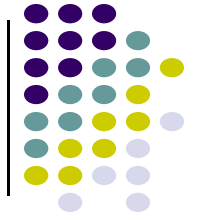
Error calculation

- Mean square error was calculated by

$$MSE = \frac{1}{n} \sum_{freq=f_1}^{f_n} (\alpha_{prediction} - \alpha_{test})^2$$

- Data at flexural resonance peaks were not considered
- Data at the frequencies from 100 Hz to 5000 Hz were used
 - Leakage is suspected above 5500 Hz





Optimization of Maa model (1)

- Using the most accurate, 87 N.C.E.J. model
- Modify the resistive end correction by introducing constant factor β

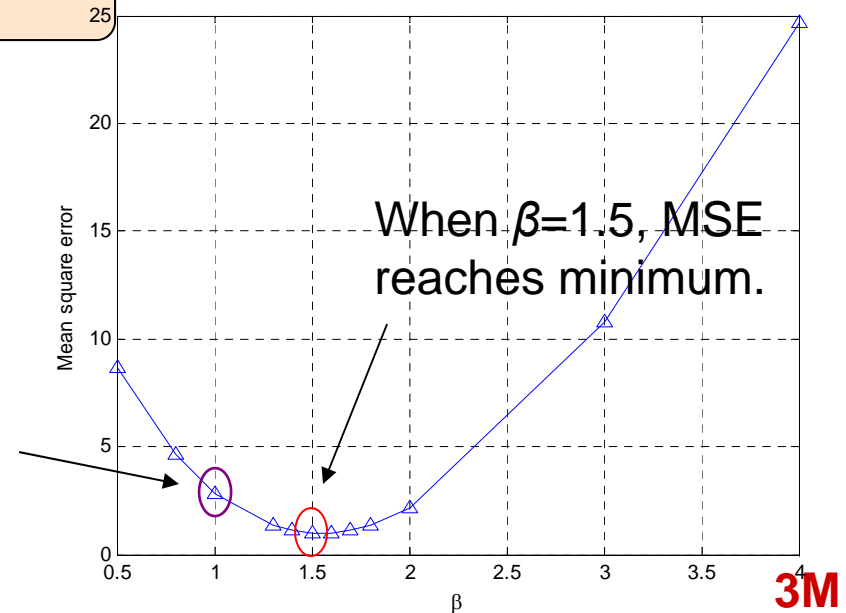
$$r = \frac{32\eta}{\sigma\rho_0 c} \frac{t}{d^2} \left(\sqrt{1 + \frac{x^2}{32}} + \beta \sqrt{\frac{2xd}{8t}} \right)$$

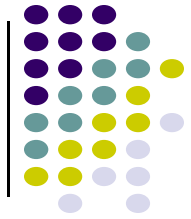
Effect from hole

End correction (effect from surface)

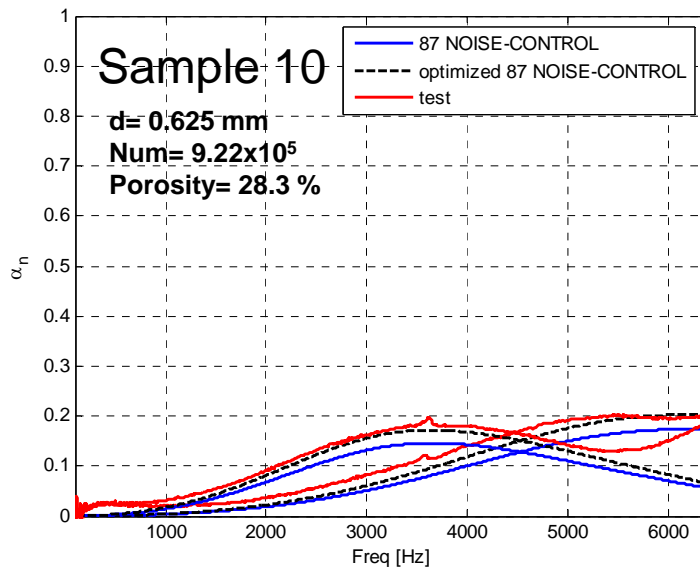
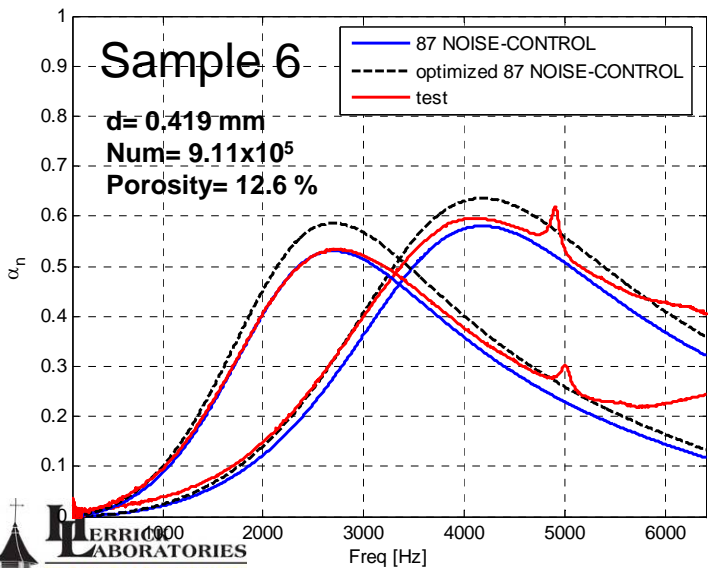
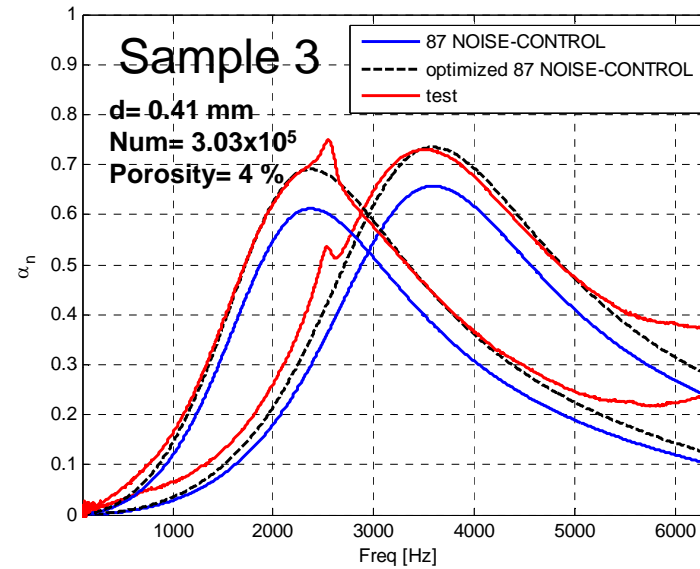
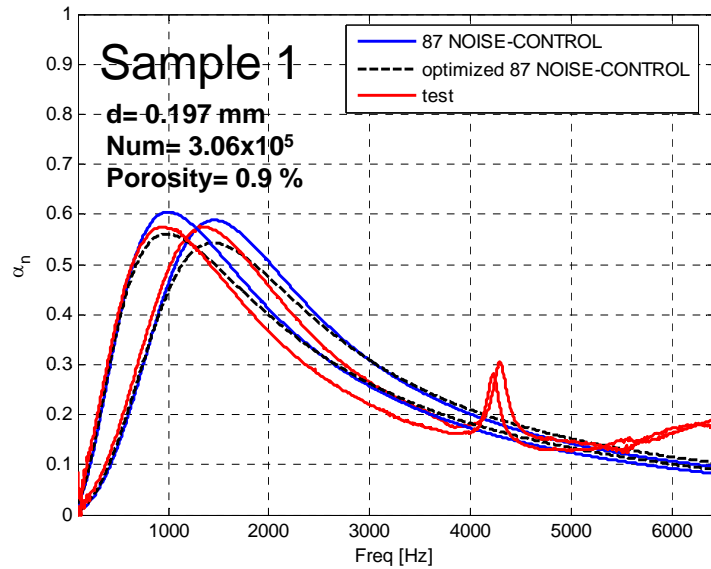
- Sum of mean square error was calculated with varying β

87 NOISE
CONTRL
model





Improved model results

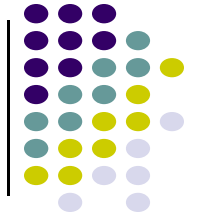


- Modified model can predict the absorption performance more accurately for large porosity cases

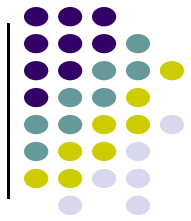
- It covers wide range of porosity

- Discrepancy remains for in some cases

Optimization of Maa model (2)

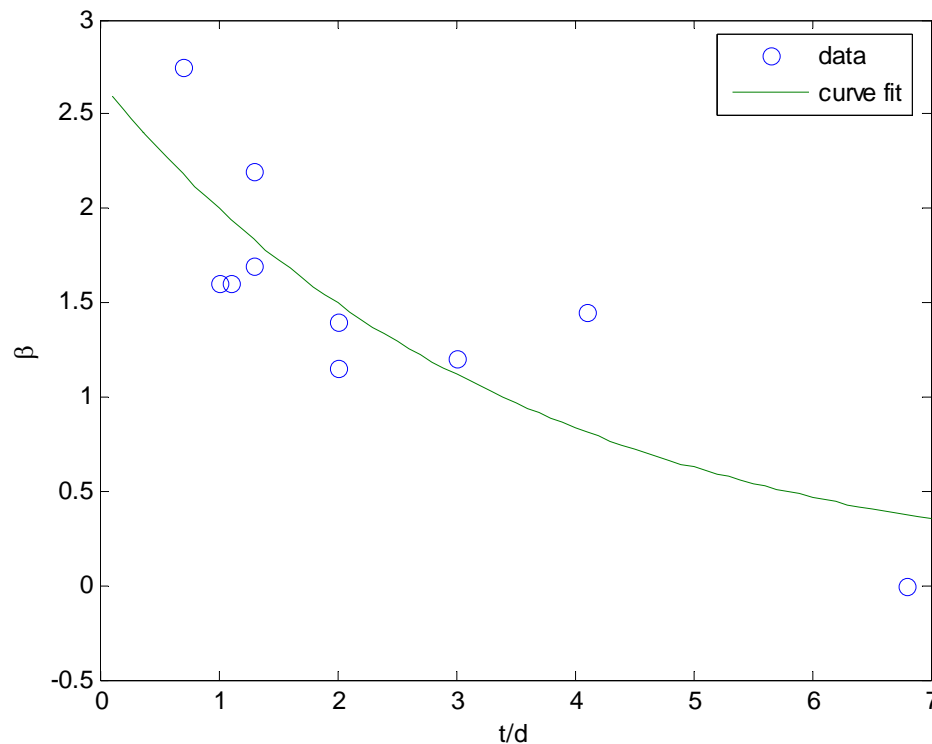


- Assume that resistive end correction is a function of hole aspect ratio (t/d)
- Determine optimal β for each sample
- Plot as function of hole aspect ratio (t/d)



Optimization of Maa model (2)

- Resistive end correction factor versus hole aspect ratio (t/d)

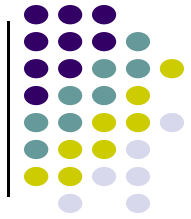


Using least square method with exponential curve fit

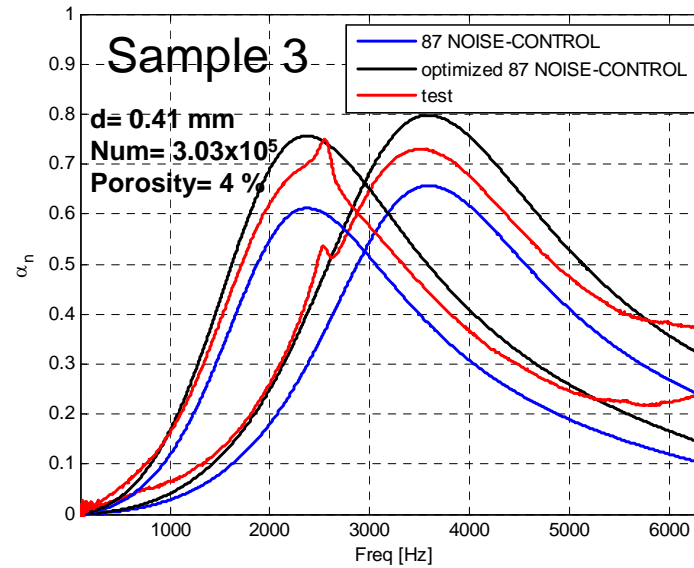
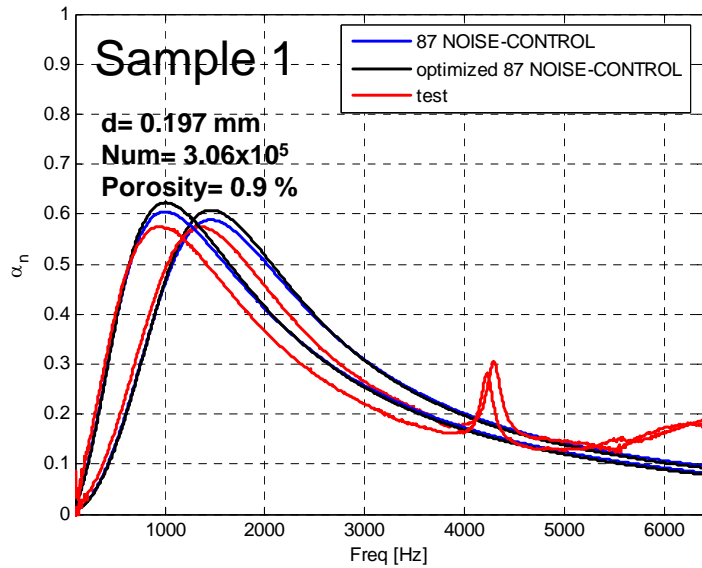
$$\beta = 2.672 \times e^{-0.2897(t/d)}$$

Substitute β to resistive calculation

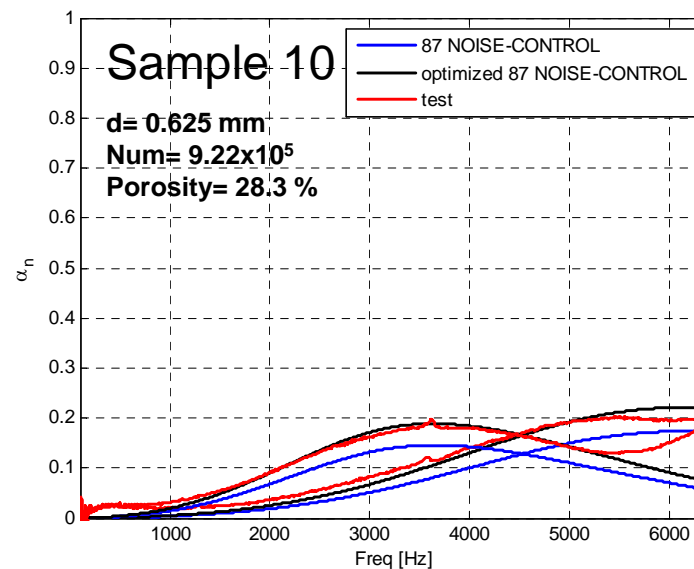
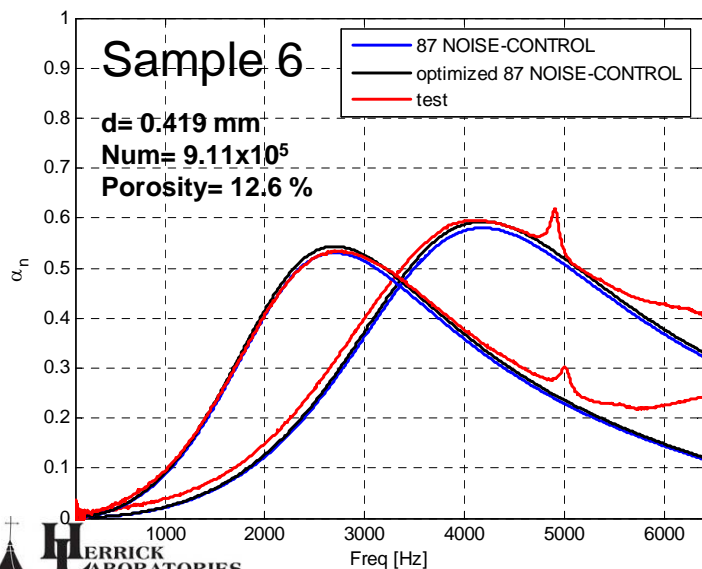
$$r = \frac{32\eta}{\sigma\rho_0 c} \frac{t}{d^2} \left(\sqrt{1 + \frac{x^2}{32}} + \beta \sqrt{\frac{2xd}{8t}} \right)$$

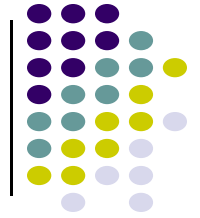


Optimization of Maa model (2)



$$\beta = 2.672 \times e^{-0.2897(t/d)}$$





Conclusions

- Microperforated panel works well as an absorber
 - Depending on hole geometries, the absorption capability changes
 - Deeper air space gives better absorption at low frequencies
 - Flexural resonances affect absorption performance in narrow frequency ranges
- Among Maa models, 1987 NCEJ model predicts the absorption performance most accurately
- By modifying the resistive end correction in the 1987 model, the accuracy of the predictions was improved over a very wide range of surface porosity
- Measurements will be repeated with non-metallic material to establish effect of thermal conduction